NAVAL POSTGRADUATE SCHOOL MONTEREY, CALIFORNIA



THESIS

CONVECTIVE HEAT TRANSFER FROM A CYLINDER IN A STRONG ACOUSTIC FIELD

by

Donald R. Harder

December, 1995

Thesis Advisor:

Thesis Co-Advisor:

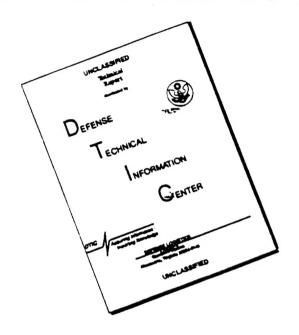
Ashok Gopinath Oscar Biblarz

Approved for public release; distribution is unlimited.

19960411 118

DTIC QUALITY RISPECTED I

DISCLAIMER NOTICE



THIS DOCUMENT IS BEST QUALITY AVAILABLE. THE COPY FURNISHED TO DTIC CONTAINED A SIGNIFICANT NUMBER OF PAGES WHICH DO NOT REPRODUCE LEGIBLY.

R	EP	O	RT	ΓD	O	CUN	MEN	TA	TI	ON	PA	GE
		~		_	$\mathbf{-}$	\sim				\mathbf{v}_{\perp}		VIII.

Form Approved OMB No. 0704-0188

Public reporting burden for this collection of information is estimated to average 1 hour per response, including the time for reviewing instruction, searching existing data sources, gathering and maintaining the data needed, and completing and reviewing the collection of information. Send comments regarding this burden estimate or any other aspect of this collection of information, including suggestions for reducing this burden, to Washington Headquarters Services, Directorate for Information Operations and Reports, 1215 Jefferson Davis Highway, Suite 1204, Arlington, VA 22202-4302, and to the Office of Management and Budget, Paperwork Reduction Project (0704-0188) Washington DC 20503.

1.	AGENCY USE ONLY (Leave blank)	2. REPORT DATE December 1995	3.			TYPE AND DATES COVERED Thesis		
	TITLE AND SUBTITLE NVECTIVE HEAT TRANSFER F OUSTIC FIELD	5.	FUNDING NUMBERS					
6. Har	AUTHOR(S) der, Donald R.							
7.	PERFORMING ORGANIZATION NAM Naval Postgraduate School Monterey CA 93943-5000		8.	PERFORMING ORGANIZATION REPORT NUMBER				
9.	SPONSORING/MONITORING AGENC		10.	SPONSORING/MONITORING AGENCY REPORT NUMBER				
11.	SUPPLEMENTARY NOTES The views expressed in this thesis are those of the author and do not reflect the official policy or position of the Department of Defense or the U.S. Government.							
12a.	DISTRIBUTION/AVAILABILITY STATEMENT Approved for public release; distribution is unlimited.					12b. DISTRIBUTION CODE		

13. ABSTRACT (maximum 200 words)

Experimental work was performed to study the convective heat transfer characteristics from a cylinder in a strong zero-mean oscillatory flow represented by an acoustic field. Two different flow regimes are discussed; that in which laminar, attached flow around the cylinder is present, and that in which instabilities, such as vortex shedding occur. The experiment utilizes a steady state measurement method. A transition from the laminar to the unstable regime was observed to occur at a streaming Reynolds number of approximately 240. Within the laminar regime, the transition from "intermediate" to "large" values of the streaming Reynolds number occurs at approximately 130. Heat transfer results for large values of the streaming Reynolds number in the laminar regime closely match the present theory (less than 13% error). Correlations were developed to relate the heat transfer rate to the streaming Reynolds number in the unstable regime. This work would find application in the design of heat exchangers for a thermoacoustic engine.

	SUBJECT TERMS rmoacoustics, Convective	15.	NUMBER OF PAGES 120				
•						16.	PRICE CODE
17.	SECURITY CLASSIFI- CATION OF REPORT Unclassified	18.	SECURITY CLASSIFI- CATION OF THIS PAGE Unclassified	19.	SECURITY CLASSIFICA- TION OF ABSTRACT Unclassified	20.	LIMITATION OF ABSTRACT UL

NSN 7540-01-280-5500

Approved for public release; distribution is unlimited.

CONVECTIVE HEAT TRANSFER FROM A CYLINDER IN A STRONG ACOUSTIC FIELD

Donald R. Harder
Lieutenant, United States Navy
B.S., University of Washington, 1985

Submitted in partial fulfillment of the requirements for the degrees of

MASTER OF SCIENCE IN MECHANICAL ENGINEERING MASTER OF SCIENCE IN ASTRONAUTICAL ENGINEERING

from the

Author:

December 1995

Author:

Donald R. Harder

Approved by:

Ashok Gopinath, Thesis Advisor

Oscar Biblarz, Thesis Co-Advisor

Matthew Kelleher, Chairman

Department of Mechanical Engineering

Daniel Collins, Chairman

Department of Aeronautics and Astronautics

iv

ABSTRACT

Experimental work was performed to study the convective heat transfer characteristics from a cylinder in a strong zero-mean oscillatory flow represented by an acoustic field. Two different flow regimes are discussed; that in which laminar, attached flow around the cylinder is present, and that in which instabilities, such as vortex shedding occur. The experiment utilizes a steady state measurement method. A transition from the laminar to the unstable regime was observed to occur at a streaming Reynolds number of approximately 240. Within the laminar regime, the transition from "intermediate" to "large" values of the streaming Reynolds number occurs at approximately 130. Heat transfer results for large values of the streaming Reynolds number in the laminar regime closely match the present theory (less than 13% error). Correlations were developed to relate the heat transfer rate to the streaming Reynolds number in the unstable regime. This work would find application in the design of heat exchangers for a thermoacoustic engine.

TABLE OF CONTENTS

I. INTRODU	UCTION		1
II. BACKGI	ROUND		3
A.	HISTORICAL		3
B.	RECENT WORK		4
C.	THERMOACOUSTIC PROCESS		5
D.	HEAT TRANSFER IN ACOUSTIC FIELDS		5
II. EXPERI	MENT	1	1
A.	INTENT	1	1
В.	FUNDAMENTAL IDEAS	1	2
	1. Criterion A	1	2
	2. Criterion B	1	3
	3. Criterion C	1	4
	4. Criterion D	1	5
	5. Criterion E	1	6
	6. Criterion F	1	7
	7. Criterion G		
	8. Basic Flow Description	1	9
C.	APPARATUS	2	1
	1. Test Cylinder	2	2
	2. Sound Chamber	2	4
	3. Acoustic Electronics Package	2	7
D.	EXPERIMENTAL METHOD	2	7
E.	EXPERIMENTAL CALCULATIONS	3	2
n, pro	a AND Diagraga		
IV. KESULT	S AND DISCUSSION	3:	5

A.	LAMINAR, ATTACHED FLOW REGIME	38
B.	SEPARATED FLOW REGIME	42
V. CONCLUS	SIONS	45
VI. RECOMM	MENDATIONS	47
APPENDIX A	A. CALIBRATIONS AND CALCULATIONS	49
APPENDIX B	B. UNCERTAINTY ERROR ANALYSIS	53
APPENDIX C	C. EXPERIMENTAL DATA	59
LIST OF REF	TERENCES	103
INITIAL DIST	TRIBUTION LIST	105

LIST OF FIGURES

1. Sondhauss Tube
2. Basic Thermoacoustic Process
3. Thermoacoustic Stack
4. Heat Exchanger
5. Pressure and Velocity Profiles in the Presence of an Acoustic Signal
6. Vortices Due to Oscillatory Flow Around a Cylinder
7. Outer Acoustic Streaming Flow
8. Experimental Test Apparatus
9. Test Cylinder Assembly
10. Photo of Test Cylinder inserted into the Sound Chamber
11. Sound Chamber Assembly
12. Photo of Test Apparatus
13. Proper Geometry so that Maximum Velocity occurs at the Test Cylinder 28
14. Parameter Map of Expected Heat Transfer Regimes
15. Data Plotted as a Function of Nusselt Number vs. Streaming Reynolds Number 37
16. Laminar, Attached Flow Regime for $R_s < 240$
17. Laminar, Attached Flow Regime for Large R_{s} with Uncertainty Bars 41
18. Unstable Regime
19. Data Plotted with Curve Fits for the Regimes of Interest
20 Thermocouple Arrangement for Calibration Tests

LIST OF TABLES

1. Sample of SPL and PR for Comparison	29
2. Thermocouple Calibration Data	50
3. Thermal Resistance and Linear Temperature Distribution Trials	52

LIST OF SYMBOLS, ACRONYMS AND/OR ABBREVIATIONS

test cylinder radius [m] a Α amplitude of particle oscillation [m] B aspect ratio speed of sound [m/s] С diameter of test cylinder [m] d diameter of sound chamber [m] D f frequency [Hz] Gr Grashoff number convective heat transfer coefficient [W/m²-K] h Ι current [Amps] k thermal conductivity [W/m-K] KC Keulegan-Carpenter number length of test cylinder [m] L distance from test cylinder to termination end plate [m] Mach number M NuNusselt number P power [W] $\mathbf{P}_{\mathbf{m}}$ mean ambient pressure [Pa] pressure level [Pa] P_0 reference pressure [Pa] P_{ref} pressure ratio PR gas constant [J/kg-K] R equivalent thermal resistance [K/W] R_{ea} R_s streaming Reynolds number pressure transducer sensitivity [mV/Pa] S SPL sound pressure level [dB] ambient temperature [K] T_a T_{c} center temperature [K] T_s surface temperature [K] particle velocity [m/s] U_0 voltage [Volts] V_0 Z aspect ratio β amplitude ratio polytropic coefficient δ Stoke's boundary layer [m] amplitude parameter ε wavelength [m] λ Λ frequency parameter kinematic viscosity [m²/s] ν cylinder length scale χ radian frequency [rad/s] ω

I. INTRODUCTION

The science of convective heat transfer in an acoustic field, while still in its infancy, presents many new and exciting possibilities for application in the future. Different experiments and theory have proven that under the correct circumstances there are certain desirable heat transfer characteristics affecting an object which is immersed in a strong acoustic field. A complete analysis concerning the processes and effects of the related heat transfer phenomena, though, is lacking and desperately needed.

Although thermoacoustics have already been applied to some advanced heat transfer designs, for instance, the thermoacoustic space refrigerator and other thermoacoustic cryocoolers developed at the Naval Postgraduate School, there has yet to be developed anything that can compete on an economic level with what is currently marketed today. The efficiencies obtained so far have been quite low, requiring nearly twice the power of a conventional vapor compression refrigerator. When the fundamentals behind the thermoacoustic phenomenon and the related heat transfer characteristics are completely understood, breakthroughs can occur which could allow industry to move ahead and apply these techniques on an every day basis toward a variety of common uses.

To properly model and control the parameters which impact upon the heat transfer behavior in a thermoacoustic engine, it would be advantageous if the various flow regimes (e.g., turbulent vs. laminar) in the engine could be isolated and analyzed in detail. Further information in this regard may be obtained through a parametric analysis of a suitable model problem by which a measure of the importance (or rather a magnitude of the effect each parameter has on the process as a whole) can be determined. This is an important element of the modeling process and requires study since by understanding the impact that each individual parameter makes upon the thermoacoustic process as a whole, it may be possible to predict the changes in the heat transfer characteristics as individual components are varied.

The work contained within provides an experimental study of some of the dominant heat transfer properties of a particular model problem that may be encountered in thermoacoustic engines. The model problem chosen is one of convective heat transfer from a cylinder in a zero-mean oscillatory flow. The flow is representative of the acoustic standing wave in a thermoacoustic engine whereas the cylinder represents a tube or other component that may be present in such an engine.

The work involves a correlation of experimental heat transfer data in terms of a suitable Nusselt number (Nu) with other appropriate dimensionless parameters in the problem, such as the streaming Reynolds number (R_s), which itself is a function of length scales, pressure ratios and frequency parameters and, of course, of the Prandtl number (Pr). Through use of high power standing resonant acoustic waves in a cylindrical chamber, a high-intensity internal oscillatory flow is established. Under these conditions, the heat removal rate from a thin cylindrical heating element immersed in the acoustic signal will supply data necessary to arrive at some basic conclusions as to heat transfer phenomena occurring around a cylinder. This work focuses upon two different flow regimes; one in which laminar, attached flow around the cylinder at large values of the streaming Reynolds number is present, and the second in which vortex shedding and other instabilities in the flow are expected to occur at the cylinder surface. The resultant experimental data additionally provides guidelines for determining when the flow transitions from one regime to the other.

II. BACKGROUND

A. HISTORICAL

It has long been understood that a large temperature gradient along the length of a cylindrical tube can, under certain suitable circumstances, spontaneously excite the fluid into oscillations strong enough to create audible sound. Glass blowers provided the earliest accounts of this acoustic effect. They found that when one end of a glass tube was placed in a furnace, the temperature difference between the end of the tube in the furnace and the end still under ambient conditions created an audible tone which was emitted from the open end of the tube. Even though early scientists knew of this effect, it was merely considered to be more of an oddity than a scientific discovery which might have useful implications. In fact, the earliest accounting of what may have been the first ever thermoacoustic engine, the Sondhauss tube (Figure 1), was described by Sondhauss (1850) himself as the "glowing glass harmonica". Lord Rayleigh (1945) gave the first good qualitative analysis of the Sondhauss tube in 1896 in which he described the mechanism behind the effect and posed the theory that mechanical work could be obtained from the vibrations, or oscillations, which were being created by the temperature gradient along the tube.

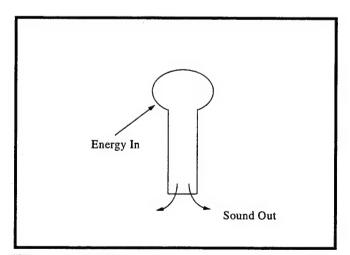


Figure 1. Sondhauss Tube

An extension of research into Sondhauss tubes was conducted by Carter (1988) and included placing a stack of plates at an appropriate point in the tube. The plates included hot heat exchanger strips at one end and cold heat exchanger strips at the other. These improved the effect of the Sondhauss tube, and inspired another scientist, Feldman (1966) to conduct similar research which consequently resulted in an oscillator which produced 27 W of acoustic power from 600 W of thermal power. All of this occurred in the 1960s.

It wasn't until much later that it was postulated that the effect could somehow be reversed, i.e., a temperature gradient could be created along a tube using a powerful, resonating acoustic signal as the driving force. It is only more recently that advances in acoustic technology have allowed serious research toward this goal.

B. RECENT WORK

The heat transfer effect related to this thermoacoustic phenomenon has since been used in the creation of heat pumps and has been explored by several scientists. The results of these attempts include what was termed a pulse-tube refrigerator as developed by Gifford and Longsworth (1966). This refrigerator utilized low frequency, high amplitude oscillations to excite the gas in a tube and create a cooling effect along the surface. The invention of "modern" thermoacoustic refrigeration occurred in the early 1980s at Los Alamos National Laboratory. It was in essence a modification to the work that Carter did, using stacks of plates with a much smaller temperature gradient. Additional engineering developments by others, such as the work at the Naval Postgraduate School with a thermoacoustic refrigerator intended for use on the space shuttle, led to increases in efficiency, as well as to an increase in commercial interest and development. Currently, there are major projects ongoing in several countries as outlined by Swift (1995), including a prototype food refrigerator based upon the Naval Postgraduate School's work being built in the Republic of South Africa. The Ford Motor Company has developed its own version of a thermoacoustic refrigerator while the Tektronix Corporation is working towards a pulse-tube type of refrigerator to be used for cooling electronics to cryogenic temperatures.

All of these attempts are especially significant given the current stigma surrounding the environmentally hazardous use of CFC's. Even though thermoacoustic refrigeration is

still not as efficient as that of current energy efficient vapor compression models, there is a growing demand for something to take their place. By advancing our knowledge-base in this area, and further incorporating a new understanding of how increases to the efficiency of thermoacoustic designs can be made, that it may well be possible that a new thermoacoustic revolution is in our future.

C. THERMOACOUSTIC PROCESS

The basic process behind a thermoacoustic engine can be best described by the model in Figure 2. The upper portion of the figure shows a sound chamber with an acoustic driver at the left end which is used to create a resonant, standing wave in the chamber. At an appropriate point within the chamber, a thermoacoustic stack (Figure 3) is placed with a heat exchanger (Figure 4) on either side of it. The flow of heat is from right to left in the figure. The process of heat transport across the plate is illustrated in the lower portion of the figure. The fluid within the chamber will oscillate due to the acoustic wave, traveling from a point of low pressure to one of high pressure, gaining and losing energy during each half cycle.

For instance, a parcel of gas at temperature T_0 at low pressure moves along line 1, increasing in pressure and temperature until it reaches T++ (has gained two units of heat). At that point, it loses one unit of heat to the plate, thereby reducing its temperature to T+. As the parcel of gas continues through the second half of the oscillatory cycle, it decreases in pressure, and it loses two unit of heat, dropping to T-. It is then able to retrieve a single unit of heat from either the plate at the right hand side of the cycle, or from a heat exchanger at that end. In effect, a "bucket-brigade" of little parcels of gas is formed as the heat transport mechanism. It is the heat exchangers on either end of the thermoacoustic stack and the heat transport mechanisms involved with them that this experiment intends to analyze.

D. HEAT TRANSFER IN ACOUSTIC FIELDS

The early 1950's saw an increased interest towards understanding the heat transfer behavior in oscillatory flows and an earnest effort towards understanding the possible benefits thereof began. Richardson (1967) produced the first significant contribution to this field by providing a coherent and detailed account on the general nature of heat transfer in oscillatory flows. He gave concise documentation on how sound and vibration fields had

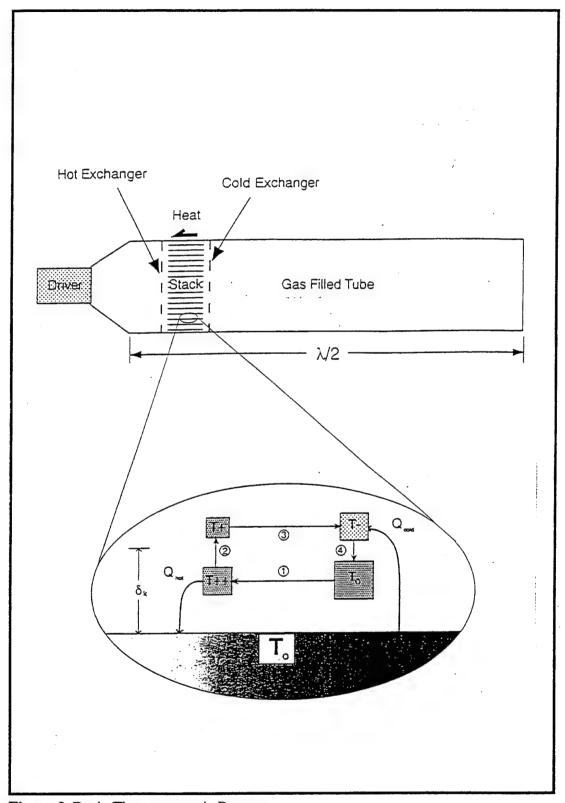


Figure 2. Basic Thermoacoustic Process.

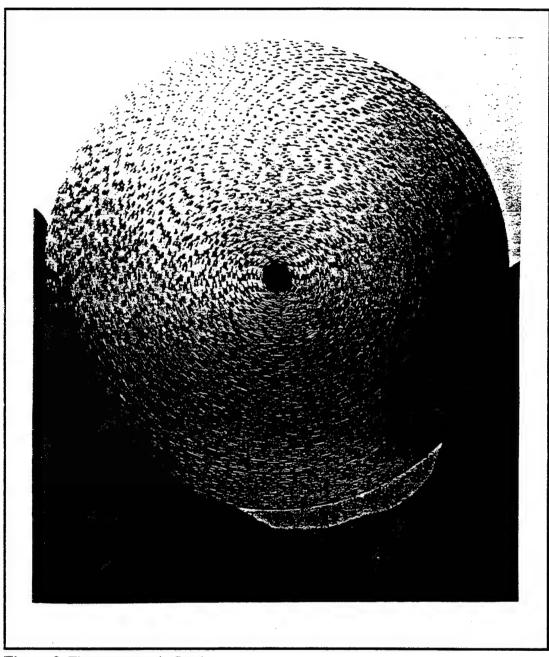


Figure 3. Thermoacoustic Stack.

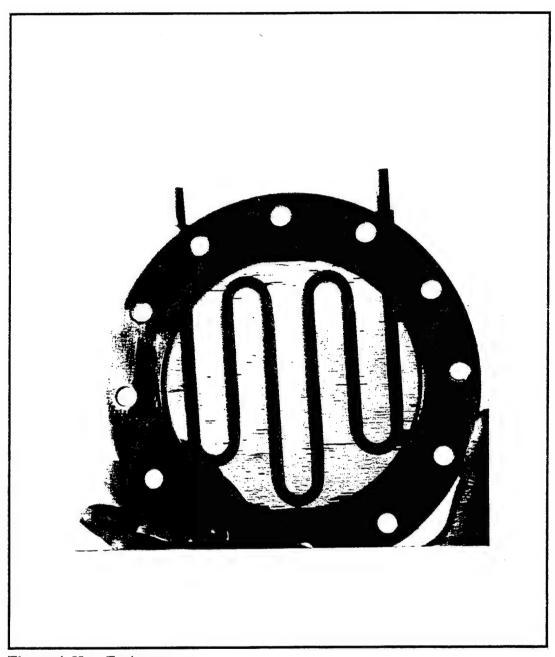


Figure 4. Heat Exchanger.

made an impact in the heat transfer field up to that point. He additionally conducted a preliminary study on how the effect can be related to that of a more traditional fluid-mechanical analysis. Davidson (1974) further expanded upon Richardson's work, analyzing the heat transfer behavior of cylinders in oscillatory flow. That occurred in the early 1970's, and since then, very little work on the subject can be found in literature.

Most recently, though, Mozurkewich (1995) performed heat transfer experiments in an acoustic field utilizing a transient analysis of a heated wire. His results, although informative in some respects, were lacking in data within the most basic flow regimes and his conclusions left some doubt to the reader.

When first attempting to analyze the process behind convective heat transfer in an acoustic field, it is often simpler to think of the heat transfer phenomenon as something akin to that of forced convection due to a steady mean flow. Initially, that there is a separate power source placed away from the test object which produces a disturbance in the fluid medium in which that object is immersed. In forced convection, that power source may be considered to be a fan or a pump which creates a pressure gradient, which in turn causes a steady flow of fluid. In the current problem, the power source is instead an acoustic driver which causes an oscillatory (or vibrational) type of time-periodic flow around the object being considered. This oscillatory flow has a zero-mean and results in no net through flow. Before analyzing this flow for heat transfer characteristics, though, it is necessary to first note which aspects of the acoustic field dominate.

A resonant, standing wave acoustic field excited across the ends of a closed cylindrical chamber has very distinct properties. Of particular interest is the fact that when such a field exists, the pressure along the length of the chamber varies sinusoidally such that a point of maximum pressure occurs at the rigid end termination at the opposite end of the chamber from which the acoustic signal is being generated. In addition, depending upon the frequency being used, there may be more than one zero-crossing, or pressure minimum, along the length of the chamber (Figure 5).

At resonance, the acoustic velocity is out of phase with the pressure. For instance, at a point of minimum pressure, a pressure node, the particle oscillations will be at their point of maximum velocity, a velocity antinode. The reverse then also holds true, (i.e., a pressure

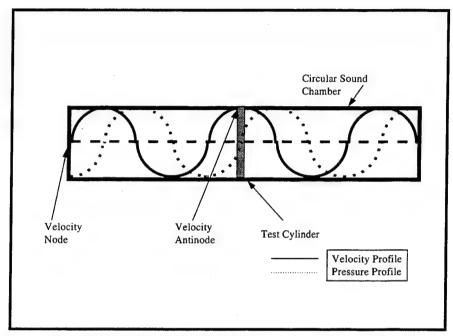


Figure 5. Pressure and Velocity Profiles in a Circular Tube in the Presence of an Acoustic Signal.

antinode can also be designated a velocity node).

To understand the reason why this relationship is extremely valuable to this research, it is useful to again refer to the forced convection model. In general, as the fluid velocity in a forced convection application increases within a particular regime, so does the heat transfer coefficient. The case is essentially the same for heat transfer in an acoustic field, and hence it is expected that the heat transfer rate would also increase as the particle velocity increases. Thus, for experimental purposes it is of prime importance that the test object be placed at a velocity anti-node to get the maximum effectiveness out of the process for a given acoustic signal. However, in contrast to the forced convective mean flow case, the current issue with oscillatory flow is considerably more complicated due to the wide range of flow parameters, and hence flow patterns and heat transport mechanisms, that can result.

II. EXPERIMENT

A. INTENT

As with any other advance in technology or new scientific discovery with which engineers desire to predict and quantify results in some manner, it is best to begin by first breaking the process down into its basic component parts. By completing an analysis for the simplest version of the model in question, a stepping stone will be established upon which the analysis of more complicated scenarios can be built. It stands to reason that this is how the analysis of the thermoacoustic heat transfer process should also begin.

For an initial starting point, the analogy once again to forced convection heat transfer is used. The problem of an isolated cylinder in a mean cross flow is well documented and understood, and has in turn been used to develop correlations for flow over a collection of cylinders, such as a tube bank, a more practical application as is evident from any basic heat transfer textbook. This is the motivation for a study of the behavior of simple shapes such as a cylinder placed in an acoustic field. It is hoped that with this knowledge for an isolated cylinder, it would be possible to extend the solution to other models.

It is only necessary then to concentrate on the evaluation of the acoustic signal itself in terms of its many different parameters. But before this analysis may begin, the type of acoustic flow around the cylinder must be established. To allow corroboration with established theory, the flow pattern initially desired is that of basic laminar, incompressible flow where well understood streaming patterns are the principal forms of flow and heat transport. This meets the requirement for maintaining the most simplified version of flow for the analysis.

The geometry of how the test cylinder is placed with reference to the acoustic signal is also of utmost importance. A theoretically perfect scenario would have the test cylinder situated normal to a unidirectional sound field with no interference from the surroundings. Such a situation cannot be exactly duplicated, though, due to the nature of acoustic waves to spread and travel in all directions. In order to restrict the acoustic signal to only one

direction, it is proposed that the sound field be set up in a cylindrical, resonant sound chamber so that the analysis is properly limited to axial wave modes only.

B. FUNDAMENTAL IDEAS

Unlike the well known dependence on the Reynolds and Prandtl numbers found in conventional cross flow over bluff bodies, the issue of heat transfer in the presence of an acoustic field is significantly more complicated due to the presence of a multitude of competing length (or time, or velocity) scales. The ways in which these length scales may be ordered are many and lead to numerous distinct parameter regimes with quite drastically different flow properties, and hence heat transfer properties. In order to closely examine the properties of heat transfer in oscillatory flows then, it is first necessary to enumerate some of the different parameters and variables involved. By establishing and maintaining a set of criteria surrounding these parameters, and by modeling the test apparatus to conform to them, the job of evaluating different regimes of flow and drawing correlations from the data obtained will become possible.

This section lists the most important of these parameters and provides reasoning for the choices involved, keeping in mind the desire to maintain the most basic and core set of conditions that flow around the test cylinder be laminar, incompressible and attached, and that only the steady-state solution be considered.

1. Criterion A

Following the first assumption that the acoustic streaming flow around the test cylinder be incompressible, two different length scales must be specified. The first of these requires that the relative size of the test cylinder be very small compared to the radian wavelength of the acoustic field, where the radian wavelength is defined as

$$\overline{\lambda} = \frac{\lambda}{2\pi} \tag{1}$$

and the characteristic length scale of the test cylinder is chosen as the radius, a. The radian wavelength can be related to the frequency by:

$$\overline{\lambda} = \frac{\lambda}{2\pi} = \frac{c/f}{2\pi} = \frac{c}{\omega} \tag{2}$$

Now, by asserting that $a \ll \overline{\lambda}$, and designating the ratio between the two as χ , it follows that:

$$\frac{a}{\overline{\lambda}} \ll 1$$
 (3)

and finally that

$$\chi = \frac{a\omega}{c} \ll 1 \tag{4}$$

The criterion $\chi \ll I$ ensures that radiation effects due to the acoustic streaming are negligible, as presented by Lighthill (1963) and indicates that it is only the local acoustic field conditions that are of importance.

2. Criterion B

The second criterion which is required to support the assumption that flow be incompressible is derived from the relationship between displacement amplitude of particle oscillation in the sound field, \overline{A} , and the cylinder radius, a. The ratio between the two, $\overline{\frac{A}{a}}$, will be designated as the amplitude parameter, ϵ , and dictates whether or not separation will occur.

When the amplitude parameter is very small,

$$\epsilon = \frac{\overline{A}}{a} \ll 1$$
 (5)

the particles in the sound field move a very short distance along the cylinder wall before reversing their direction. This ensures that the flow remains attached, with little chance of separation occurring, and hence the flow will remain laminar at all times. It can also be noted that ϵ is directly proportional to the pressure ratio P_0 / P_m , and can therefore take on much larger values for strong acoustic fields. This is accomplished by observing that the displacement amplitude of a particle in the flow is directly related to its velocity and that the

particle velocity is in turn related to the pressure ratio by the following (in a plane standing sound field)

$$\overline{A} = \frac{U_0}{\omega} \tag{6}$$

and

$$U_0 = \frac{cP_0}{\gamma P_m} \tag{7}$$

therefore

$$\epsilon = \frac{c}{a\omega} \left(\frac{P_0}{\gamma P_m}\right) \ll 1 \tag{8}$$

Yet another form of this parameter often used in the literature is called the Keulegan Carpenter number and is defined as $KC=U_0/2af$ and can additionally be expressed as

$$KC = \pi \epsilon$$
 (9)

Of importance is the fact that the product of the parameters defined in the first two criteria (A and B) is the flow Mach number, which can be defined as

$$M = \chi \epsilon = \frac{U_0}{c} \tag{10}$$

When criteria A and B are satisfied, $M \ll 1$, and this in turn is the second condition which satisfies the assumption of incompressible flow.

3. Criterion C

The Stokes boundary layer thickness δ is related to the kinematic viscosity and the radian frequency of oscillations by

$$\delta = \sqrt{\frac{v}{\omega}} \tag{11}$$

and is the well known length scale which is a measure of the extent of viscous effects in an oscillatory flow. A frequency parameter Λ^2 can be defined as follows

$$\Lambda^2 = (\frac{a}{\delta})^2 = \frac{a^2 \omega}{v} \tag{12}$$

For the case when $\Lambda^2 \gg 1$, the Stokes shear layer is confined to a narrow region and the acoustic streaming effect appears as a slip velocity along the cylinder surface. Utilizing the knowledge that the boundary layer thickness is on the order of 10δ and imposing the condition (somewhat arbitrarily) that

$$\frac{a}{10\delta} > 4 \tag{13}$$

it follows that

$$\Lambda^2 > 1600 \tag{14}$$

is a good criterion to ensure "large" values of the frequency parameter.

The frequency parameter may also be often found in the literature in the form of $\beta = (2a)^2 f/v$, and can be expressed as

$$\beta = (\frac{2}{\pi})\Lambda^2 \tag{15}$$

4. Criterion D

When criteria A - C are satisfied, the acoustic streaming velocity is of magnitude $O(\epsilon U_0)$. A streaming Reynolds number, R_s , can then be defined as

$$R_s = \frac{(\epsilon U_0)a}{v} \tag{16}$$

Through substitution for ϵ and U_0

$$R_{s} = \frac{U_{0}}{a\omega} \frac{U_{0}a}{v} = \frac{U_{0}^{2}}{\omega v} = \frac{c^{2}}{\omega v} (\frac{P_{0}}{\gamma p_{m}})^{2}$$
 (17)

which yields

$$R_s = \epsilon^2 \Lambda^2 \tag{18}$$

This streaming Reynolds number becomes the driving factor in determining what will be the primary mode of heat transport within the region due to the acoustic streaming flow. Stuart (1966) demonstrated that when $R_s \ll I$, a Stokes flow becomes prevalent in the outer region while a boundary layer flow is predominant when this parameter takes on values much greater than one. In order to ensure that forced convective heat transfer is dominant then, we impose the following constraint

$$R_{\rm s} \gg 1$$
 (19)

5. Criterion E

It was first observed by Honji (1981) that flow around a cylinder will become centrifugally unstable and separate into vortices as the amplitude of particle oscillation increases. This instability occurs in the Stokes layer where the flow is parallel to the direction of particle oscillation. This was confirmed by Hall (1984) who conducted a linear stability analysis on the unsteady boundary layer in the high-frequency limit. He found that a critical value of the Reynolds streaming number exists for which instabilities begin to form, namely when R_s becomes greater than 4.24Λ . Recent work in the area by Sarpkaya (1986) provides further explanation for this phenomenon. Since vortex shedding can make a large impact on the heat transport from the cylinder, we will maintain the following criterion, which can be expressed in two different ways as

$$R_{s} < 4.24\Lambda \tag{20}$$

or, alternately as

$$\epsilon < \frac{2.06}{\sqrt{\Lambda}} \tag{21}$$

An example of what these vortices may look like is shown in Figure 6.

6. Criterion F

In order to maintain the condition in which there is minimal influence from the buoyancy effects of natural convection as compared to the forced convective heat transfer due to the acoustic streaming effects, a dimensional analysis of the governing equations produces the following requirement for the Grashof number

$$\frac{Gr}{R_s^2} \ll 1 \tag{22}$$

For the case when $Gr / R_s^2 \approx 1$ or greater, buoyancy effects must be taken into account and any heat transfer correlations developed will have to be modified accordingly.

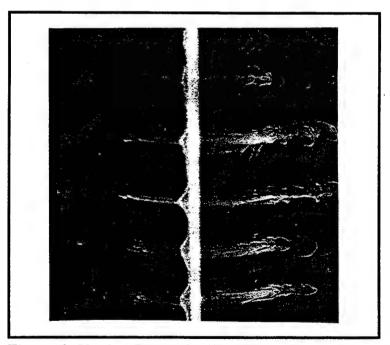


Figure 6. Vortices Due to Oscillatory Flow Around a Cylinder. (Sarpkaya, 1986)

7. Criterion G

This final criterion stems from the desire to maintain only a plane standing sound field within the test chamber. This means that only the axial wave components excited by the resonant acoustic signal be present, without interference from transverse modes such as the azimuthal modes and radial modes. It may be shown from theory that there is a certain cut-off frequency developed from a solution of the wave equation, below which the transverse modes will not be present. For the cylindrical waveguide geometry being used, the solution for the above condition is obtained in the form of appropriate roots of Bessel functions. If L is the length of the test chamber, the longitudinal frequency can be expressed as

$$f_l = \frac{lc}{2L} \tag{23}$$

where l is the 1^{st} , 2^{nd} , ..., mode number. The transverse frequency mode is expressed by

$$f_{mn} = \alpha_{mn}' \frac{c}{\pi D} \tag{24}$$

where D is the test chamber diameter and α'_{mn} represents the eigenvalues obtained from roots of the Bessel functions. Transverse modes will be present if $f_l = f_{mn}$, so it is desired to maintain f_l well below f_{mn} and f_l now represents the maximum frequency possible while still maintaining this criterion. By substituting for both frequencies, the condition becomes

$$\frac{lc}{2L} < \alpha'_{mn} \frac{c}{\pi D} \tag{25}$$

which is reduced to

$$\frac{l}{2(\frac{L}{D})} < \frac{\alpha'_{mn}}{\pi} \tag{26}$$

By introducing a new parameter called the aspect ratio,

$$Z = \frac{L}{D} \tag{27}$$

and finding the smallest root of this Bessel function,

$$\frac{\alpha'_{mn}}{\pi}min = 0.586 \tag{28}$$

the criterion now becomes

$$\frac{l}{2Z} < 0.586$$
 (29)

which can be rearranged to finally get

$$Z = \frac{L}{D} > 0.85 l_{\text{max}} \tag{30}$$

where

$$l_{\text{max}} = \frac{f_{\text{max}}}{f_{\text{min}}} \tag{31}$$

This gives a relationship between the geometry and the maximum frequency. But since the maximum frequency is already defined using criteria A - F, criterion G in fact gives us the maximum diameter that can be used, or if the diameter is also given, it determines the length of the chamber instead.

8. Basic Flow Description

When a cylinder is immersed in a standing acoustic field and all of the previous criteria have been met, particle oscillations will initiate the most basic form of an acoustic streaming flow. This steady flow (Figure 7) is symmetrical about two axis, is circular in nature, and includes a well defined boundary layer. The boundary layer is actually quite small and is greatly exaggerated in the figure for clarity. Although this is not the only flow which is present, it does represent the flow pattern that has the biggest impact as a heat transport mechanism when these criteria are satisfied.

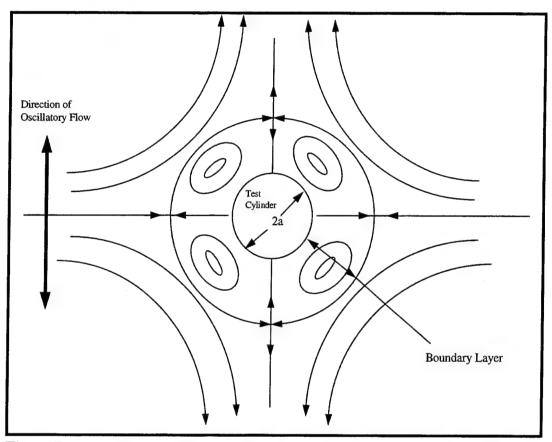


Figure 7. Outer Acoustic Streaming Flow (also known as Boundary Layer Flow)

As discussed earlier, this fluid flow regime is the main focus of this investigation in which at first we consider only laminar, unseparated flow around the test cylinder. One of the goals of this experimental study is to determine the range of values for the previously discussed parameters where one flow regime transitions to another and relate them in terms of the streaming Reynolds number. It is believed that such transitions will be reflected in the heat transfer behavior.

In order to define a specific range for laminar flow, though, these criteria needed to refinement by asking the questions; how small is "much less than one" or how big is "much

greater than one"? It was determined that Criterion A in Eq. 4 would be met by picking $\chi < 0.1$ and Criterion B in Eq. 5 would be met by choosing $\epsilon < 0.3$. This ensures incompressible flow around the test cylinder and that flow remains attached. Criterion C becomes $\Lambda^2 > 1600$, confining the Stokes shear layer to a narrow region. The above ranges have been fixed based on prior experience and preliminary experimentation, but are not by any means intended to be "hard and fast". They may indeed be modified if necessary. Criteria D and E require verification after each experimental run.

C. APPARATUS

As previously discussed, the experiment consists of a heated test cylinder placed normal to a simple acoustic standing wave within a resonating sound chamber (Figure 8). This general description can be broken down into three major components; the test cylinder, the sound chamber and the acoustic electronics package, which are further described below.

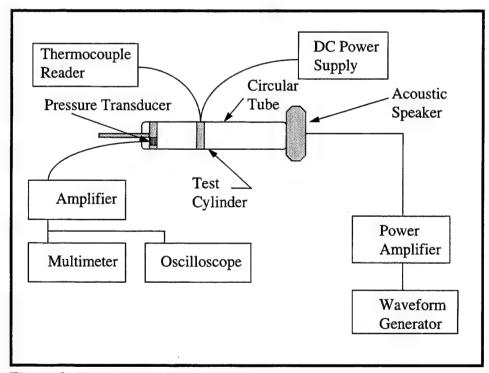


Figure 8. Experimental Test Apparatus.

1. Test Cylinder

It was originally proposed that a Watlow stainless steel cartridge heater be used as the test cylinder for the experiments. They are available in varying diameters and additionally feature an imbedded type "J" thermocouple placed at the midpoint of its length. This presented problems, however, since the heating along the length of the cartridge heater was uneven and the relatively large thermal resistance of the stainless steel produced large variances in surface temperature along the length. Since constant surface temperature is a feature which is very important to the analysis of heat transfer characteristics, this was unacceptable. Therefore, a copper sheath was designed to fit over the cartridge heater. It was assumed that the large thermal conductivity of copper would even out the axial surface temperature gradient. In addition, silicon oil was liberally applied to the inside of the copper sheath prior to insertion of the cartridge heater before each experimental run (Figure 9). This provides better thermal contact in the narrow annular gap (approximately 2 mm) between the sheath and heater, preventing air gaps which can cause local temperature discontinuities at the surface. This arrangement (as was later verified, Appendix B) provides for an axial temperature variation of less than 0.5°C from one end of the heater to the other for the temperature range of interest in this experiment. A picture of the test cylinder is shown in Figure 10.

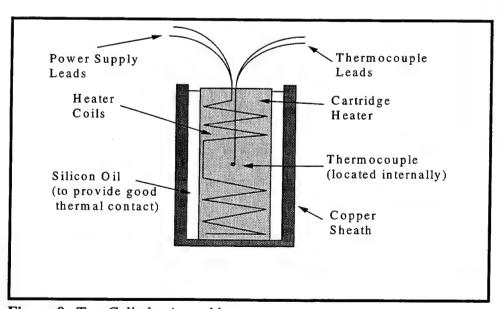


Figure 9. Test Cylinder Assembly.

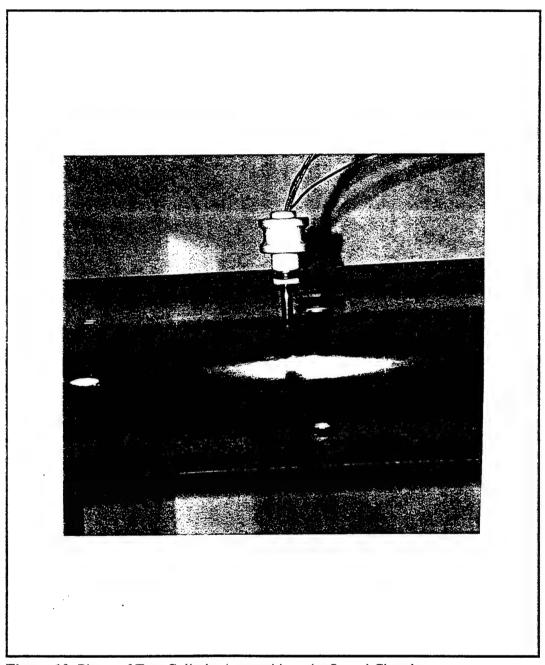


Figure 10. Photo of Test Cylinder inserted into the Sound Chamber.

As was stated earlier, the surface temperature of the test cylinder is a required datum point for the analysis. Since the only provision for temperature measurement is at the thermocouple placed at the center of the test cylinder, the equivalent resistance of the steel/oil/copper circuit is required in order to deduce the surface temperature from the cartridge heater center temperature as measured by the imbedded thermocouple. Appendix B gives a detailed analysis of the derivation of the resistance, which is approximately 1.019 K/W.

The cartridge heater receives its power from a Kikusui Model PAR 160A regulated DC power supply. It provides power control measurement down to 0.01 amps and 0.01 volts. Thermocouple measurements are provided by a Keithley Model 740 scanning thermometer system. Calibration data for all thermocouples and the thermocouple reader is provided in Appendix B.

2. Sound Chamber

The purpose of the sound chamber is to provide an environment through which acoustic signals of various frequencies can be used to excite a resonant standing wave. In order to accomplish this for all frequencies which may be used during the experiment, a resonant chamber that would be adjustable in length was highly desired. Figure 11 shows the final configuration of the test chamber while Figure 12 shows a photo of the test apparatus.

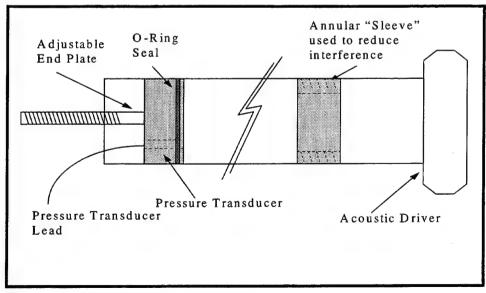


Figure 11. Sound Chamber Assembly.

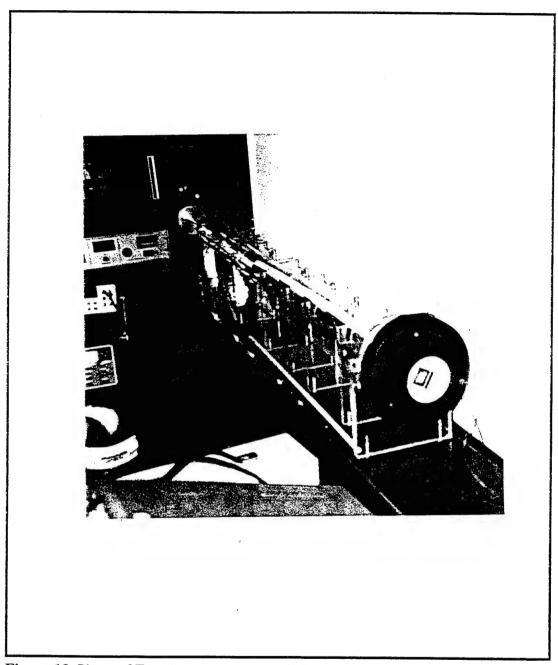


Figure 12. Photo of Test Apparatus. Acoustic Driver is on to the right.

The test chamber itself is a plexiglass tube approximately two meters long which is mounted firmly to a plexiglass base plate. The acoustic driver is mounted at one end and provides the source of the acoustic signal. The opposite end has a movable flat end plate which has an o-ring seal to help isolate the acoustic wave in the chamber. The end plate serves as the rigid end termination while the o-ring provides good sound confinement, as well as good stability for the end face so that it doesn't become offset to either side as its position is varied along the length of the enclosing resonant chamber.

A hole through this end plate provides access for a pressure transducer which is used to help deduce the pressure ratio, and hence the sound pressure level for that configuration of frequency and input power. The pressure transducer used is an Endevco model 8510B-5 which has an output in millivolts and has a pressure sensitivity of 50.89 mV/psi. This is connected to a preamplifier with a gain of 100 and provides output to both an oscilloscope and a Hewlett-Packard model 34401A multimeter. Measurement of the voltage output from the multimeter is important to determine when resonance has occurred within the chamber for as the frequency is varied, the output voltage from the microphone decreases on either side of the resonant operating point.

The oscilloscope provides a visual representation of the time trace of the acoustic signal at the end of the sound chamber, corresponding to a pressure antinode, and is used to ensure that the signal remains sinusoidal throughout the experimental range of powers and frequencies. During the initial stages of the experiment, the oscilloscope allowed for the discovery of interference patterns in the sinusoidal waveform as caused by higher order harmonics at high SPLs (> 155 dB). In order to limit the interference that was present, the use of "sleeves" within the chamber was recommended to detune the resonant mechanism. This would prevent the harmonics from being integral multiples of each other and thereby prevent them from reinforcing each other to form interference patterns. By placing a sleeve at an appropriate spot in the chamber, some of the high frequency harmonics leading to interference could be eliminated, allowing for even higher SPLs to be achieved before interference occurred.

3. Acoustic Electronics Package

Of great importance to the experiment is the ability to generate a nearly pure sinusoidal waveform at varying frequencies and high power ranges. As the strength of the signal generated increases, the effect of the flow around the test cylinder becomes more pronounced, enhancing the heat transfer characteristics of the system.

The acoustic signal being generated at the driver end of the sound chamber is provided by a Hewlett-Packard model 33120A arbitrary waveform generator. It sends a sinusoidal waveform at the proper frequency through a Techron model 7540 power supply amplifier to a JBL model 2490H acoustic compression driver.

A more detailed review of each item of equipment used in the system is provided in Appendix D.

D. EXPERIMENTAL METHOD

Prior to gathering data for this experiment, it was first necessary to develop a coherent plan with which to approach the problem. The first step required was to find specific frequencies at which resonance occurred within the chamber, and which would also provide a velocity anti-node at the position of the test cylinder. By looking at the geometry of the problem (Figure 13), it became obvious that these limiting factors could be met by a combination of adjusting the end plate distance from the test cylinder, as well as the signal frequency, so that the length L from the heated cylinder to the end plate termination was an odd multiple of $\lambda/4$. This could be further refined to state that the value of 4Lf/c needs to be an odd integer. When this condition was satisfied, the requirement of a velocity anti-node occurring at the cylinder location was met.

The next thing needed was an estimate of the maximum pressure ratio, and therefore the sound pressure level that could be obtained. This was achieved by increasing the amplitude of the input signal waveform until the output waveform on the oscilloscope began showing traces of interference or other disturbances. When the maximum input amplitude was obtained, the multimeter output was recorded.

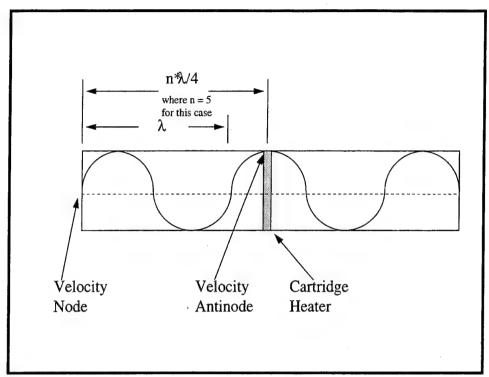


Figure 13. Proper Geometry so that Maximum Velocity occurs at the Test Cylinder.

In order to convert the output voltage to an actual pressure ratio and sound pressure level, it was necessary to first understand what form the multimeter uses to present the output. The multimeter gives the voltage output in terms of the true RMS value of the sinusoidal signal, or rather $V_0/\sqrt{2}$, where Vo represents the voltage amplitude from zero to peak of the sinusoidal signal. Recall that the true RMS value for the pressure can also be expressed as $P_0/\sqrt{2}$. Then, the following relationship holds true, that

$$P_0/\sqrt{2} = \frac{V_0/\sqrt{2}}{S} \tag{32}$$

where S is the sensitivity of the pressure transducer in mV/Pa.

The sensitivity of the pressure transducer after passing through the 100 gain setting of the preamplifier was converted and expressed as 0.74 mV/Pa. By substituting into the above equation, the pressure amplitude becomes

$$P_0 = \frac{V_0/\sqrt{2}}{0.523mV/Pa} \tag{33}$$

where it is noted again that V_0 / $\sqrt{2}$ is the multimeter output.

From this, a pressure ratio can be defined as

$$PR = \frac{P_0}{P} \tag{34}$$

 $PR = \frac{P_0}{P}$ where P_m is the mean ambient pressure of 101 kPa. The pressure ratio is typically expressed in terms of a percentage.

The sound pressure level is defined as the logarithmically scaled ratio of the RMS pressure and a predetermined reference pressure, P_{ref} , where P_{ref} is chosen to be 20 μPa for gases (by convention). This gives

$$SPL = 20 \log_{10} \frac{P_0/\sqrt{2}}{P_{ref}}$$
 (35)

In order to obtain an idea of how strong the acoustic signal is during the experiments, Table 1 gives the sound pressure level for various activities

Activity	Sound Pressure Level	Pressure Ratio	
	(dB)	(%.)	
Normal Conversation	60	< 0.001	
Jet Airplane at Take-off	90	< 0.01	
Pain Threshold	120	0.28	
Minimum Experimental Level	150.6	0.94	
Maximum Experimental Level	161.2	3.20	

Table 1. Samples of SPL and PR for Comparison.

After finding the pressure ratio, and therefore the SPL, the actual data extraction phase of the experiment could be initiated. It was preceded by with an understanding of what type of data was needed for analysis. A crucial element of this experiment is the need to obtain a broad base of data with which to incorporate the results. In order to obtain this, experiments were run at several different pressure ratios for each resonant frequency found, starting from as low as 0.9% and building up to the maximum pressure ratio by increments of 0.1%. In order to obtain a spread of data points at each pressure ratio evaluated, the test cylinder was heated to approximately 8, 12 and 16 degrees above the ambient temperature. This kept the power requirements low and reduced the amount of thermal input into the ambient air within the sound chamber. This latter point was necessary as the ambient temperature could rise as much as 0.3 degrees during a single experimental trial. In order to ensure reproducibility and reduce the effect of anomalistic behavior, three runs at each temperature point were conducted to ensure consistency of data. The selected temperatures were only guidelines and were not meant to be hard set points for the experiment. Instead, they were treated as aim points with an acceptable range of ± 1 degree. Therefore, in order to produce an even broader spread of data, the power input to the test cylinder was varied slightly for each of the three trials at each specific temperature point.

Once the selected frequencies and pressure ratios had been determined and a suitable starting pressure ratio and temperature had been obtained, the experimental process was initiated. In order to obtain the selected pressure ratio for a particular set of runs, it was necessary to get an idea of the settings required for each specific piece of equipment. This was done by selecting the appropriate frequency on the waveform generator and modifying the power amplification on both the waveform generator and the power amplifier until the appropriate pressure ratio was obtained from the multimeter. The frequency was then adjusted to fine tune the resonance. A check of the oscilloscope at this point ensured that the signal being generated was of the right waveform and that interference was not occurring. Then the power amplifier to the acoustic driver was turned down to zero after noting the level at which it was set. This allowed for obtaining the correct pressure ratio in a quick manner by simply turning the power amplifier up to the previously noted value.

It was necessary at this point to ensure that the test cylinder was properly prepared. This entailed introducing approximately ten drops of silicon oil into the copper sheath and inserting the cartridge heater. The cartridge heater would be completely immersed in the silicon oil after being fully inserted. After a period of time, there would be some loss of silicon oil due to the wick action of the thermocouple and power leads emerging from the top. This was insignificant during the runs required for a single pressure ratio and caused a negligible change in the center-to-surface resistance as noted in Appendix B, but it became good practice to add some of drops of oil each time a new set of runs at a different pressure ratio were to be taken.

The test cylinder was then inserted into the sound chamber, making sure that the bottom of the cylinder was resting in a shallow indentation specifically machined into the inside face of the chamber. Power to the cartridge heater was then turned on and set so that a steady state temperature of approximately eight degrees above ambient would occur when the acoustic signal was present. This became more of an art form and required familiarity with the system to accomplish it with any degree of accuracy.

Since the power to the acoustic driver at this point was still at zero, the power to the cartridge heater would drive the temperature of the test cylinder past the projected steady state temperature. As it approached the projected temperature, though, the power to the acoustic driver was increased to the previously noted set point. This provided the fluid flow at the predetermined pressure ratio, in effect beginning to cool the test cylinder until it reached a steady state condition. At this point, the rate of energy input to the cylinder was equal to the rate of energy being convected away from the cylinder. By monitoring the interior thermocouple temperature, it was easy to see when the lowest temperature was reached. When the temperature began to rise once again, it was determined that the steady state condition had been reached and that the resultant temperature rise being witnessed was due only to the test cylinder transferring heat into the surrounding fluid medium, thus raising the overall ambient temperature.

Two situations other than the ideal one presented above occurred frequently due to the coarse means of trying to arrive at the desired cylinder temperature. If upon engaging the power to the acoustic driver the temperature of the test cylinder did not decrease but continued to increase at a slower rate instead, then the power being supplied to the cartridge heater was deemed too high and the voltage reduced until a decrease in temperature was witnessed. If upon engaging power to the acoustic driver the temperature of the test cylinder were to continue decreasing lower than the desired temperature range, it was an easy matter to increase the voltage to the cartridge heater to increase the steady state temperature solution.

When the steady state solution point was reached, the frequency and microphone voltage were noted, as was the voltage and current supplied to the cartridge heater and the temperature of the thermocouple in the cartridge heater. The power to the heater and the power to the acoustic driver were then simultaneously turned off and the test cylinder was quickly removed from the sound chamber. A second thermocouple was then introduced through the access hole in the sound chamber (from which the test cylinder had just been removed) such that the location of the thermocouple was approximately at the center (i.e., along the axis) of the sound chamber. This thermocouple temperature was then monitored until it "plateaued" and provided the measurement of the ambient temperature within the sound chamber.

This completed a single experimental trial for a specified resonant frequency, pressure ratio and temperature. The procedure was carried out a total of nine times for each different pressure ratio. Once the six outputs for each run were recorded, they were then transferred over to a spreadsheet where all other significant parameters were computed automatically. The following section delineates the various calculations performed in the spreadsheet.

E. EXPERIMENTAL CALCULATIONS

The first calculation desired from the spreadsheet entails finding the actual power being supplied to the cartridge heater. This is expressed in terms of the current and voltage outputs from each experimental run.

$$P = IV \tag{36}$$

Once the power to the cartridge heater was known, the surface temperature of the test cylinder itself could be calculated. Knowing the resistance of the thermal circuit as given in Appendix B, the difference between the interior temperature and the surface temperature is simply

$$\Delta T = PR_{eq} \tag{37}$$

Utilizing the interior temperature output as provided by the thermocouple embedded in the cartridge heater, the surface temperature of the cylinder is then expressed as

$$T_s = T_c - PR_{eq} \tag{38}$$

Once the surface temperature is known, the difference between it and the ambient temperature, as measured during the experiment, is calculated. The result is then combined with the power calculated in Eq. 35, as well as with the external surface area of the test cylinder, to find the convective heat transfer coefficient

$$h = \frac{P}{A(T_s - T_a)} \tag{39}$$

The Nusselt number is then derived from the following equation

$$Nu = \frac{hd}{k} \tag{40}$$

where d is the test cylinder diameter and k is the thermal conductivity of air.

The next step is to calculate the various criteria as previously listed in the theory section. In order to find the length scale ratio, χ , the speed of sound within the chamber must first be calculated using the ambient temperature

$$c = \sqrt{\gamma R(T_a + 273.15)} \tag{41}$$

The value of χ can now be found from Eqs. 1 - 4. The amplitude parameter, ϵ , is derived in the spreadsheet by combining the pressure ratio from Eq. 33 along with Eq. 8. The third parameter calculated is the frequency parameter, Λ^2 , from Eqs. 10 and 11.

The streaming Reynolds number, R_s , is found by using Eq. 16. This, however, does not represent the true value at the test cylinder position due to its not being precisely at the velocity antinode. A corrected value for R_s can be deduced by finding the particle velocity offset between the velocity antinode location and the test cylinder position. This offset is derived by knowing the frequency of the sinusoidal signal being generated, the speed of sound within the chamber from Eq. 40, and the distance from the test cylinder to the end plate termination face, L. Hence

$$U_{0_{corrected}} = U_0 |\sin(\frac{4L}{c/f} \frac{\pi}{2})| \tag{42}$$

or

$$U_{0_{corrected}} = U_0 | \sin \left(\frac{2\pi L}{\lambda} \right) |$$
 (43)

Since R_s is proportional to U_0^2 , it then follows that

$$R_{s_{corrected}} = R_s |\sin(\frac{2\pi L}{\lambda})|^2$$
 (44)

from which it was found that the corrected value for R_s had less than a 0.1% error due to the slightly displaced location of the cylinder.

IV. RESULTS AND DISCUSSION

Nearly 600 experimental trials were performed throughout the course of this study which produced results for 183 distinct data points. Data were obtained for five different frequencies at various pressure ratios ranging from 0.9 % to 3.2 %. Three separate series of trials were performed at each acoustic signal setting, (i.e., at each frequency and pressure ratio setting) corresponding to three separate settings for the driving temperature difference between the test cylinder surface and ambient conditions. The resultant values of the streaming Reynolds number ranged from 40 to 1070 while the corresponding Nusselt numbers obtained varied from 8 to 38. Figure 14 is a parameter map as suggested by Richardson (1967) which shows the range of values for the experimental data covered plotted as a function of the amplitude parameter versus the frequency parameter, and delineates the expected different regimes of flow. The data obtained cover a very narrow regime of this parameter map as was intentionally planned for this experiment. Now it can be seen that the heat transfer results obtained through experimentation are quite evenly distributed between two distinct regions on the map. Region A represents the regime in which the flow is expected to remain laminar, incompressible and attached, and outer acoustic streaming is the main heat transport mechanism. This region is well understood in theory but has yet to be thoroughly verified through experimentation. It is anticipated that the heat transport mechanism for the data in region B will be presented as a combination of effects, including that of vortex shedding, as predicted by Honji (1981), Hall (1984) and Sarpkaya (1986).

The heat transfer results are presented as plots of Nu vs. R_s (Figure 15). Here the difference between the two regions of varying heat transport mechanisms becomes more apparent. A break clearly occurs in the region where $R_s \sim 240 \ (\pm 20)$ and it can be concluded that there is some critical point in this range where there is a transition in the flow at which vortex shedding begins to become a dominant factor in the heat transport away from the test cylinder. In order to better examine the differences between these two regimes, the data is divided into their respective groups and individually analyzed.

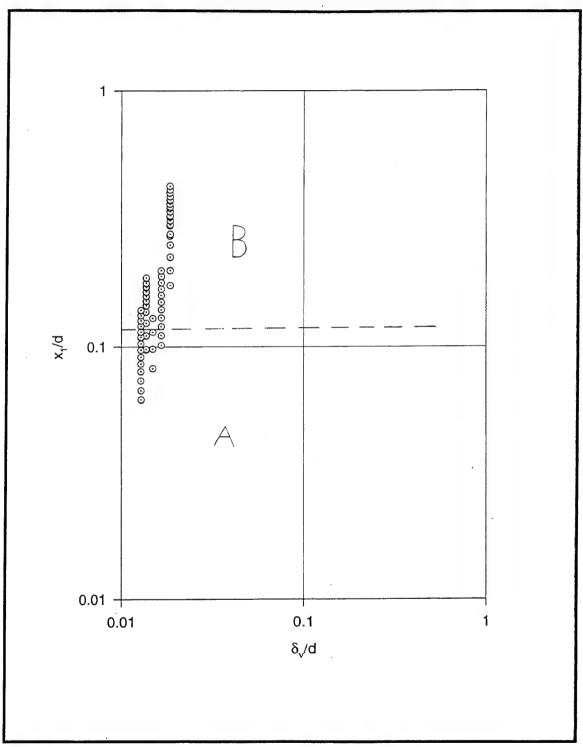


Figure 14. Parameter Map of Expected Heat Transfer Regimes as presented by Richardson (1963): Convection by Inner Acoustic Streaming (A), by Outer Acoustic Streaming (B).

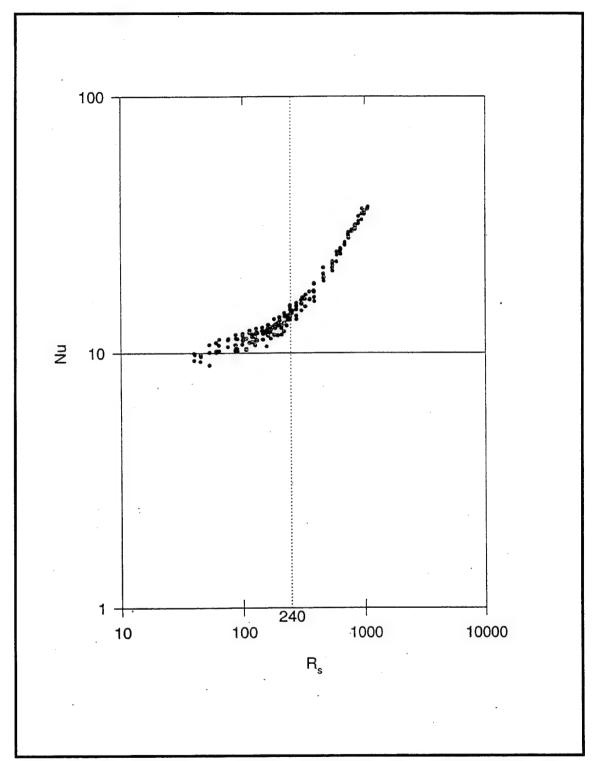


Figure 15. All Data Plotted as a Function of Nusselt Number vs. Streaming Reynolds Number.

A. LAMINAR, ATTACHED FLOW REGIME

Figure 16 is a plot of the heat transfer results in terms of the Nusselt number versus the streaming Reynolds number for the data in which $R_s < 240$, and for which criteria A through C (Eqs. 4, 5, 15) have been met. Those data points which do not meet these criteria have been disregarded. The critical parameters for the remaining data are as follows:

- $\chi < 0.1$
- $\epsilon < 0.3$
- $\qquad \Lambda^2 > 1800$
- $-\frac{R_s}{\Lambda} < 4.5$

Theory clearly indicates that the dependency of the Nusselt number on the streaming Reynolds number in this regime is of the form $\mathrm{Nu} = \mathrm{xPr}^{\mathrm{y}}\mathrm{R_s}^{0.5}$. Since the Prandtl number remains constant throughout the experiment, the solution for this dependency be can further simplified as $\mathrm{Nu} = \mathrm{CR_s}^{0.5}$, where the term "C" encompasses both the Prandtl number and the qualitative constant of the previous equation. However, this solution form is only valid for "large" values of $\mathrm{R_s}$. Since the theory does not provide a definite limit for what qualifies as "large", this criterion had to be determined from a careful examination of the experimental data. From Figure 16, it can be observed that there is indeed a break point at $\mathrm{R_s} \sim 130$ where the results diverge into two separate solutions. It was found that the square-root dependency on $\mathrm{R_s}$ does not significantly change past this value, and it is therefore suggested that this may be in the range of the lower limit of "large" values of $\mathrm{R_s}$. For those values in which $\mathrm{R_s} > 130$, a curve fit of the heat transfer characteristics results in a solution of the form

$$Nu = 0.94R_s^{0.5} (45)$$

and it can therefore be determined that the range of values $130 < R_s < 240$ is representative of "large" values of the streaming Reynolds number. The values below $R_s = 130$ are excluded as not being large enough due to a variety of reasons as described later.

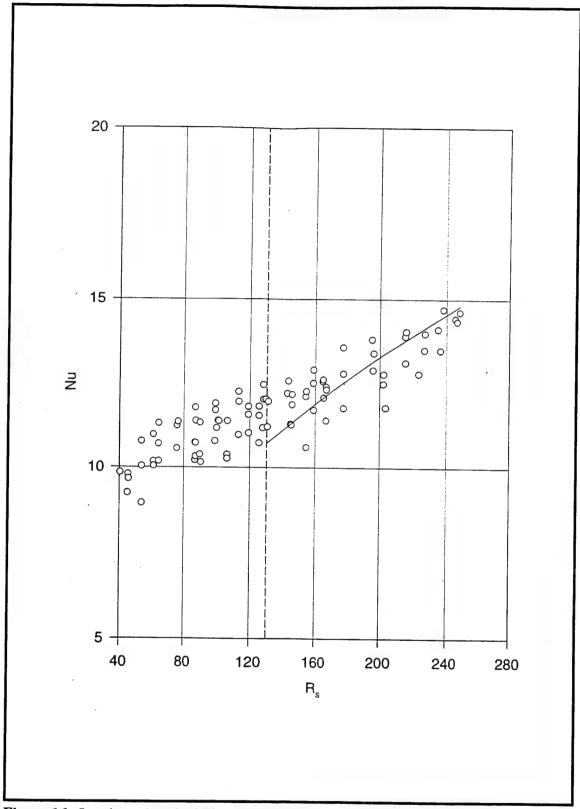


Figure 16. Laminar, Attached Flow Regime for $R_s < 240$ (Curve Fit Shown).

Davidson (1973), in an extension of the work by Richardson, analytically and numerically tackled the problem of heat transfer from a cylinder in a strong acoustic field in great detail. He obtained a correlation of the dependency of the Nusselt number on both the Prandtl and streaming Reynolds numbers. The correlation, as extracted from the work of Davidson by Gopinath and Mills (1993) for this regime, is of the form

$$Nu = 1.388 Pr^{0.73} R_s^{0.5} (46)$$

By taking Pr = 0.7 for air for these experiments, the equation then becomes

$$Nu = 1.07R_s^{0.5} (47)$$

The experimental fit in Eq. 45 under predicts by about 13%, but supports this correlation well within the limits of uncertainty.

Figure 17 is a plot of the region in which Eq. 45 is valid, and includes the experimental uncertainty of each data point as derived in Appendix C. It can be observed from this plot that the deviation from the curve fit in this range of values for R_s is well within experimental uncertainty limits.

Although the lower end of the range for "large" R_s in which the predicted solution is valid has been determined to be 130 for the results obtained from these experiments, it is by no means an absolute boundary. Even though the resultant heat transport characteristics deviate significantly below that point in the region where "intermediate" values of R_s are present, there are several factors which could account for part of the discrepancy, especially in the region around $100 < R_s < 130$. These include uncertainty due to equipment limitations and the effects of natural convection and conduction of heat away from the test cylinder.

The effect of natural convection on most of the intermediate values of \mathbf{R}_s is negligible, though, as characterized by the very low ratio of the Grashof number to the square of the streaming Reynolds number (except at very low values of \mathbf{R}_s). The effect due to conduction is much harder to quantify in so simple a form, though. Conduction can and does occur during the experiment at two separate places where the test cylinder is in contact with

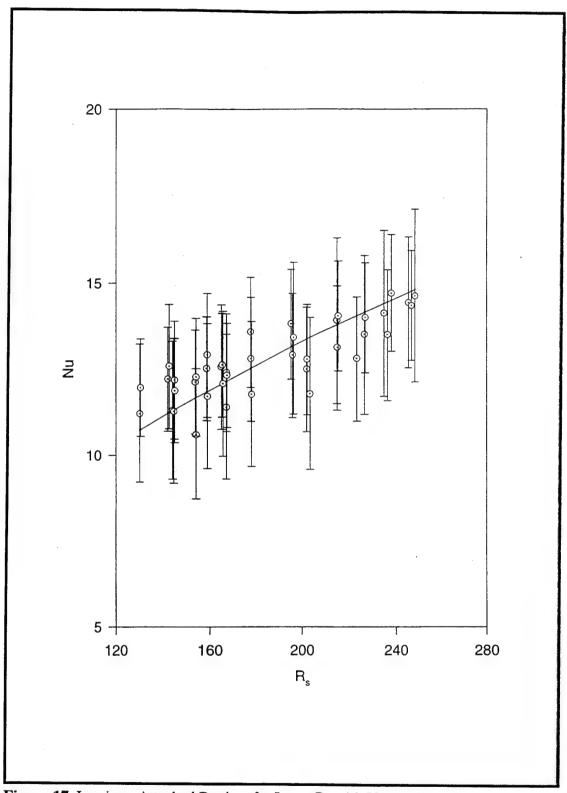


Figure 17. Laminar, Attached Regime for Large R_s with Uncertainty Bars.

other apparatus. The top of the test cylinder is in contact with the "plug" used to hold it in place, while the bottom of the test cylinder is allowed to rest in a small depression on the wall of the plexiglass sound chamber. Realizing, though, that the contact area at both points is very small, and that the thermal resistance of the plug and the plexiglass wall are both relatively high (due to their low thermal conductivity), the heat transport away from the test cylinder can be assumed to be negligible as well.

The uncertainty in the results due to the equipment limitations, though, does have a very significant impact on the results. The calculated value of the Nusselt number has an uncertainty of up to 20% (see Appendix C) depending upon the power being dissipated by Since the experiment revolves around finding the heat transfer the test cylinder. characteristics at specific values of temperature difference between the test cylinder surface and the ambient conditions, correspondingly low power dissipation from the cylinder occurs as the streaming Reynolds number decreases. Therefore, the region of intermediate values of R, have relatively low electrical heat dissipation, and hence low current values associated with them, dropping to as low as 0.06 amps in some cases. Since the equipment uncertainty for the current reading is of the order of the last digit present, this particular component of the Nusselt number has an uncertainty of nearly 18% by itself and greatly influences the overall uncertainty. Therefore, it may be more accurate to define the lower limit of large R_s values as somewhere between 100 and 130. However, the error due to equipment measurements is not enough to compensate for the disparity between theory and experiment at values much less than 100.

B. SEPARATED FLOW REGIME

The second regime which this experiment encompasses is that in which vortex shedding and other forms of unsteady flow begin to affect the heat transport characteristics. Figure 18 is a plot of the resultant data in this regime as obtained during experimentation in terms of the Nusselt number versus the streaming Reynolds number. A curve fit of the data results in a solution of the form

$$Nu = 0.31R_s^{0.69} (48)$$

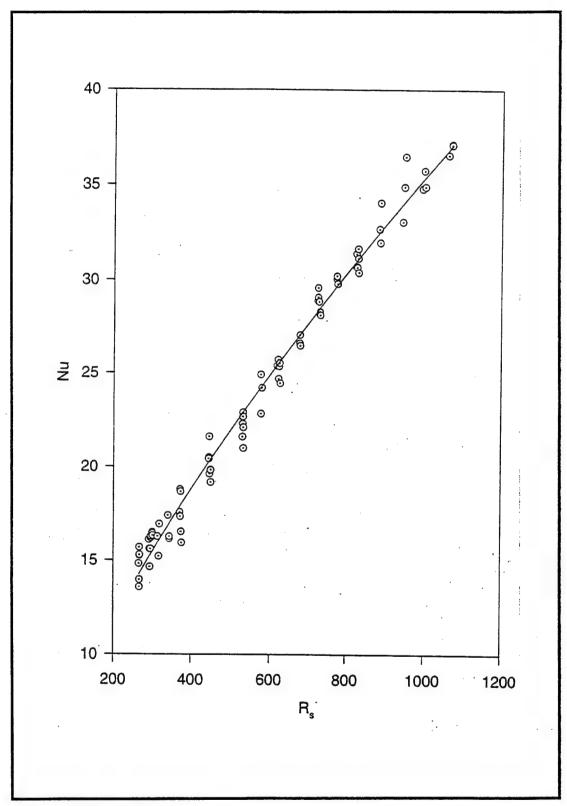


Figure 18. Unstable Regime.

There is no theory for this flow regime with which to compare the results, but it is reasonable to assume that this curve fit is representative of the correct solution. As expected, the unstable flow resulting at higher values of R_s would increase the heat transfer rate from the cylinder, and hence the stronger dependency that the Nusselt number has on the streaming Reynolds number, as opposed to the usual square root dependence.

V. CONCLUSIONS

Experiments were conducted to observe the convective heat transfer rates to an isolated cylinder in an acoustic standing wave. A comparison of the various length scales and other parameters was conducted, and the experimental method stated. During the experiment, the properties of the acoustic field were varied to provide a large base of data which was then analyzed and discussed. Several regimes of interest were investigated and the results presented. Figure 19 is a plot of all data obtained with curve fits for the regimes of interest.

Essentially, heat transfer from a cylinder in a zero-mean oscillatory flow as represented by an acoustic standing wave can be divided into at four separate regimes in which different heat transport mechanisms dominate. For very low values of the streaming Reynolds number, R_s, convective effects due to the acoustic field are negligible and natural convection is then the dominant mode of heat transport. For intermediate values of R_s, there is a stronger dependence on R_s but not yet on the order of R_s 0.5 since R_s is still not large enough for flow to be of the boundary layer type. Buoyancy effects are comparable in the lower end of this regime, becoming small for larger values of R_s. For much larger values of R_s, past 100 or so, an acoustic streaming flow presents itself in the boundary layer, resulting in a square-root dependency on R_s. Experimentally, the results obtained in this regime closely match the expected theory, and the heat transfer characteristics may be estimated by Eq. 45. Finally, past a critical value of R_s ~240 (which confirms well with theory), an unstable flow with vortex shedding begins to take place at the surface of the cylinder, increasing the dependency on R_s which the overall heat transfer solution has. The heat transfer characteristics in this regime may be estimated from Eq. 48. It is these last two of the above regimes that formed the focus of this study.

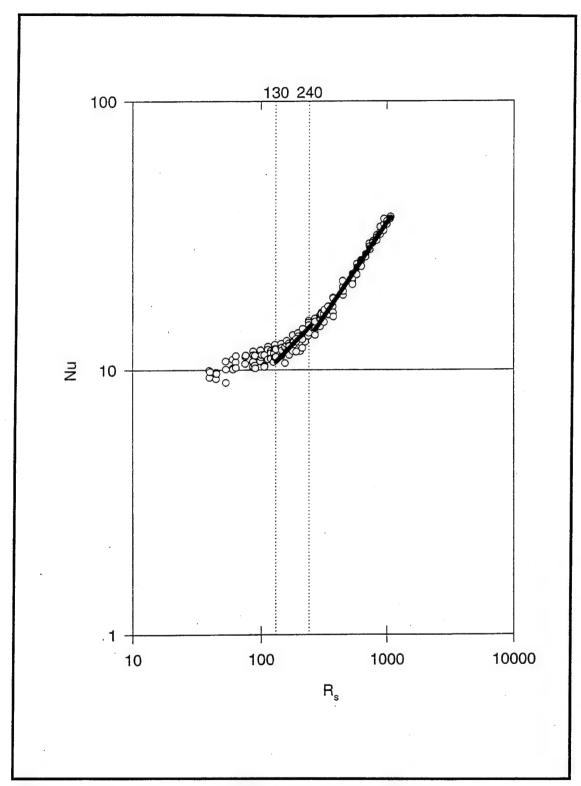


Figure 19. Data Plotted with Curve Fits for the Regimes of Interest.

VI. RECOMMENDATIONS

Several directions for further study may be suggested by the data obtained from this work. One factor which may need further investigation is a study of the effects of varying the aspect ratio of the chamber diameter to the cylinder length to determine whether it was large enough for these experiments. A ratio of 15 was used for this experiment, and was assumed to be large enough to discount flow effects caused by the walls of the chamber, but this number was chosen somewhat arbitrarily and only a detailed experimental study can determine the actual effect.

Another possible source of error which cannot be accurately accounted for concerns itself with the experimental method used. Specifically, the measurement of the ambient temperature within the chamber at the time of each experimental trial. The method used, that of removing the test cylinder from the sound chamber after its temperature has been recorded and the acoustic signal has ceased, then placing a thermocouple through the hole it has just evacuated in the chamber wall, can be improved upon. During the time required while waiting for the thermocouple temperature to peak, the heated air in the sound chamber is rising to the top of the chamber and the accuracy of correlating the thermocouple temperature reading to that of the ambient temperature at the time of the experimental trial is left in doubt. A better method would be to permanently affix the thermocouple in the chamber so that simultaneous measurement of both the cylinder temperature and the ambient temperature can be taken.

Additional research into three different areas of the problem come to mind. First, a test of the effects of placing the test cylinder horizontally in the sound chamber is suggested, although there should be little if any dependency on this orientation since natural convection effects are small for the strong acoustic fields being used. Completely new geometries may also be tested which would mimic actual heat exchanger component shapes expected in a thermoacoustic engine. Finally, additional research using different gases in the sound chamber would provide data on the dependency of the Nusselt number on the Prandtl number.

APPENDIX A. CALIBRATIONS AND CALCULATIONS

Several pieces of the experimental apparatus required some form of calibration, or calculation of a specific parameter, prior to initiating the experiments. The most important equipment items of concern were the "unattached" J-type thermocouples which were to be used for various applications throughout the experiment, as well as the thermocouple which was embedded in the cartridge heater. In addition, an equivalent thermal resistance for the cartridge heater/silicon oil/copper sheath circuit needed to be calculated along with assurances that the linear temperature distribution along the test cylinder was within reasonable limits. An additional study was performed to analyze the heat transfer effects on the test cylinder due only to natural convection.

Three J-type thermocouples were used throughout the experiment and were the first items to be calibrated. Since accurate temperature information was crucial to the reliability of the data obtained through experimentation, the thermocouples were tested to see if any of them showed a tendency to read either higher or lower than the actual temperature (a somewhat common occurrence). The embedded thermocouple in the cartridge heater was also tested for the same reason. All thermocouple leads were attached to the thermocouple reader to be used throughout the experiment to ensure that the entire circuit was tested concurrently.

The unattached thermocouples and the cartridge heater were all placed in an ethyl glycol solution belonging to a Rosemont Model 913A calibration bath. A Rosemont Model 920A commutating bridge, which utilizes a precision temperature probe as its input, provides the reference temperature. The temperature of the bath was then set at varying points between 22° and 48°C, the expected experimental temperatures being well within that range. Table 2 shows the results of the calibration. All thermocouples were found to read within 0.1°C of the reference temperature for all cases. This was well within the possible uncertainty of 0.5°C listed for J-type thermocouples.

Next, Figure 20 shows the thermocouple arrangement which was used to experimentally derive the equivalent thermal resistance and the linear temperature

Reference Temperature (C)	Cartridge Heater (C)	Thermocouple "A" (C)	Thermocouple "B" (C)	Thermocouple "C" (C)	Maximum Deviation (C)
22.83	22.8	22.8	22.8	22.8	< 0.1
23.96	23.9	23.9	23.9	23.9	< 0.1
26.83	26.8	26.8	26.8	26.8	< 0.1
29.63	29.6	29.6	29.6	29.6	< 0.1
32.49	32.5	32.5	32.5	32.5	< 0.1
35.37	35.4	35.4	35.4	35.4	< 0.1
38.04	38.1	38.0	38.0	38.0	< 0.1
40.74	40.8	40.8	40.8	40.8	< 0.1
43.02	43.0	43.0	43.0	43.0	< 0.1
45.68	45.7	45.7	45.7	45.7	< 0.1
48.19	48.2	48.2	48.2	48.2	< 0.1

Table 2: Thermocouple Calibration Data (all values in °C)

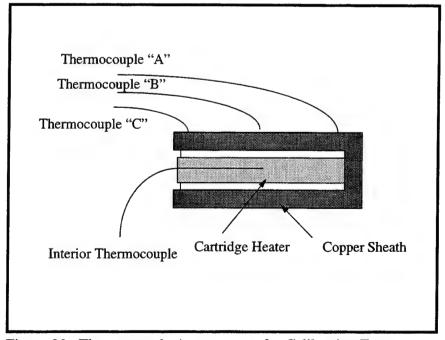


Figure 20. Thermocouple Arrangement for Calibration Tests.

distribution data. The three unattached thermocouples which were previously calibrated were securely placed on the outer surface of the test cylinder using clamps, the test cylinder being already prepared as it normally would for an experimental run. The test cylinder was then suspended horizontally and power was supplied to the cartridge heater. Once all four of the thermocouples reached a steady-state temperature, the temperatures were recorded along with the voltage and current being supplied to the heater. The equivalent thermal resistance was then derived using the equation

$$R_{eq} = \frac{T_c - T_s}{IV} \tag{A.1}$$

Temperatures which were to be comparable with those used during the experiments were obtained in a still air environment. Some additional tests were performed using a fan blowing air across the test cylinder to create a higher heat transfer rate away from the cylinder so that higher power levels to the heater could be reached while staying within the limits of the expected temperature range. An average temperature was then obtained using all three thermocouples, and this value was in turn utilized to derive an equivalent thermal resistance for each test run. The final value of the thermal resistance resulted in averaging the values from each test run, obtaining 1.022 K/W with a maximum deviation of 0.089 K/W. The results are annotated in Table 3.

The same test results were used to determine the change in temperature along the length of the test cylinder. The difference in temperature along the length at the lower power settings demonstrated a fairly even distribution of heat, with a variation of less than 0.6° C. This variation increased as the power to the heater was also increased, as could be expected. For the temperature range in which the experiments were run (again, 26° to 40° C), a maximum variation of 1.6° C occurred, although this was at a very high level of power being supplied to the heater and was not representative of the remaining data.

Current (Amps)	Voltage (Volts)	Power (Watts)	Interior Temp (C)	Thermo- couple "A"	Thermo- couple "B" (C)	Thermo- couple "C"	Average Surf Temp (C)	Equiv Resistance (C)
0.08	8.30	0.664	38.4	37.8	38.1	37.3	37.73	1.009
0.07	7.40	0.518	36.0	35.5	35.5	35.4	35.47	1.023
0.08	8.40	0.672	37.3	36.9	36.8	36.3	36.67	0.938
0.09	9.50	0.855	40.8	40.2	40.1	39.5	39.93	1.018
0.09	10.30	0.927	42.9	42.3	42.0	41.5	41.93	1.046
0.10	10.70	1.070	44.8	44.2	43.7	43.2	43.70	1.028
0.08	9.00	0.720	27.4	26.9	26.6	26.3	26.60	- 1.111
0.09	10.00	0.900	28.6	27.9	27.8	27.2	27.63	1.078
0.10	11.00	1.100	30.1	29.3	29.2	28.6	29.03	0.973
0.11	12.00	1.320	31.5	30.5	30.5	29.6	30.20	0.985
0.14	15.50	2.170	37.6	35.7	36.0	34.4	35.37	1.028

Table 3: Thermal Resistance and Linear Temperature Distribution Trials

APPENDIX B. UNCERTAINTY ERROR ANALYSIS

In order to obtain a measure of the reliability of the data obtained during experimentation, an uncertainty error analysis was performed for both the Nusselt number and the streaming Reynolds number. The analysis consisted of finding the maximum possible deviation for all components appearing in the equations which define both parameters. Then, in a standard fashion, a root mean square analysis was performed to derive a reasonable overall possible error for each experimental run. The error analysis formulae were themselves incorporated into the "results worksheet" provided in Appendix C so that each data point has an associated possible error derived from the input provided.

The Nusselt number can be derived as shown from Eqs. 37 through 40

$$Nu = \frac{IV}{\pi k l (T_c - IVR_{eq} - T_q)}$$
 (B.1)

where l is the length of the test cylinder.

Using the manufacturers' recommended equipment error ranges, the maximum uncertainty in each measured component in Eq. B.1 is as follows

$$V = 0.05\% \text{ reading} + 0.02\% \text{ full scale} + 1 \text{ digit}$$
 (B.2)

$$I = 0.5\% RDG + 1 digit$$
 (B.3)

$$T_c = \pm 0.5^{\circ} C$$
 (B.4)

$$T_a = \pm 0.5^o C$$
 (B.5)

The maximum uncertainty in the equivalent resistance is obtained from the calibration data in Appendix A as

$$R_{eq} = \pm 0.089 \ \text{K/W}$$
 (B.6)

The analysis for the overall uncertainty itself is structured as follows. For the Nusselt number, the root mean square error is given by

$$\triangle(Nu) = \left[\sum_{i} \left(\frac{\partial Nu}{\partial X_{i}} \triangle X_{i}\right)^{2}\right]^{1/2}$$
(B.7)

where X_i represents each individual component of Eq. B.1. The $\frac{\partial Nu}{\partial X_i}$ term, as the partial derivative indicates, physically represents the sensitivity of the Nusselt number to the variable X_i , provided all other variables are unchanged. The ΔX_i represents the uncertainty in the corresponding variable as given in Eqs. B.2 to B.5. For instance, the contribution to the uncertainty due to the voltage measurement is

$$(\triangle Nu)_{V} = \frac{\partial Nu}{\partial V} \triangle V = Nu \frac{(T_{c} - T_{a})(\triangle V)}{V(T_{c} - VIR_{ea} - T_{a})}$$
(B.8)

A value with more significance, though, is the individual fractional uncertainty which takes into account the calculated value of the Nusselt number and can be expressed as a percentage possible error. This is represented by dividing the individual uncertainty by the Nusselt number in the following manner

individual fractional uncertainty =
$$\frac{(\triangle Nu)_V}{Nu} = \frac{(T_c - T_a)(\triangle V)}{V(T_c - VIR_{eq} - T_a)}$$
 (B.9)

This new term leads directly to the desired method of expressing the possible error in the calculated value of the Nusselt number as an overall fractional uncertainty using a root mean square analysis.

$$\frac{\triangle(Nu)}{Nu} = \left[\sum_{i} \left(\frac{\triangle(Nu)_{i}}{Nu}\right)^{2}\right]^{1/2}$$
 (B.10)

In a similar fashion, the individual fractional uncertainty of the remaining terms are

as follows

$$\frac{\left(\triangle Nu\right)_{I}}{Nu} = \frac{\left(T_{c} - T_{a}\right)\left(\triangle I\right)}{I\left(T_{c} - VIR_{eq} - T_{a}\right)}$$
(B.11)

$$\frac{\left(\triangle Nu\right)_{T_c}}{Nu} = \frac{\left(\triangle T_c\right)}{\left(T_c - VIR_{eq} - T_a\right)}$$
(B.12)

$$\frac{\left(\triangle Nu\right)_{T_a}}{Nu} = \frac{\left(\triangle T_a\right)}{\left(T_c - VIR_{eq} - T_a\right)}$$
(B.13)

$$\frac{\left(\triangle Nu\right)_{R_{eq}}}{Nu} = \frac{VI(\triangle R_{eq})}{\left(T_c - VIR_{eq} - T_a\right)}$$
(B.14)

By examining the data results, the largest contributor to the overall error in the Nusselt number is due to the current. This is caused by the small currents being utilized during the experiment, which were as low as 0.05 amps. Since the uncertainty in the current is of the order of 0.01 amps, there can be an error in the calculated Nusselt number of approximately 20% due to the current term alone. One way to lessen the effect that the current term has on the overall error is by decreasing the voltage output of the power supply and hence increasing the current needed to maintain the same power being generated. This capability, though, is not a feature of the equipment being used. It must also be noted that although $\Delta T_{c,a} = \pm 0.5^{\circ}$ C in Eqs. B.4 and B.5, the calibration of the thermocouples described in Appendix A indicated an error of less than 0.1°C for the temperature range of the experiment. If this is taken into account, the error due to the ambient temperature (T_a) and the center temperature (T_c) terms in Eq. B.1 would diminish by 80%.

The error analysis for the streaming Reynolds number is similar to that just performed for the Nusselt number. Utilizing Eq. 17 and substituting Eqs. 33 and 41 into it, the value of R_s can be shown to be

$$R_s = \frac{R(T_a + 273.15)(V_0/\sqrt{2})^2}{2\pi f v \gamma P_m^2 (0.523)^2}$$
(B.15)

Recall that $V_0/\sqrt{2}$ is the multimeter output voltage as derived from the pressure transducer

after being passed through the 100 gain preamplifier. Thus, R_s becomes a function of only the following measured variables: the frequency, the ambient temperature and the pressure transducer output as read on the multimeter, while all additional parameters remain constant. The maximum uncertainty for the thermocouple is the same as previously listed ($\pm 0.5^{\circ}$ C) while the other two variables have uncertainties provided in the manufacturers' specifications. Calibration data for the pressure transducer used during the experiment indicates an uncertainty equivalent to 0.15% of the percentage of Full Scale Output (FSO) where the FSO is 254 mV. This correlates to a maximum deviation of 2.3 mV after passing through the 100 gain preamplifier. In addition to this, the multimeter which is used to read this voltage has an error of 0.06% reading + 0.03% Range. These combine to give a total microphone voltage output error of

$$Mic = 2.3 \ mV + 0.06\% \ reading + 0.03\% \ range$$
 (B.16)

The uncertainty in the frequency signal from the function generator error is given as 20 ppm, i.e.,

$$f = 20x10^{-6} reading (B.17)$$

Again a root mean square analysis was performed in a manner similar to Eq. B.7 to derive the overall fractional uncertainty for the streaming Reynolds number.

$$\frac{\Delta(R_s)}{R_s} = \left[\sum_{i} \left(\frac{\Delta(R_s)_i}{R_s} \right)^2 \right]^{1/2}$$
 (B.18)

The individual fractional uncertainties are

$$\frac{(\triangle R_s)_{T_a}}{R_s} = \frac{\triangle T_a}{(T_a + 273.15)}$$
 (B.19)

$$\frac{(\triangle R_s)_{Mic}}{R_s} = \frac{2(\triangle Mic)}{V_0/\sqrt{2}}$$
 (B.20)

$$\frac{(\triangle R_s)_f}{R_s} = \frac{\triangle f}{f} \tag{B.21}$$

The largest contribution to the error in the streaming Reynolds number is the error due to the microphone output voltage, specifically, the possible error in the pressure transducer itself. However, the data presented in Appendix C shows that this error is very small and in the range of <2% of the total value.

APPENDIX C. EXPERIMENTAL DATA

The following pages contain the data obtained through experimentation in spreadsheet fashion. All of the relevant parameters are listed, although some constants have been left out due to size constraints.

SPL (dB)	153.5	!	1		153.5	-			_1	153.5	-	-	155.7	1_	1		155.7	155.7	1	:	155.7		155.7		157.5	157.4	157		157	1 -	157		157.5	157.	157.5
PR %	1.3244	1.3263	1.3263		1.3263	1 3244	1 3263		1.3263	1.3263	1.3244		1.7092	1.7074	1.7074		1.7017	1.7036	1.7036		1.7055	1.7055	1.7074		2.0828	2.0733	2.0884	-	2.0866	2.0884	2.0884		2.0809	2.0884	2.0884
Rs A	4.692	4.704	4.705	4.7	4.705	4 694	4 70B	4 702	4.708	4.709	4.696	4.704	7.82	7.803	7.803	7.808	7.752	7.773	7.774	7.767	7.79	7.791	7.809	7.797	11.6	1.5	11.67	11,59	11.65	11.68	11,68	11.67	11,59	11.68	11.68
Яs*Re	0.005	0.005	900.0	0.005	0.009	0.009	0.009	0.009	0.012	0.013	0.012	0.012	0.002	0.002	0.002	0.002	0.003	0.003	0.003	0.003	0.004	0.004	0.004	0.004	9E-04	0.001	1E-03	1E:03	0.001	0.001	0.001	0.001	0.002		
β (2ΔΔ/π)	944.6	944.6	944.6	944.6	944.6	946.2	946.2	945.7	946.2	946.2	946.2	946.2	946.2	946.2	946.2	946.2	946.2	947.9	947.9	947.3	947.9	947.9	947.9	947.9	944.6	944.6	946.2	945.2	946.2	946.2	946.2	946.2	946.2	947.9	947.9
V#V	1484	1484	1484	1484	1484	1486	1486	1485	1486	1486	1486	1486	1486	1486	1486	1486	1486	1489	1489	1488	1489	1489	1489	1489	1484	1484	1486	1485	1486	1486	1486	1486	1486	1489	1489
KC (TE)	1.104	1.105	1.105	1.105	1.105	1.102	1.104	1.104	1.104	1,105	1.104	1.104	1.423	1.422	1.422	1.423	1.418	1.418	1.418	1,418	1.419	1.42	1.422	1.42	1.735	1.728	1.739	1,734	1.738	1.74	1.74	1,739	1.735	1.738	1.739
ů.	0.351	0.352	0.352	0.352	0.352	0.351	0.351	0.351	0.352	0.352	0.351	0.352	0.453	0.453	0.453	0.453	0.451	0.451	0.452	0.451	0.452	0.452	0.453	0.452	0.552	0.55	0.553	0.552	0.553	0.554	0.554	0.554	0.552	0.553	0.554
×	0.027	0.027	0.027	0,027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027			0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	_	0.027	_
ã.	12.2	13		12.5	13	13	12.7	-20	12.5	13.3	_	12.8	17.5	15.9	_	16.3				16.5		-		16.4	;	19.2	_	19.6	19.3	20	20.5	19.8			20
⊽	1 181	9 181		8 181	8 181	3 181		1 181	4 182	1 182				301		7 301					- 1	- 1			i	į			7 449			6 450	į		4 451
- S	- :			63.98	4 66.38	3 66.73	4 65.19	66,1	-									اس		3333 <u> </u>				100	-:				5 98.77			101.5			
			03 184			02 183			33 184	i	02 184		i		905 305			!	33 304			1	905 305			99 449	recensio		106 455	07 456	7 456			77 456	
			582 0,703		582 0,703	583 0,702	583 0,703		583 0,703	10.00	583 0.702	- N	0	0	583 0,90		Ó	4	34 0,903		584 0,90	ď	584 0,9(660'L 289	33 1.107		_	-	Ξ		13 1,103		1,107
		7.13 51			9.53 51					1.82 5			9,36 51				10,75 5	S Y	.86 58.			3,15 5			9,46 56	7	2			1,86 583			×	3.89 584	
		0.07			0.09					0.11 11			6 60'0				0,1 10					0.12 13			0.09	2				0,11 11		- 1		0.12 13	
œ.		6.19		3320	9	27.44	11.1	-	2	<u> </u>	er was p		*	•		-	4	7			N.	15.1	www.		6.83			-	-	2	man,	2266	S. 250	ල (22.000
9 <u>0</u>	28.6	28.9	29.8		33.7	33.6	34.5		38	39.5	38.8		31.3	30.8	31.1		34.4	32	35.1	3	38.4	39.6	39.7	1	29.7	30.7	30.6		34.8	34.4	34.2		38.2	38.6	39.4
military.	29	7 29,4	9 30.3		1 34.6	2,33	4 35.4		5 39.1	8 40,8	1 40		6 32.2	7 31.5	7 31,8		4 35.5	3 36,1	5 36.2		39.8	5 41,2	41.3		9 0 0 7	מינס סינס סינס	4 31.5			3.35.7			39.0	60.3 60.3	ا ئ
	23.	22,7	22.	2	23.1	23	23,		23.	23.8	24,		23.6	233	23.	ľ	24	24	24.		44	24.5	24.	100	A (::::::::::::::::::::::::::::::::::::::	or suspec		23.	23.9	23,6		24	24.3	7.47
Information	~ 583	_ = 73 cm	PH ~ 1.3			d challenge of the state of the							~ 583	= /3 cm	PR ~ 1.7									001	~ 583	100	7H ~ Z.1							1	***************************************

liejikalitäjejiti)	Tine Ort	uncan	uncent uncen	uncen	uncert	umcent	uncen	uncen	uncert	negun
f ~ 583	0.2123	18.493	9.2866	9.2866	0.6972	22.693	0.1687	0.9402	0.005	0.9552
73	0.2059	16.001	8.0759	8.0759		19.674	0.169	0.9388	0.005	0.9539
PR ~ 1.3	0.1968	15.947	7.2873	7.2873	0.7086	19.001	0.1689	0.9388	0.002	0.9539
						20.456				0.9543
	0.1677	12.567	4.7056	4.7056	0.7426	14.24	0.1688	0.9388	0.002	0.9539
	0.1692	12.572	4.7908	4.7908	0.7465	14.302	0.1687	0.9402	0.002	0.9552
	0.1646	12.549	4.5034	4.5034	0.7294	14.093	0.1686	0.9388	0.005	0.9539
						14.272				0.9543
	0.15	11.332	3.4592	3.4592	0.7154	12.365	0.1685	0.9388	0.002	0.9538
	0.146		3.1899	3.1899	0.7631	11.365	0.1684	0.9388	0.002	0.9538
	0.1478	11.337	3.3952	3.3952	0.719	12.333	0.1682	0.9402	0.002	0.9551
						12.021				0.9543
~ 583	0.1742	12.899	6.4589	6.4589	1.0011	15.838	0.1685	0.7285	0.002	0.7477
L = 73 cm	0.1806	14.312		7.0574	0.9111	17.473	0.1684	0.7293	0.002	0.7485
PR ~ 1.7	0.1779	14.286	6.7834	6.7834	0.8927	17.232	0.1684	0.7293	0.002	0.7485
						16.848				0.7482
	0.1581	11.606	4.8058	4.8058	0.9506	13.484	0.1683	0.7317	0.002	0.7508
	0.1562	11.595	4.6791	4.6791	0.941	13.384	0.1681	0.7309	0.002	0.75
	0.1569	11.597	4.7201	4.7201	0.9432	13.415	0.168	0.7309	0.002	0.75
						13.428				0.7502
	0.1417	10.554	3.5271	3.5271	0.9066	11.71	0.1682	0.7301	0.002	0.7492
	0.1395	9.7749	3.3131	3.3131	0.962	10.883	0.168	0.7301	0.005	0.7492
	0.1404	9.7692	3.3312	3.3312	0.9562	10.889	0.1679	0.7293	0.002	0.7484
						11.161				0.7489
~ 583	0.1755	13.086	7.3184	7.3184	1.1465	16.724	0.1689	0.5978	0.005	0.6212
_ = 73 cm	0.1691	13.022	6.6756	6.6756	1.0966	16.122	0.1687	0.6005	0.005	0.6238
PR ~ 2.1	0.171	13.061	6.9436	6.9436	1.1269	16.38	0.1686	0.5962	0.005	0.6196
						16.409				0.6215
	0.1484	10.765	4.4897	4.4897	1.105	12.548	0.1684	0.5967	0.002	0.6201
	0.1514	10.809	4.7755	4.7755	1.1463	12.798	0.1683	0.5962	0.005	0.6195
	0.1525	10.817	4.8616	4.8616	1.1542	12.87	0.1683	0.5962	0.005	0.6195
						12.739				0.6197
	0.1379	9.8854	3.564	3.564	1.075	11.149	0.1682	0.5984	0.005	0.6215
	0.1365	9.8828	3,4963	3.4963	1.0723	11.103	0.1681	0.5962	0.002	0.6195
	0.1353	9.2322	3.354	3.354	1.1456	10.443	0.168	0.5962	0.00	0.6194
						10.899				0,6201

THE	2002	0.0	0 0	200.8	0	58.9	158.9	58.9		58.9	58.9	158 9		+ 09		900	2	60 1	010	900	3	60 1	90.0	160		54 7	54.7	54.7		547	54.7	247	5	7 4 7	54.7	54.7	
H %	4691	4500	200	3			.4526 1			1		2.4601		318 1	1	2 8218 1	-!		8185			-1	- 1	8299			-	5168 1			5149 1		•	Į	- 1 *-	5149	
Big Pi	٥	0	4 0	i	10	N i	N	2	က	25 2.			4			i	_	0	110	110	1	0	110	liα	<u> </u>	-	-	: -	1	1	-	1	•	L	-	1 -	į
	16	2 4		<u>د</u> -	1	0.0			16.13	16	16.2	-	- 43	2				2	1 0			16	2	2	2	6.1		6.157		L	9			\vdash	6.144		-:6
GE R3*R3	4F-04		4E-04		100		/E-04	/E-04	7E-04	9E-04	9E-04	8E-04	9E-04		2F-04	2F-04	2F-04		4F-04	4E-04	4F-04	4E-04	4E-04	5E-04	4E-04	0.003	0.003	0.004	0.003	0.005	0.005	0.005	0.005	0.007	0.007	0.007	7000
β	9479	947 9		0.470	0470				947.9	949.5	949.5	949.5	949.5	949.5	949 5	949.5	949.5	9495	949 5	949.5	949.5	951.1	951.1	951.1	951.1	946.2	946.2	946.2	946.2	946.2	946.2	946.2	946.2	946.2	946.2	946.2	0 970
V*A	1489	1489	1489	1480	1480	000	488	1489	1489	1491	1491	1491	1491	1491	1491	1491	1491	1491	1491	1491	1491		1494		. 770		1486		1486	1486		1486	_3866	1486		:	400
KG (mg)	0.48	045	048	047	045	2 0	2.043	.043	.044	1	1	2.048	2.047	.355	353	2.356	2.355	.355	2.346	351	.351	358	354	2.354	.355	.262	263	263	262	263	262	264	263	262	263	:	
ш	┖		0.652	-333	651 2	2 0	COO	2 00.	::::1	0.652 2		0.652 2	7770000					_		0.748 2			0.749 2		0.75 2.	0.402 1	-	_	402 1,	0.402 1.	-	0.402 1.	402 1.	-	-	0.402 1.	Ŧ
×		-	0.027 0	_ 333	-		0.027		ា				0.027 0.	<u> </u>		0.027	ں -	ـــ			- 200	—							.027 0.4	-	.027 0.4		027 0.4	.027 0.4	027 0.4		700
Nu			25.5 0.0				27.0				-		4	32.1 0.0				•	31.4 0.027			1	.4 0.027		C	15.2 0.027			0		0	0	4 0	0	0	-0	0
Corr Re I	626 26		-		ــــ	600		_ 1	3 -				627 24,			-	830 31.7		822 31		826 30.7	833 30	830 29.4	-	831 30.4		- '			237 15.4			237 15	237 15.4		237 15.	237 15
h /m/2k	33.8	27.6		30.7	_	7 76	i	- 3	L	- ;	/	_	_		55.1	!	62,1 8	54.2 8			67.3 B		50.4 8		% 1	77.83 2	1			:	78.44 2				1	77.22 2	34
FIS W		631 12	633 13	<u></u>	631 13		1	_}		634	Ť-;		12		_	839 16	16	_	832	_	16	841 15		839 15	40000	240 77			220		240 78				240 76		77
mic V (mV)		303 (ternan			o.	o cr	. i			بأريس	304			vacanos s	.501			494 8	497. B		.503				804 2	- 1	- 1		i	803 2			803 2		803 2	
= -	584 1.	-	-		584 1.		584		ľ	585 1.	_	-	1	Ť	585	•			-	585 1.		586 1.			- [0	583 O.8	0	1	0	0	583 0.8		0	0	583 O.E	
	0.61				2.97				Ę	N (5	04						14.53 5		day.			16.3 5			8,2 5	W.			0.59 5				5	4	 3	
	0,1 10				ັດ	~	a	ľ		5 5	ຕ ທີ່	n	1.			0.11		0,13 14		V. 1	- [0.14 15				0.08			1	0.1 10			1		12.		
					9.81 0	2	10.7 0.1		**************************************	4.0	Ni.	SA QC	333.	6.56 0.1				10.1	822.74			11.6 0.	V-255	e de la companie de	S .	6.93	Section,				M,	11.4		sanç.	14.6 0.11	14.4 0.	*
	30.5					1	35.5			0.00				31.3			- 1	35.2	-	1		17.3	18.4	8.8		30.1			- 1	34.6					8.7	2.5	_
) (0)	31.6	32.2	32.1				37.1		4h7	7.0	4.	40.4	2	34.0	31.8	32.8		37.1 3	36.8	37.4		39.6 37.3	40.7	4.1.4 S	2	2000		7) 4: 0	1	S	36,1	35.4 3		.υ .υ	40,1 38.7	υ. Σ	
F (0	. 24	24.1	24.3				24.8			- 020			320, 333	24.7				25.1			1	,	8.07 8.07	6.62	0 00	7 C		0.0		20.00				יי עריק עריק	- -		
ation	ecceps;	7	8200Ç0 -	edt.	-websi		gain					1		:00.30 i	1	**********						LTOQUE					26470	anger.						en l		adejn.	100
Inform	~ 584	L = /3 cm	7H~2							1			204	~ 204	L = /3 cm	H - 2				-		!			204	- 204	100	-									

overall Rs uncert	0.533	0.5337	0.5333	0.5334	0.534	0.5347	0.5347	0.5345	0.5328	0.5335	0.5331	0.5332	0.4707	0.4709	0.4706	0.4707	0.4709	0.4725	0.4716	0.4716	0.4699	0.4707	0.4707	0.4704	0.8381	0.838	0.838	0.838	0.838	0.839	0.838	0.8383	0.839	0.839	0.839	0.839
funcert	0.002	0.005	0.002		0.002	0,002			0.005	0.005	0.002		0.002		0.002		0.005	0.002	0.005		0.002	0.00	0.002		0.005	0.005	0.005		0.005	0.005	0.005		0.005	0.005	0.005	
Mic V ungert		0.5065	0.5061			0.5077		!	0.5057		0.5061		0.4397	0.44	0.4397		0.44	0.4418	0.4409		0.4391	0.44	0.44		0.8209	0.8209	0.8209		0.8209	0.8219	0.8209		0.8219		0.8219	
Ta	0.1683	0.1682	0.1681		0.168	0.1679			0.1676	0.1676	0.1675			0.1679	0.1676		0.1676		0.1675		0.1673		0.1672		0.1687		0.1687		0.1685	0.1684	0.1683		0.1683	0.1682		
overall Nusselt uncert	16.426			16.163	_	12.308	12.24	12.391	10.433	10.929		10.773		17.342	16.131	16.458	12.144	12.458	12.181	12.261	11.104		-	10.755	:	-	17.15	17.374	13.224		13.122	13,162	11.705			11.645
Req	1.4975		1.4608		1.4591	1.3947	1.3877		-	1.362	1.3724			1.7349	1.865		1.7249		1.7547			:	1.7878		-	0.8826	0.8468		0.8841	_	0.8814		0.8807		0.8639	
To uncert	7.6705		7.4478			4.7708	-		-		3.8148		-		7.8353		4.963	5.2611	4.9768	1		4.0084	3.8648			7	6.752		4.537		4.3906			3.4227		
Te	7.6705	:	7.4478		5.095	4.7708	4.6901		3.7256	3.7859	3.8148		~	:	7.8353		4.963	i	4.9768			4.0084	3.8648		7.2137	\sim	6.752				4.3906			3.4227		
	12.242		12.2		-		10.191		8.8839	9.4286	9.438				11.573		o) i	<u> </u>	9.7851	_	1	9.0681	8.5863	_		14.271	14.22			!	11.525		10.527	10.504	10.509	
V uncert under	0.1682	0.1646	0.1671		0.1476	0.1449	0.1438		0.1342	0.1341	0.1342		0.1611	0.1658	0.1629		0.1415	0.1438	0.1408		0.1351	0.1321	0.1314		0.1885	0.1837	0.1831		0.1586	0.1566	0.1555		0.1448	0.143	0.1438	
Expariment Information V	í ~ 584	3	PR ~ 2.5										f ~ 584	73	PR ~ 2.8										1 ~ 584	73	PR ~ 1.5									-

TuS %	L	9 156.7	1	ï		-:	1567		8 156.7		156.7		158	158	9 158.2		1	-			1	158	1		2 159.6	_	3 159.6		_	-	159.6		Ľ	-	
E	1.8979		1.8979		L	-	1 8998	· !		1.8941	-	!	1	N	2.2639		101	2.2677	N		101	103	2.2658		2	a	2.6563			N	2	<u>:</u>	2	2.6544	: N
Bs.		9.642			6	6	9.667	_3026	_	9.61	9.725	4000	L	13.76		_776	13.79	: "	13.82	- T.	_		-	13.79			18.9		18	18	18.93	18	18.91	18.89	
Hs*Rs		0.001		-0.77	⊢	0	0		-	0		0.003			7E-04			1E-03					0.001	0.001	4E-04			4E-04	5E-04	5E-04	5E-04	5E-04		6E-04	6E-04
β (2ΔΔ/π)	946	946.2	946	946	946.2	<u>. </u>	947.9	- 1700	947	947.9	947	947.9	946	946	946.2	946		947.9		947	947.9		947.9	947.9	947.9		947	947.9	949.5	949.5	949.5	949.5	949.5	949.5	949.5
*	1486	1486	1486	1486	1486	1489	1489	1488	1489	1489	1489	1489	1486	1486	1486	1486	1489	1489	1489	1489	1489	1489	1489	1489	1489	1489	1489	1489	1491	1491	1491	1491	1491	1491	1491
9 (ag	1.581	1.581	1.581	1.581	1.58	1.579	1.581	1.58	1.581	1.577	1.586	1.581	1.889	1.889	1.886	1,888	1.888	1.887	1.89	1.889	1.891	1.89	1.887	1,889	2.212	2.206	2.211	2.21	2.211	2.21	2.21	2.21	2.21	2.209	2.211
ω	0.503	0.503	0.503	0.503	0.503	0.503	0.503	0.503	0.503	0.502	0.505	0.503	0.601	0.601	9.0	0.601	0.601	0.601	0.602	0.601	0.602	0.602	0.601	0.601			_	0.703	0.704	0.703	0.704	0.704	_		0.704
×	0.027	0.027	0.027	0.027			0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027		0.027	_	_	0.027		0.027	0.027
Z	18	19.2	18	18.8	17.4	-		17.3			18	18.7	21.1	_	22.8	21.6	21.1		ď	21				22.1	29.6	29.1	27.8	28.9	28.8			28.1	28.5	28.1	28.5
n Rs 2K		3 372		3 372	371	372	i	5 372	5 373	3 371			- 1			5 530		531						1	730	1	36		- ;	- 1				:	
h Wmrzł		98.16		96,19		89.11		88.75	96.55			22.		- :	116	110.5	<u> </u>			107,4			112		151	148	142	14		136.3		3L	144.5		146
/ Rs)	3 376	3 376	3 377	N.C	376		377	7.000		i				:	536		2000	ernd	539		i 	539	mar la	į		/34			739	L	738			737	
mic V (mV)		1,006	1,006			1,006			1.007	1,004	<u>.</u>		1,202	1,202	1.5		1,203	1.202	1.204		1,204	1,203	1.201		1.409	7.405	1.408		1.41	1,409	1,409		1,408	1,407	1.408
f Freq (Hz)			583		200	584				584				583					584			584			584						585	2		585	
	Ĭ				31.1	11.32	11,52		13		<u> </u>			10,3	10,56		12.51	12,73	12,6		X (4)	14.66			12.12) - -	5 2 2		45.44	2,0	14,17		5		
CC CDC AND V 280 1		2000	20.400			0.1		3230	0.12	and in	0000	4000	60'0	ane,	0.1	380 P	0.1			1000	a q	0,13	2222	000	0 0	ana	T. Miller	3	3 6	3 ;				0.14	
(C)					0.	10.4	의		13.3	9	14.		6.88	7	7.4			9	9		<u>e</u>		14	ľ	7.24	1	Ö	5		2 9	2		12.6	12.9	12.6
Tito Ts/s=T (D) (C) (C	8 30.1	30.8	31.2			34.6			2 37.6					7 30.8			34.5		34.9		38.	4 38.5	38.8		4.12	0 0	36.	70	24.0	0,10	34.5		37.6	38	3/.8
Ta . To	23.8 30.8	3.8 34	3.9 32.			24.2 35.8			24.9 39.2				23.6 31.4	3.6 31.7			24 35.9					24,6 40,4			24.2 32.8			20 1	24.4 00.7		4.7 3b.l		25 39.	25.1 40.3	5.2 40.
	300030	ww.	7000		5	ભ	Ň		ര്	Č.	Ň			zzy.				Ň (3		, v	N C	77		28.07.0	100	1	0	v č	i c	77			N C	V
Information	f ~ 584	L = 73 cm	PH ~ 1.9										- 584	L = /3 cm	PH ~ 2.									703	~ 584	00 00	2								:

0.1803 14.517 7.9764 0.1747 13.024 7.0105 0.1734 12.997 6.8253 0.1536 11.656 4.8266 0.1536 11.65 4.6617 0.142 9.8905 3.7573 0.1393 9.8788 3.6076 0.1393 9.8788 3.6076 0.1695 13.168 7.2696 0.1665 13.145 6.9881 0.1658 12.021 6.7344 0.1457 10.874 4.6849 0.1465 10.859 4.6785	7.9764 7.9764 7.0105 7.0105 6.8253 6.8253 6.8253 6.8253 4.7865 4.7865 4.6617 4.6617 3.7573 3.7573 3.6076 3.6076 3.5133 3.5133 3.5133 3.5133 6.9881 6.9881 6.7344 6.7344 6.7344 6.7344	0.9938 0.9938 0.9881 1.0802 1.0682 1.2099 1.1919 1.3073	16.406 16.226 17.016 13.545 13.423 13.496 11.128 11.171 11.101 11.101 11.184 16.783 16.211	0.1685 0.1688 0.1681 0.1681 0.1689 0.1689 0.1689 0.1689	0.5591 0.5591 0.5591 0.5591 0.5591 0.5591 0.5591	0.002 0.002 0.002 0.002 0.002 0.002 0.002	0.6773 0.6773 0.6773 0.6773 0.6773 0.6773 0.6773
13.024 12.997 11.656 11.65 11.65 9.8905 9.8905 9.869 13.145 12.021 10.873 10.859							
12.997 11.656 11.65 11.65 9.8905 9.869 9.869 13.168 13.168 12.021 10.873 10.859		L 0 0 LL-	16.226 13.545 13.423 13.496 11.28 11.101 11.184 16.75 16.78 16.393				
9.8905 9.8905 9.8905 9.869 9.869 13.168 12.021 12.021 10.873		0 0	13.545 13.545 13.423 13.496 11.184 11.184 16.75 16.783 16.781				
9.8905 9.8905 9.869 9.869 13.168 13.168 12.021 10.873		0 0	13.545 13.52 13.496 11.28 11.101 11.184 16.75 16.489 16.211				
9.8905 9.8905 9.869 9.869 13.168 13.145 12.021 10.873 10.859		0	13.423 13.496 11.28 11.171 11.101 11.184 16.489 15.393 16.393				
9.8905 3 9.8788 3 9.8788 3 9.869 3 9.869 3 12.021 6 10.873 4 10.874 4 10.874 4 10.876 3 9.3507 3			13.423 13.496 11.28 11.101 11.184 16.75 16.489 15.393				
9.8905 3 9.8788 3 9.869 3 9.869 3 12.021 6 10.873 4 10.874 4 10.874 4 10.859 4 9.3507 3			13.496 11.28 11.171 11.184 16.75 16.489 15.393	000			
9 8905 3 9 8905 3 9 8 8005 3 9 8 8005 3 9 8 800 3 9 8 800 3 9 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8			11.28 11.171 11.184 16.75 16.489 15.393				0.6766
9 8 7 8 8 3 9 8 7 9 9 8 7 9 9 8 6 9 9 9 8 9 9 9 8 9 9 9 9 9 9 9 9			11.171 11.184 16.75 16.489 15.393	0.00			0.6785
9.869 13.145 12.021 10.873 4 10.859 4 10.859 4 10.859 3507	6 6 6 4		11.101 11.184 16.75 16.489 15.393 16.211	0,0,0			
13.168 13.145 12.021 10.873 10.874 10.859		1.2099 1.1919 1.3073	11.184 16.75 16.489 15.393 16.211		0.549	0.002	0.6747
13.168 13.145 12.021 10.873 10.874 10.859		1.2099 1.1919 1.3073	16.75 16.489 15.393 16.211		0.549	0.002	0.6766
13.145 12.021 10.873 10.874 10.859		1.1919	16.489 15.393 16.211		0.549	0.002	0.5744
12.021 10.873 10.874 10.859		1.3073	15.393		0.5	100	0.5744
10.873 10.874 10.859			16.211		9	0.0021	0.5752
10.873 10.874 10.859	4			_			0.5746
10.874		1.206	12.848	0.1683	0.5486	0.005	0.5739
10.859		1.2071	12.791	0.1682	0.5491	0.005	0.5743
9.3507	785 4.6785	1.1931	12.772			0.002	0.5734
9.3507	_		12.804				0.5738
	- :	1.2762	10.779	0.168	0.5482	0.005	0.5733
9.3408	ന	1.2653	10.72	0.1679	0.5486		0.5738
0.1338 9.3305 3.537	37 3.537	1.2539	10.662	0.1679	0.5495		0.5746
	-		10.72				0.5739
11.391			15.1	0.1682	0.4684	0.002	0.4977
282 11.365			15.063	0.1681	0.4698	0.005	0.4989
0.1544 11.284 6.3941	41 6.3941	1.5931	14.548	0.1681	0.4688	0.002	0.498
			14.904				0.4982
9.6893	4		11.952	0.168	0.4681	0.002	0.4973
10.325		1.5245	12.571	0.168	0.4684	!	0.4976
0.1427 9.6972 4.8913	13 4.8913	1.6579	12.027	0.1679	0.4684	0.005	0.4976
		- 1	12.183				0.4975
0.1333 9.0118 3.9568	3.9568	1.6165			0.4688	0.002	0.4978
9.0009 3.0	3.8766			0.1676	0.4691		0.4981
9.0201	3.903	1.0334	10.75		0.4688		0.4978

	X.			_		153	6 153.6	153		153	8 153.6	153		Ľ		1547				1507	-		_ : "	154.7		L		1557	- !	- 1	122.7		- :		155.7	
PR%	Ľ	- ; -	1.3244			_	1.3376	_		-	1.3338	1		-	-	15149		-	1	1 5074	- 1			1 5140		1.	,	1 7036		1 6998	1,6998	1,600,8		1 6979	1.7036	1.7017
ш	1	320	1.925		٦Ľ	-	_			二	1.953			LCA	ICV		CV				2.497	2515	0 500	2 522	2 500	3 182	3 10	3 0	3 187	3 175	3.174	3 174	3,174	3.165	3.191	3.184
Rs*Rs			0.02	200	7L	0.03		<u> </u>	0.031		O			0.012	0.012	0.012	0.012	0.019	0.019	0.018	0.019	-1			0.023	0.008	0.007	0.007	0.00	0.011	0.011	0.011	0.011		014	
A B			1714	_	L	1/14		_	1714	_	1714	_	1714	1714	1714	_	1714	1714	1714	1716	1714	1716	1716	1716	1716	1716	1716	1716	1716	1716	1716	1716	1716	1716	1717	1717
V*V	2690	2600		_6	81	- 1			2692	2692	2692	2692	2692	2692	2692	2692	2692	2692	2692	2695	2693	2695	2695	2695	2695	2695	2695	2695	2695	2695	2695	2695	2695	2695	2697	2697
9 8		0 800	0.003	0.61	0,0	0.013	0.615	0.614	0.614	0.614	0.613	0.612	0.613	0.695	0.696	0.696	0.696	0.694	0.693	0.693	0.693	0.695	0.697	0.696	0.696	0.782	0.783	0.783	0.783	0.781	0.781	0.781	0.781	0.781	0.783	0.782
ů.	0 194	0 104	0.195	0 194		0.00	0.10	0.195	0.195	0.195	0.195	0.195	0.195	0.221	0.222	0.222	0.222	_	0.221		0.221	_	222	222	0.222			0.249	0.249	L	249				0.249	249
×	0.049	0.049	0.049	0.049	SIC.	o∵ c	0.0	⊃ 1	0.049	0.049	0.049	0.049	. 200 L				0.049		0.049	0.049	0.049	0.049	0.049		0.049	0.049		049	0.049	0.049	0.049		049	049	049	0.049 (
n N	=	113	1	11.2	L	, `	1:0				£.	_	7:	=	11.2	11.3		72		Ξ	12		11.9	-	12	5			12.1	4	7	5.	9	Q:		2
h Hs ^2K	33 99.9		<u>-</u> 2	100	L		100		٦Ľ	102					5 131	_		7 130	_		٦l	5 131	-	Т,	2 131	2 165		_	8 165			_	TL			601
Rs W/m		101 57.7	102 56.69	57,16	٠.		03 50 64	_0	₩ .		03 57.83				32 57.5				61.3		ို		3 60.82		inc.		7 59.49		222	63.6		9	ini .	<u> </u>	63.02	33
	0.702 10							1	1	-: 1	707						. '	0.8	-	/99 13			804 133	-:)3 167		ernada ernada	T :	11 167			168	-!
	055 0.7	13.7			056 0.7	056 0 709		160	į	> <	000	>	ľ		USB 0,803		ı	•	O (0	ľ	9	O	Ó		7 0.902	80.8°	100		0		5	1	140	o c	>
	7.31 10				200	45	53 1					190.0			10 to	3000				7501 84			11.28 1057		- 1	8.22 1057	7501 7.	A		9,4 1057	730L 69			0.00	10 1058 44 1058	3.63
	0.07				90:		0.09				- ; - ;							600	١	D .			0.1			0,00 0,00 0,00				3 60'0	э (77			1 11 44	
State of the State	1000		7.85 0	2324	62000	11.5	20020	8222 -		ij.	15.00 G			S###	200	il in		7 0					15.3		250 Bar	0 7 5 7 0 0	Sec. 25.	30.400	8004	Special Con	5 c	20,22	82, 822	J. 188	14.9	S 60
20	0.2	30.6	30.8		34.3	34.7	35.1				30.6							25.0				0.00		8		21 0			!	0.40	- ; ▼	- 1			39.4	
0 0 0	30.7		9 7 97				36		2 3 4 7 7	14.5. 1	40.7	-3590			6		36.4	28.5	2,46	3	30.7		40.4	1,7890	4 00	, a	2 6	40		52.0 26			415	40.0	40.6	
(C)	22.8	900p	53		23.1	23.2	23,5	•	23.6	23.7	23.8		23.7	D 2.0	200	}	23.7	23.7	7.00		24	3	4 2	, A	0.00	22.0	0.00	5.5	Ye	† VO	24.1	- 1	24.4	7 7 Z	24.55	
Information T.a. T.to	- 1055 - 1	/3 cm	~ 1.3							:	:		1055	73 cm	PR ~ 15										055	= 73 cm	17									
Ξ,	1	1 (ĭ				i			-	;		2	11	РВ				!						7	1	a d			1	-				!	

18.222 0.1688 18.404	18.222	0 6343 18 222	6.3707 0.6343 18.222	5 ROE 6 3707 6 3707 0 6343 18 999	0.6343 18.222
		2000		0,000	
. 1	15.249	0.6089 15.249	4.4478 0.6089 15.249	4.4478 4.4478 0.6089 15.249	13.877 4.4478 4.4478 0.6089 15.249
13.926 0.1687	13.926	0.6785 13.926	4.3354 0.6785 13.926	4.3354 4.3354 0.6785 13.926	12.47 4.3354 4.3354 0.6785 13.926
	14.34	14.34	14.34	14.34	14.34
12.104 0.1685	12.104	0.6548 12.104	3.0999 0.6548 12.104	3.0999 3.0999 0.6548 12.104	3.0999 3.0999 0.6548 12.104
12.131 0.1684	12.131	0.647 12.131	3.1707 0.647 12.131	3.1707 3.1707 0.647 12.131	3.1707 3.1707 0.647 12.131
12.141 0.1684	12.141	0.6545 12.141	3.1731 0.6545 12.141	3.1731 3.1731 0.6545 12.141	3.1731 3.1731 0.6545 12.141
18.392 0.1684		0.6357 18.392	6.6077 0.6357 18.392	0.6357 18.392	6.6077 6.6077 0.6357 18.392
18.293 0.1686	18.293	0.6434 18.293	6.4535 0.6434 18.293	6.4535 6.4535 0.6434 18.293	15.84 6.4535 6.4535 0.6434 18.293
18.245 0.1685	18.245	0.6466 18.245	6.3788 0.6466 18.245	6.3788 6.3788 0.6466 18.245	6.3788 6.3788 0.6466 18.245
	18.31	18,31	18.31	18.31	18.31
	13.893 0	0.6989 13.893 0.	4.2419 0.6989 13.893 0	4.2419 4.2419 0.6989 13.893 0	12.51 4.2419 4.2419 0.6989 13.893 0
13.834 0.1684	13.834	0.6815 13.93	4.3368 0.6815 13.93	4.2698 4.2698 0.6859 13.894 4.3368 4.3368 0.6815 13.93	12.493 4.2698 4.2698 0.6859 13.894 12.488 4.3368 4.3368 0.6815 13.93
	13,906	13.906	13.906	13.906	13.906
- 1	12.316	0.6881 12.316	3,4275 0.6881 12,316	11.3 3.4275 3.4275 0.6881 12.316	11.3 3.4275 3.4275 0.6881 12.316
12.226 0.1683	12.226	0.6805 12.226	3.2787 0.6805 12.226	1.292 3.2787 3.2787 0.6805 12.226	11.292 3.2787 3.2787 0.6805 12.226
12.266 0.1683	12.266	0.6855 12.266	3.3415 0.6855 12.266	1.298 3.3415 3.3415 0.6855 12.266	11.298 3.3415 3.3415 0.6855 12.266
12.27	12.27	12.27	12.27	12.27	1000
	0.1683	0.7442 16.561 0.1683	6 1503 0 7442 16 561 0 1683		12.7
0.1684	200			6 1503 6 1503 0 7449 16 561 0 1683	14 072 6 1503 6 1503 0 7442 16 561 0 1683
0.1004		10 E4 0 400 A	6 7100 0 6666 10 51 0 1684	6.1503 6.1503 0.7442 16.561 0.1683	14.072 6.1503 6.1503 0.7442 16.561 0.1683
	18.51 0.1684	0.6656 18.51 0.1684	6.7109 0.6656 18.51 0.1684	6.1503 6.1503 0.7442 16.561 0.1683 6.7109 6.7109 0.6656 18.51 0.1684	14.072 6.1503 6.1503 0.7442 16.561 0.1683 15.876 6.7109 6.7109 0.6656 18.51 0.1684
0.1084		10000	001.00	6.1503 6.1503 0.7442 16.561 0.1683	14.072 6.1503 6.1503 0.7442 16.561 0.1683
_		1 00000	2000	6.1503 6.1503 0.7442 16.561	14.072 6.1503 6.1503 0.7442 16.561
	12.26	0.6855 12.266 12.27 0.7442 16.561	3.3415 0.6855 12.266 0. 12.27 6.1503 0.7442 16.561 0	3.3415 3.3415 0.6855 12.266 0. 12.27	11.298 3.3415 3.3415 0.6855 12.266
13.846 12.134 12.141 12.145 18.392 18.393 13.893 13.893 13.893 13.893 13.893 13.894 13.996 12.226 12.226 12.226 12.276 16.561		0.6673 0.6548 0.647 0.6545 0.6434 0.6466 0.6466 0.6869 0.6881 0.6881 0.6885 0.6885	4.2281 0.6673 3.0999 0.6548 3.1707 0.647 3.1731 0.6545 6.6077 0.6357 6.4535 0.6434 6.3788 0.6466 4.2419 0.6989 4.2698 0.6859 4.3368 0.6859 4.3368 0.6815 3.34275 0.6805 3.3415 0.6805 6.1503 0.7442	4.2281 4.2281 0.6673 3.0999 3.0999 0.6548 3.1707 3.1707 0.647 3.1731 0.6357 6.6077 6.6077 0.6357 6.4535 6.4535 0.6434 6.3788 6.3788 0.6434 6.3788 6.3788 0.6665 4.2419 4.2419 0.6989 4.2698 4.2698 0.6859 4.3368 4.3368 0.6815 3.2787 3.2787 0.6805 3.3415 3.3415 0.6855	12.47 4.2281 4.2281 0.6673 11.262 3.0999 3.0999 0.6548 11.263 3.1707 3.1707 0.647 11.261 3.1731 3.1731 0.6545 15.827 6.6077 6.6077 0.6357 15.84 6.4535 6.4535 0.6434 15.845 6.3788 6.3788 0.6466 12.51 4.2419 4.2419 0.6989 12.493 4.2698 4.2698 0.6815 12.498 4.3368 4.3368 0.6815 11.292 3.2787 3.4275 0.6805 11.292 3.2787 3.4275 0.68055
	0.673 0.6548 0.6548 0.6545 0.647 0.647 0.6445 0.6445 0.6445 0.6889 0.6889 0.6881 0.6881 0.6881 0.6881 0.6881 0.6881		4.3354 4.2281 3.1707 3.1731 6.6077 6.4535 6.3788 4.2419 4.2698 4.3368 3.4275 3.34275 6.1503	4.335.4 4.335.4 4.2281 3.0999 3.0999 3.0999 3.1707 3.1707 3.1707 3.1707 3.1707 3.1707 3.1707 3.1707 3.1707 3.1707 3.1707 3.1707 3.1707 3.1707 6.4535 6.4535 6.4535 6.4535 6.4535 6.4535 6.3788 6.3788 6.3788 6.3788 7.2698 4.2698 4.2698 4.2698 4.2698 4.2698 4.2698 4.2698 4.2698 3.375 3.3775 3	12.484 4.3354 4.3354 12.47 4.2281 4.2281 4.2281 11.262 3.0999 3.0999 11.253 3.1707 3.1707 11.261 3.1731 3.1731 15.845 6.3788 6.3788 6.3788 12.493 4.2698 4.2698 12.493 4.2698 4.2698 12.493 4.2698 4.3368 4.3368 11.292 3.2787 3.4275 11.298 3.3415 3.3415

Tas 9			156.0			7 156.6		_	1	156	156	156.6		1	1	157.5	1	157 4	157	1,7	5	157	157	157.5		158.2		1		158	-	158	3	158		158	3
PR %		1 8866	1	-!	ľ	-!	1.8809		ļ	_	1	1.8809	,	_	_	2080			2 0771		- · · · · ·			2.0809				2.2733		,	ļΝ			2 2582	2.262	2.2639	
Bs A	3.91	C.	ď	9	8 L	3.904		3.88	3.89	3.884		က	-0.00	1				4			_ 3/ 5/		4 758		4,755	5.66	5.66	5.679	5.666	5.649	5.667	5.629	5.648	5.606	5.624	5.633	
Rs*Rs	0.005	0.005	0 005	0.005	2000	0.007	0.006	0.007	0.007	0.00	-		0.003	0.003		0 003		0	0.005		200	0 007		0.007	0.007	0.002	0.002	0.002	0.005	0.003	0.003	0.004	0.004	0.005	0.005	0.005	
AAA/#)	1717	1717	1717	1717	1717		1717	1717	1717	1719	1719	1719	1719	1712	1712	1712	1712	1712	1712	1712	1712	1714	1714	1714	1714	1714	1714	1714	1714	1714	1714	1714	1714	1716	1716	1716	-
1*1	2697	2697	2697	2697	7000	7607	24.97	26.97	2697	2700	2700	2700	2700	2690	2690	2690	2690	2690	2690	2690	2690	2692	2692	2692	2692	2692	2692	2692	2692	2692	2692	2692	2692	2695	2692	2695	1 1 1 1
¥ (€	0.867	0.867	0.869	0.867	9980	0.000	0.805	0.864	0.865	0.863	0.865	0.864	0.864	0.963	0.956	0.956	0.959	0.955	0.954	0.952	0.954	0.955	0.956	0.956	0.956	1.043	1.043	1.044	1,043	1.042	1.044	1.041	1.042	1.037	1.039	1.04	
ů.	0.276	0.276	0.277	0.276	0.076			275	0.275		0.275		0.275	0.307		0.304			0.304						0.304	0.332	0.332	0.332	0.332	0.332	0.332	0.331	0.332	0.33	0.331	0.331	1000
X	0.049	0.049	0.049	0.049	070		0.049		0.1149		0.049		0.049	049	0.049	0.049	0.049		0.049		0.049		0.049	_			_		0.049 (0.049 (0.049 (0.049		049	4
Na			11.7	11.8	_			5	860	13		9	200	15 (14.5	14.6				14.4	Q			14.3				14.6 C	<u> </u>	9	4	15.6 0			9	(, ()
h As v2K		203	204	204	L	1	1			:	203					247	248	246	246								i		294	293	294	292	293	-			200
W/m			59.95	60,3	65.52			_	~ L							74	74.84		75.02		300	_			73,38	75.05	74.87	74.98	74.97	80.56	80.01	79.04	79.87	84.92	77.54	84.9	97 00
V Rs	1 205		2 206		9 205		i	1		i	i	i waa a	1.		249			248		i				3 249		age day	297				297	ana ita				295	200
V mid V (mV)	8	(XX)	8 1.002		8 0.999	S	9000	5 O S		966'0 6		9 0,997				5 1,104		5 1,102		3000		3		3 1,103		7	503.1			-	~	C!			1,15	3	- Vallet Voltage Volta
	7,61 1058		7.95 1058		58 1058	9.14 1058		100		6 1059		34 1059				29 1055		3 1055	6 1055	9 1055			9 1056				4 1055				B 1056		(I			201 8	000000000000000000000000000000000000000
									ľ		1 11.49		ľ	x	00	₩		7	10.6				1 12.19			0,00 0,00				10,6		11,03	5	2.2	3		N. 800 Million
SECTION STATE	7.16 0,07	886.3		23333	2000	9.95 0.08	1415.14	3883		n i	0.0	4 .		7.03 0.08		~	. J.	٥.	o.		VAC: W	way.	0 '	.00,2,44		32 U.U8				8. 0.1	<u>. </u>		1	o c	- C	5	
			32 7.			34.8 9.					39.4 14	9.4			29.8 7.6	9.6 7.32		33.4					38 14.9			30.4 7.32		1	_		-1	34.9		30.4 14.0	38 0 4.8		_
	. 1		32.6			35.5			900	0.0	0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0	ני ני		0 0				D 0		ກຸ			39,4		1 1	- c	2 4	הׄוֹ ה		N O		00			40 K		
	2,42	1. 2000	1,6230			24.8					14.3 14.3		90 100	- 0 0 0				4.4		KE.0	80 KO	99.99	23.1			93.				200		£0,4			0.00 A EC		
Information	~ 1033		9.										25	2	3	-					200	1			,	Ca Ca		?				55,700					88
ioju.	CCO ~ 1	100	D.							1			1055	2 2	100/-1	~		-					-	***************************************	1055	13	PRZOS										

IIIOMIIIOMI		uncert	Ta	Ta uncert	Req uncert	Nusselt Ta uncert uncert	Tauncert	Mic V 1: uncert uncert	fi	Rs uncert
~ 1055	0.1952				-	18.74	0.1681	99.0	1	0.681
33	0.1926			9	0.6683	18.515			i	1
PH ~ 1.9	0.1889	15.885	6.5507	6.5507	0.6708	18.402	0.168	Ö		
						18.552				0.6806
	0.1669	12.554	i	4.6206	0.733	14.173	0.1679	0.6607	0.005	1 -
	0.1713			5.0228	0.6758	15.691	<u> </u>	0.662	1	1
	0.1665		4.6222	4.6222	0.7363	14.178	0.1678	ļO	0.005	
						14.681			:	
	0.1511	=	[0.746		0.1678		乚	
	0.1481	=	က	3.4415		12.369	:	0.6613	0.005	
	0.1501	11.338	3.4833		0.7204		0.1677	0.662	0.005	
		-				12.417				0.6829
	0.1882	T:			0.8595				0.002	0.6172
~	0.1833		_ :	!	0.8193	16.974	0.1693	0	0.002	0.6213
PH ~ 2.1	0.1864	14.2	6.8269	6.8269	0.8331	17.192	0.1692		0.005	0.6213
		_				17.207				0.62
	0.1595		4.5653	4.5653		14.194		0.5989	0.002	0.6223
	0.15/8	= ;			0	13.018	0.1691	0.5995	0.002	0.6228
	0.1601	11,498		4.488	0.858	13.163	0.1689		0.002	0.6244
		_ {	_]			13.458				0.6232
	0.1421				0.8127	11.436	0.1688	0.5989	0.005	0.6223
	0.1441	2	3.3483	3.3483	O į	11.52	0.1688	0.5984	0.005	0.6217
	0.1426	10.467			0.8243	11.483	0.1688	0.5984	0.002	0.6217
100	0.000		- 1			11.48				0.6219
~ 1022	0.1856	14.209		6.8315	0.8397	17.204	0.1688	0.5486	0.005	0.574
2		14.207		6.6635	0.8377	17.069	0.1688	0.5486	0.002	0.574
FR ~ Z.3	0.1/94	14.208	6.5057	6.5057	0.8389	16.949	0.1688	0.5477	0.005	0.5731
						17.074				0.5737
	0.1588	11.549	4.6213	4.6213	0.9013	13.301	0.1687	0.5491	0.002	0.5744
		11.541	4.5049	4.5049	0.8952	13.214	0.1686	0.5482	0.005	0.5735
	0.1544	11.529	4.3571	4.3571	0.8843	13.103	0.1686	0.55	0.005	0.5753
	0					13.206				0.5744
	0.1418	9.7633			0.9501	10.907	0.1685	0.5514	0.002	0.5766
	0.1413	10.513		3831	0.8676	11.584	0.1684	0.5505	0.002	0.5757
	0.1403	9.763	3.3091		0.9498		0.1684	0.55	0.002	0.5752
				_		11.12				0 5758

TdS %	1	1	563 158 9		158	7.00	77 158 9		158	158	1		079 157.4		33 157 4		33 157.4	! ***	-		96 157.4		33 157.4		158	58 158.2	158		158		158		77 158.2	158.	158
E.	٥	100	IIΟ		1	0	6 2 4507	i.	_	8 2.445		<u>.</u>	2		9 2.0733		2.07	2.07	iQ		5 2.0696	Ø		i .	0	3 2.2658	N	9	2	5 2.2677	2		10	N	N
I S A	9		9	. 041	2 6.63	1	9.9		9	9		ဖ်	_	11.53	-	- 1, 2°	二	2 11.49	-	3.64	2 11.45	:		2 11.48	13.7		Τ.	13.73	Ŀ	3 13.75		. 🐬	Ł	53	
p Rs*As	3	0.005		- 4	0.005		0.003		1					-	0.001	- 1000	0.001		3 0.001	3 0.002	_			3 0.002	3 7E-0	J 7E-04	3 7E-0	1 7E-04		1E-03	0.001	1E-03			0.001
2AA/n		1716	-	1716			1716		-	1716		1716	<u> </u>		952.7	. 177	_	952.7		953.3	L		954.3	954.3		954.3		954.3		954.3	954.3	954.3	954.3		954.3
*	2695	1	2695	. 18			2695					2695	_	-	1497	1497	-	1497		1497	1499			1499		1499		1499	1499	-	1499	1499	1499	1499	
\$ (g	_	_	1.128	. 📑	_	-	1.126	. 🖺	L	_		1.123	_	1.718	1-	1.717	-	1.715	_	1.715	-	1.714	-	1.713	1.872	-	- :		1.872	-	1.874	1.874	1.876	-	1.87
4	0.359	_		0.359	0.359		0.358	. 1330	0.358	0.358	0.357	0.357	0.547	0.547	0.546	0.547		0.546		0.546	0.545	0.545	0.546	0.545	0.596	0.596	0.597	0.596		0	0.596	0,596		Oi	0.595
×		0.049		0.049	0.049		0.049	0.049	0.049	0.049		0.049	0.027	0.027	0.027	0.027		0.027		0.027	0.027	0.027	0.027	0.027	0.027	0.027		3 L			0.027	0.027			0.027
s. Nu			17.3	4.16.3	16.3	_	3 16.1	16.2	\vdash	17.4		0.17.4	7 20.9	5 22.4	5 21.4	3 21.6	5 20	Ò.	5 20	5 20,4		ŭ	N:	1 20.5			_	5.	1	Ν		22.7	3 22.6		22
h Rs W/m/2K	73 344	i	45 344	25 344.	58 344	344	38 343	56 344	32 341		32 339	37 340	.1 447	.9 446	.8 445	.6 446	445	9 445		.6 445		!	3		2 531			1		- !		116 532	9	G I	./ 529
Hs W/m	347 80.73		348 88.45	83,25	348 83.58		346 82.38	82,66	15 89.32		343 88.82	89,07	448 107.1	448 114.	16 109.	110,	446 102	446 108.9	446 102	104,6	444 100.8	106.2	446 107	104	32 118.2	533 118	3			-!	533	-		533 113.	_
mia V - I (mV)		,302 3			302 3	Saara	ture reco		296 3		293 3				.099			i Sames	099 4		097 4		099 4		i armete	201	renda.	-	(V)	- :			en anne de		0 981
Freq. m (Hz): ((057 1.3	057 1.3		-	1057 1.3	1057 1.2		1057 1.2	•	1057 1.3			587 1	587 1.0			7	588 1,(-	588 1.0	-			588 1,					588 1,2		-		200
	1000		7				11,18 10		3,12 1(3.46 1(43	0.64				2.93				4.22				99.0			820	5 ;	3.35			4.94	ø
Cur (A)	80.0				0.1				0.12 1		_			0.1			0,11				CI.	0,13	m m	1						7 (72		<u>.</u>		2
le-Ta (⊙	7	7.27	7.84	- 1	10.8			859863	14.5	14.7	15				8.1		11.1	11.7	10.7		13.6	14.3	14.1		7.41			0	10.6	10.7	11:2		13.6	14	5.5
1 0	1 30.7				3 34.7				38.5						3 31.8		3 34.9			- 1		38.5	,		31.7				35					38.7	
	23.7 31.4	20,000	9.000	800 SO B.	23.9 35.8				24 40.1			12000	23.4 31.3		200	20000	23,8 36,3		24 36,1	33333 (3)		24.2 40.4	2000 00 100	333 B	1,3 32.8	17000	2000	22.0	4,05,4	201000	Sec. 200			40.7	
lion.		noe, i	8000 :	0000	ଝ	ଝ	23				24		Grane I	:	230 XQC 1		83	X			54	Ž.	24		24.3	and per-	1000000	-	4.4	¥ 3	47		24	24.6	4
Information	f ~ 1055	L = 73 cn	PR ~ 2.5										- 585	L = 73 cn	PR ~ 2.1		:					-			~ 585	13 CID	2.7								

overall Rs uncert	0.5345	0.5342	0.5342	0.5343	-	0.5341		0.5345	0.5363	0.5363		0.5367	0.6222	0.6227	0.6237	0.6229				0.6237	0.6247	0.6236	0.6236	0.624	0.5751	0.5747	0.5742	0.5747	-	0.5742	0.5746	0.5747		0.5751	
funcert	0.002	0.002	0.002		0.005	0.002	1	:		0	0.002			0.005	0.002		ĺ	0.002			-		0.002			0.005	0.002		0.002	0.002	0.002		0.002	0.002	0.002
Mic V uncert	0.5073	0.5069			0.5069		0.5081		0.5093					0.5995			0.6005	0.6005				0.6005	0.6005		0.55	0.5495	0.5491		0.55	0.5491	0.5495		0.5491	0.55	0.5509
Ta uncert	0.1684	0.1684			0.1683	0.1683			0.1683					0.1685			0.1684	0.1684	0.1683		0.1682	0.1682	0.1681		0.1681	0.1681	0.1681		0.168	0.168	0.168		0.1679	0.1679	0.1678
overall Nussett uncert	17.534	17.315	15.759	16.869	13.349	13.149	-	13.241	11.004		-	10.97			-	15.57			12.709	12.359	11.272	10.572	10.62	10.822	15.417	15.417	15.099	15.311	12.852	12.181	12.016	12.35	10.782	10.7	10.641
Red uncert	0.9032	0.9016	0.9895		0.9351	0.9178	0.9216		0.9993	0.9965	0.9937		1.1979	1.2854	1.2281		1.146	1.2182	1.1451		1.1275	1.188	1.2048		1.3229	1.3229	1.2861		1.2316	1.3413	1.3198		1.2928	1.2745	1.2611
To uncert	7.1434	6.8744	6.3773		4.637	4.3718	4.4801		3.4494	3.4062	3.3437		7.1692	6.5656	6.1743		4.5079	4.2668	4.6565		3.6762	3.4927	3.5422		6.7446	6.7446	6.4186		4.7352	4.6692	4.4775		3.6642	3.5663	3.4961
Ta uncert	7.1434	6.8744	6.3773		4.637	4.3718	4.4801		3.4494	3.4062	3.3437		7.1692	6.5656	6.1743				4.6565		3.6762	3.4927	3.5422		6.7446	6.7446	6.4186		4.7352		4.4775		3.6642	3.5663	3.4961
l uncert	14.301	14.299	12.884		11.588	11.568	11.572		9.8114	9.8087	9.8059		13.152	11.995	11.929		10.809	10.026	10.808		9.9368	9.2707	9.286		12.039	12.039	11.996		10.9	10.146	10.125		9.3658	9.3492	9.337
V uncert	0.1831	0.1784	0,1739		0.1559	0.1516	0.1537		0.1402	0.1393	0.138		0.1689	0.1645	0.1619		0.1461	0.1445	0.1491		0.1372	0.1362	0.1364		0.1649	0.1649	0.162		0.1453	0.1457	0.1432		0.1347	0.1334	0.1326
Information V	f ~ 1055	L = 73 cm	PR ~ 2.5										f ~ 585	L = 73 cm	PR ~ 2.1										f ~ 585	73	PR ~ 2.3								

JAS (GB)	58.9	58.9	58.9		58.9	0 0	2000	2	58.0	0 80	58.0		59.5	70.0	159.5		59.5	159 6	59.6	2!	59.5	59.5	59.6	-	60.1	60.1	60.1	:	90.1	60,1	50.1		30.1	30.1	60.1	
₽€	582	: _	1		4544 1		4488						л.		6469 1				6525 1	.;	1	-	.6506	!	1-	! -	.8374		1-	2.8299 1		. 1			8318	
As PR	2.4	3 2.4	3 2.4	ĺ	0	110	1; C	1!	. —	-		ii	100	•		i	10	N	is	11	L		cq	-	2	2	N	1	-	-			N	ijα	ll (VI	4
	_	16.13	:		L	1	16.02	34	ாட	,	15.98		1		18.73	3.00	1		18.8	- 34	ł		18.79	1000		21	21.48	ò	2	21.39	2	2	2	2	21.42	2
Gr Rs*Rs	5E-04	5E-04	5E-04	5E-04	7E-0	7F-02	7F-04		9F-04	9E-04	1E-03		3E-04		4E-04		5E-04	5E-04	5E-04	5E-04			7E-04	7E-04		3E-04	3E-04	3E-04	4E-04	4E-04	4E-04	4E-04	5E-04	5E-04	5E-04	6E-04
β (1/4/π)	951.1	952.7	952.7	952.2		952 7	952 7	952.7	952.7	952.7	952.7	952.7	952.7	952.7	952.7	952.7	952.7	954.3	954.3	953.8	954.3	954.3	954.3	954.3	954.3	954.3	952.7	953.8	952.7	952.7	952.7	952.7	952.7	952.7		953.3
1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1	1494	1497	1497	1496	1497	1497	1497	1497	1497	1497	1497	1497	1497	1497	1497	1497	1497	1499	1499	1498	1499	1499	1499	1499	1499	1499	1497	1498	1497	1497		1497	<u> </u>		1499	1497
9 (€	035	031		.032	.028	028	2.025	2.027	2.022	2.024	2.023	2,023	189	2.188	2.19	2.189	2.186	2.19	2.192	2.189	.187	2.187	2.192	2,188	.341	339	2.344	.341	2.34	.339	338	339	337	332	2.339	936
3	648	646 2.	646	647 2	646	645	645	645	_	0.644 2	0.644 2	644 2			0.697	0.697 2	<u> </u>		0.698 2			0.696 2		0.697 2	0.745 2			745 2.		CV.	N	0.745 2			0.745 2	
1	0	0.027 0.	0	027 0.	0	10	0.027 0.	0	-			027 0.		-	0.027 0.	027 0.	0.027 0.			4183				30.00	_	_		0		_		339		027 0.		027 0.
D.O.	25.9 0.0			25.4 0.0			25.3 0.0	25.7 0.0	1		25.5 0.0	4	L	29.4 0.0		28.9 O.C	30.6 0.0			. 322	_	29.7 0.027		1200	30.7 0.0			31.2 0.027		8	6	7 0	4	0	7	4. O
Corr		624 25		624 2E		622 2E	-	621 25	618 24		618 25	619 25.				724 28	722 30			22	27	724 29		3925	830 30			830 31		827 30		827 30,		22 31.	829 31	20 3
h W/m^2K	7	6		29.8	33.5		129.4	31.6		34.3		29.9 (:	150.4 7		148 7		151.1 7		4		į		48.9 7	1	0	4			57.6	5	157 8		-	62.2	
Rs W	627 13		_	12	624 13	623 13	-	7	匚	621 13	-	12			727 14				730 14	<u>.</u>		726 15	-		832 15	:	833 161	15	830 15		829 15			825 15	-	٩
mic V (mV)	303	:	!			1.3	*******			,297			.403 7						406 7			8	405 7		Ξİ	T. Samo	504 8		- :	 8	1			495 8		
A4450 CO.	586 1.		587 1,		Ť	87	587 1.		-	587 1.	Τ		587 1.	7			587	•	-			588 1,4	-		-	588	Ť		7	587	7		7	587 1,4	Τ.	1
	98	22	13		83	26	62		63	15.6 5			33	02	38		. .	52	63			6.48 5			12.5 5					4.92				7.13 6		
	0,1 10	Ŧ	Ξ		12 13	7	-		13	4	4		Ξ	<u> </u>	S.		13	13 14	T.			15 16			_		_			3 14			i La	S		
	8	9	7		0	о́ С	5.0		1 0.	0.	0		6.45 0.1			3332	9.63 0.		22.92	***	-1	3.4 0,	৵৾	888810	7.2 0,1		*****		33.2.3	10.1	Lane N			13.3 0.1		
	29.8 6.7	0.1			2.9	3.4 1	33.9 10.		36.7 13.		37.8		30				33.5					37.8 13.			31.5					33.3				36.8		-
20	30.9	31.2	37.03		34.5	35.1 3	35.6		38.7 3	39.3	40,1		0 0	32,3	32.6 3	•	35,4 3	36,2 3	36.6		£_7.68	40,3	6,3		32.9 3	32.6	31:1		5,3	35,33 20,13	2.0		က ၈၈	න . ආ	ଫ ଧ୍ୟ ପ୍ର	1000
7. (C)	23 30.9	22.9	22.9		23.1				23.6	23.7	23.8		23.6	23.7	23,7 .:		1000	24	2000			24.4			24.3	24.3	22.8		23.1	7.00 K	50.07		43.4	23,5 39,4	23.8	CASSING AND
	1					0.000		SS\$00.					2.000)2 :	9000) :								-										33 (18 k
Experiment Information	~ 585	L = /3 cm	H ~ 2.										~ 585	L = 73 cm	H~2.									100	~ 585		PH ~ 2.8				:		4	:		

Information V uncert	f truncet	Taumcert	To Uncert	Req 1	Ausee	I Ta uncert	Mig V uncert	funcert	Rs
0.1646	46 12.228	7.3705	7.370	1.485	16.13	0.1688	0.5065		0.5339
0.1614			9	-	15.749	0.1689	0.5065	1	:
0.162	62 12.163	6.9779	6.9779	1.429	15.728	0.1689	0.5065		
					15.872			!	0.5339
0.1464	64 10.295	5.1116		•	12.668	0.1688	0.5073	0.005	0.5346
0.144		4		1.4762	12.508		0.5077		
0.1422	22 10.251	4.7559	4.7559		-		_	:	- 0
					12.508		The same of the sa		
0.1333				1.3977	10.988	0.1685	0.5093		
0.1331	31 8.9152	3.7387			10.474			0.002	
0.13	312 8.8789		3.5636	1.4596			0.5093	1	0.5364
	_	- 1			10.592				0.5363
0.163	63 12.382			1.6173	_	0.1685	0.4704	0.005	0.4997
0.1578		6.8977	6.8977		15.083	0.1684	0.4708		0.5
0.1549				1.6678	14.851	0.1684	0.4704	0.005	0.4997
	-				15.519				0.4998
0.1444				1.7509	12.356	0.1683	0.4714		0.5006
0.1411	11 9.7263	4			12.034	- 1	0.4698	0.002	0.499
0.1399		4.6899	4.6899	1.6412	11.851	0.1683	0.4694		0.4987
	-				12.08				0.4994
0.1314			_ :	1.5954	10.614	0.1681	0.4708	0.002	0.4999
0.1315	15 8.5164		:	1.6997	10.167	0.168	0.4708	0.002	0.4999
0.1315		3.7377	3.7377	1.7022	10.169	0.168	0.4698	0.002	0.4989
		1			10.317				0.4995
0.1553	- :		:	- :	15.195	0.1681	0.4397	0.005	0.4707
0.1577			7.2188		15.477	0.1681	0.44	0.002	0.471
0.1573	73 11.51	7.2293	7.2293	1.8056	15.501	0.1689	0.4388	0.005	0.4702
					15.391				0.4707
0.1391		4.9018	4.9018	1.7693	12.133	0.1688	0.4397	0.002	0.471
0.1399		4.9393	4.9393	1.7628	12.158	0.1687	0.44	0.005	0.4712
0.1386	36 9.7687	4.8082	4.8082	1.7367	12.029	0.1687	0.4403	0.005	0.4715
					12.107				0.4712
0.1307	9.5956		3.8442	1.7994	10.329	0.1686	0.4406	0.005	0.4718
0.1298			3.7649	1.78	10.254	0.1685	0.4415	0.002	0.4726
0.1313	3 8.6076	3.8995	3.8995	1.8146	10.383	0.1684	0.4397	0.002	0.4708
					10 999				7717

SP (B)		1	7.00		1	60.7	160.7	160.7		60.7	60.7	1			i c	VI C	Ÿ.			Λic				N)		1.	-1.		-!	1-	- 1	72.4	3:	Ţ÷			:
				- :	- 1					1	1		- :			0 9		1		0 9	- :			9		157		157	-	157 1	- }	1	i	157	- ; -	157	1
PR%	3 028		0.000	 د	I		3.0261				30185			10		סוכ	٠	1		3.2204	0.666	- 10	დ į (3.2166	3.214/	1 0041	2 0017	20017		1 9885	1 0808	10000	270	1 9979	1 9979	1.9998	1
As ,	24 48	24 57	27.57	0.45	40.42	24.52	24.47	24.44	24.48	24 44	24.36	24.39	24.39	97 AG	07.70	07 70	27.72	07 50	07.03	27.70	07.0	00'/2	27.65	27.03	0.72	4 348	4 381	4.381	4.37	4 323	4 207	4 297	4.306	4 371	4.371	4.379	
G <u>r</u> Rs*Rs	2E-04	2F.04	20.75		10.10	3E-04	3E-04	3E-04	3E-04	4E-04	4E-04	4F-04	4E-04	2F.04	20.00	2F.04	2010	20 00	210	2F-04	0 H 0	100	3E-04	опо 20-11-04	3F-04	0 004	0.00	0.004	0.004	0.006	0.006	0.00	0.006	0.008	0.008	0.008	0000
β1 2ΔΔ/π)	95,	954 3		054.9	0.4.00	304.0	954.3	954.3	954.3	954.3	954.3	954.3	954.3		050									952.7		1709	1709	1709	1709	1709	1709	1709	1709		1711		
V+V	1499	1499	1499	1499	1 100	200	1499	1499	1499	1499	1499	1499	1499	1494	1494	1494	1494	1497	1497	1497	1407	1407	1497	1497	1497	2685	2685	2685	2685	2685	2685	2685	2685	2687	2687	2687	1000
9 (2)	2.501	2.535	2 505	2504	2 503	200	2.5	2.45.9	2.501	2.439	2.495	2.497	2.497	2.651	2 651	2.649	2 651	2 655	2 659	2.661	2 65R	0 650	0.00	2 657	2.658	0.919	0.922	0.922	0.921	3.916	0.913	0.913	0.914	0.92	1		000
¥	·	-	0.797	- 66	707 0	300			0.796	0.796		0.795			0.844		844			0.847		_		0.846		1		0.293	60.00	0.292 (-	0.291	-222	.293	0.293		000
×	2	0.027		-33		0.00		_	0.027		0.027		0.027	0.027		-	- 13	-		0.027	-96	-			-933	0.049	0.049		0.049 (0.049 (0.049 (0.049	049	049	2
2			36.4	************	21		0 0		34.9		33.1		33.1			37 (37.1	_		36.6	60.00	13.5	13.5					12.8 0	12.8 C		14 0.		
h 2K	948		951	. 330		047			948		943		945		1061		- 7 1000	1067	1070	1071	1070	-					227		226	224			223			227	1000
W/m/		_	186.5	186.6	181	÷	:	8	178.5		_	168	169,3	183		189.4	187	188			189,8	-		187.6	Alle	69.15			69,04	65.29	65.55	65.55	65,46	71.89	71.41		04
V As		i 	8 953			4 950		entes		3 949	أوسو	1 947	secole	1 1064	7 1064	9 1063		1		٠.	lanca.		5 1072		2.62	en i	231	mode		secretar	escore è	227			230	- I	200
q mia V (mV)	_	Ť	8 1,608		9 1,606	-		•		=		3 1,601		3 1,701	5 1.7	3 1.699			707,1	Т			7	1,704			1,061			1,054			1		7	1,06	
Volt Freq (V) (Hz)			35 588			35 588					180	.6 588				1 586			7 687	11 587		7 587			80 M	2000	3 1053		88 L	2 1053		4 1053				8 1054	CONTRACTOR OF THE PARTY OF THE
	13.29						4 15 46				5 17,49		1		12 13.48				4 15.87			16 17,67	-				8 8.23					9 10:24			12.04		
7a Cur (C) (A)	07 0.12	81 0.	6.9		C)	9.8833 	0.14	.	ı	2.5 0.15	100,000	n i	ď	0	7.05 0.1	o.	224523	ಜನಾಗ	54 0.14	-		Ö	ó	9			33 0.08	44.7		60,0	W. All	22.3		. 6000	0.7	00,000	200000000000000000000000000000000000000
TSTS-Ta (C) (C)	30.8	30.5		•			34.1 10			30.0		37.1 12.			29.3 7.				32 9.54		1	34.8 11.9	36 12.9	36.1 12	- 1		30.8 7.83			34.5	4./ 11.6				38.4 15.2		-
SXXXXXXXXXX	32.4		32.2		35.8	36.1	36.3			N C			1	2	30,9	30,7			34,3		- 1		www.ve.l	39,1 3			31.5		7 3	ψ.(c) ψ.(c)	33.0	35.6			کر در در در د		4000000000
F 0	23,7	1500		12.		Sec. 11	7	23	383		7.4				22.2				22.5		200000	22.20	23.1	300000	A		33 8			3 .			ON \$100	USS 26.	, , , ,	dost	10000000000000000000000000000000000000
Information	- 585	= /3 CIII	FH ~ 3.0										202	200	L = /3 cm	H ~ 3.2									4000	~ 1033	L = /3 cm	2.5				otto, le					0

1	2	0.00	0.005		0.411 0.002 0.4441	1	0000	0.00	0000	0.00	0.002	0.4452	88 0.002 0.4234	0.005	0.005		0.005	0.005	0.005	0.4221	0.005	0.005	0.005	0.4223	0.005	0	0.002 0.6446	0.6453	0.005	_	0.005	0.6497	2 0.002 0 6457	1
Overall Nusselt Ta Mic V	0.1684 0.4112	0.1684	0.1684		0.1683	0.1683	•		0 1682	0.1682		- 1	0.1695	0.1693	0.1693 0.3885		0.1692		0,1691		0.1689 0.3869	0.1688 0.387	0.1687 0.3873				0.1688 0.6221		0.1688 0.6262	0.1688 0.628	0.1688 0.628	-+	0.1687 0.6232	0 4004
	4	*	-	15.061	0249 12.026	_	-		9034 10.541	-	1		_ i		-:			37 12.209		_	24 10.5		1	_	16.776	16.776	7 16.713	_	14.003	13.986	13.986	13.991	11.436	11 150
To Req uncert uncert	8	101			5.1444 2.02	5.0457 1.9952	4.9536 1.9728		1-	3.9291 1.8967	1.8733 1.8815			7.0911 2.1106	.2631 2.1185	-		5.2437 2.1437	5.1466 2.122	-		al	3.8758 2.0984			966 0.	0.31	-		4.3251 0.7334	3251 0.7334	- 1	3.2414 0.8043	787
Ta T	7.0678	7.3384	7.2425		5.1444	5.0457	4.9536		4.0096	3.9291	3.8733	000	6.668	7.0911	7.2631	0.0	5.1809	5.2437	5,1400	1	4.1953	3.8758	3.8758	-		90	0.3		_	3251	4.3251 4.	20414 06	0.5414 3.0	0.7787
í t uncert	54 10.863		9 10.876		3 9.3576		9.3135			7 8.6727	_	- -		0.833		\perp		D C	9,4403	5		φ (α	8.3194	- 1	44.1.4		-			12.555	- 1	10 446	277	77.0
ent Ion V uncert	0.154	1	0.1559		0.1413	0.1406	0.1398		0.1309	0.1297	0.1291	0 4 40	0.1493	0.1332	0.1352	0 400	0.1389	0.1388	0.100	1001	0.1324	0.1280	0.1280	0 4060	0.1002	0 1852	0.100	0 1600	0.1009	0.1397	0.1397	0 1433	0 1448	うたたこう
Experiment Information V	f ~ 585	33	PR ~ 3.0									f FBE		0000	₹ .									1058		PR ~ 20	i		The second secon	The second section of the second seco				

×	20) 	Volt Fred		mic V F	Hs 	- H	H3	DC.	×	ω.	ð	V*V	σ.	Hs*Rs	. Bs	s PR%	SPL
7.81		၂ဗ	ľ	52		4R 2	71	81 2	266		٥	1)	(2.A.A.m.)				
31.2 7.99 0.08		98		8.66 1054		1,15	272 71	27 2		13.9 0.0	049	318	000	7897	171	0.003	5.137	2.1658	- 1
7.99		08	æ	3,66 1054		1	7	27	-	0	0	1:00		2687	171	0.00			157.8
40.2	Wiek.						7.1	.45 2	267	14 0,0	049 0.	318 0	966	2687	1711	0 003	-		-
34.2 11 0.09	ame ji	60		0.36 1054	1,000					_	0	_	.002	2687	1711	0 004	_		1
-!	28.99.y.	8			∵.	,148 27	271 69		266 13	3.6 0.0	0	_	997	2687	1711	0.004	5 135		
F	W 999	60	2					:	_		0.049 0.		0.999	2687	1711	0.004	-	2 1696	157.0
						1	2000		267 13.	9	0.049 0.		666	2687	1711	0.004	×		-:
14.6	0 ∘ 9 ∣	Ξ.	2		Ţ.,	.149 27				_			866.	2687	1711	0.006	اير	0	157
38.2 14.7 0.11	0 (\	Ξ:	<u> </u>	2.38 1054		48 271		76.12 20	266 14	6	0.049 0.3	0.318 0	0.997	2687	1711	0.006		2.1658	157.8
14.8	Σ D	=	N.		8.00	.148 27				8			766.	2687	1711	0.006		N	157
7.57	330 B.C	ş	្រ	1						9	.049 0.:	318 0.	866	2687	1711	0.006	5.136	j: -	i
75.7	district 10	9 8	Ď	5 5				;		5.1 0.0	0.049 0.0	-	.084	2687	1711	0.002		_	1
90.0 / 5.7 6.08		9 8		9 1054		254 323						0.346	.088	2687	1711	0.002		2.3658	158.6
00'/		3	D)	5					_	Ŋ	0.049 0.3	-	780	2687	1711	0.005			
007						1	22		317 15	Q	. 5 %	0.346 1	980	2687	1711	0.005			•
33.5 10.3 0,1	(°)		9 '			243 317		83.67	312 16		L	0.344	620	2687	1711	0.003	1	L	
7.0.7	 			10.9 1055		- 1				16.4 0.049		0.344	1.08	2690	1712	0.003			
1.0	- -		-	3901 BO		49 320						_	084	2690	1712	0.003	6.091	2.3563	158.5
44.7	1.0.40		: I (80 F			W000		314 16.	9	وسرقي	0.344	.081	2689	1712	0.003			
7.4	<u> </u>	V (N	2.96 1055		54 323	-			17 0.049		0.346 1	980	2690	1712			2	
20.2 14.6 0.12	ם מ	N C	2			254 32	3 86.84			/	:	-	.088	2690	1712			i N	1
2	200,000	VI.		3,1 1058		55 323		_ 3	_	8			680	2690	1712	0.004	6.146	2.3677	158.6
S 1999	S 1999	g	°		7	-		L	\\L	ol	and L	2	43.33	2690	1712	0,004	6.14		
7 25	A	? 5				·		- 1				-		2690	1712	0.001	7.251	101	159
31 7 55 0.08	Agriculture.	Şα	D 0	0.19 1025 0.10 105E		364 360		1	5 16.1	.1 0.049		9	.182	2690	1712	0.001	7.24	2.5695	159
			>			an An an	00.00	0/5 70			9	9		2690	1712	0.001	7.229	CV.	159.3
10.0	1	·	7				∞⊩	L		၁ ၈	ee L	3.0	.182	2690	1712	0.001	7.24		!
0.00 0.10 0.00 0.10 0.00 0.10		- •	= ;	1,26 1055	ä.	1,36 380	0 86.28			9 0.049		0.376 1.	181	2690	1712	0.002	7.217	CA	
v 0		- , ,	É,	100			_	- 1	4 16.	4		-		2690	1712	0.002	7.215	0	
7.1		-	Ξ	432		379			16	4				2690	1712	0.002		2 5639	150.2
							84,59	59 374	16	5 0	049 0.3	1		2690	1712	0000	_ 033	1	
14.7	0	CV.	<u> </u>		7			56 372	Ľ	7 0.049	49 0.37	150	1	2690	1719	0000		5	_L
0	0	CW		3.32 1055	5 1,355	55 377	7 89.59		1	5	_	110	1.177	2690	1719	0000	7 161	20002	23.0
14.7 0	0	C)			T				0 17.	0	049 0.3	ist		2690	1712	0000		4 C	
	2000 COOK 2000				100 V 100 V			1000										ı	

Experiment Information V		Та	.10	Red	Overal	Ta	MicV	-	overall
uncert		ert	uncert	uncert	uncert	uncert	uncert	uncert	uncert
14.157	- !	6.406			16.828	0.1687	0.5749		
9	9	2 1	- :	0.7973			<u> </u>	0.005	;
0.1801 14.148 6.2549	9	49	6.2549		-:	0.1687		<u>i </u>	0.5977
	-	- 1	_4		16.747				0.5984
12.61	ব :	25	4	0.7763	14.163	0	0.5719	1	
12.609		4.563			14.187		0.5749	:	
0.1589 12.613 4.5262		62	4.5262	0.7788	14.167	Ö	0.5739	0.005	0.5982
	_	-			14.172				
10.491		4182		0.8468	11.583		0.5744		0.5987
		98	3.3986		11.576	:	0.5749	0.005	0.5991
0.143 10.49 3.37		27		0.8458	11.557	0.1685	0.5749	- 1	0.5991
0.1778 14 247 6 6091	+-	15	B 5001	0 0000	710.1	000	1		0.599
14.261		85	ی اد		17.057	0.1689	0.5284	0.005	0.5548
14 255	4	312		0.0700	17.074		0.5263	0.002	0.5528
	Ď.		0.057	20.0	17.006	0.1089	0.5272	0.002	0.5536
		60		0.9362	13.494	0.1687	0.531	0.002	0.5571
0.1565 11.591 4.6778		Ω:	4.6778	0.9382	13.38	0.1687	0.5301	0.002	0.5563
		35		0.9208	13.25	0.1687	0.5284		0.5547
100					13.375			-	0.5561
0.1409 9.785 3.398		98	3.398	0.9724	10.946	0.1686	0.5263	0.002	0.5527
9.7842		7	3.377	0.9716	10.932		0.5263	002	0.5527
1772 3.3	က	339	3.3339	0.9643	10.898	0.1685	0.5259	0.002	0.5522
- 1	- 1	1			10.925				0.5525
14.320		0 9		0.9206	17.349		0.4842	0.002	0.5127
17.17.0 14.32.0 0		V.		0.9202	17.281	0.1686	0.4846	0.00	0.5131
0.1746 14.29 6.622	- 1	2	6.622	0.8958	17.11	0.1686	0.4849	0.005	0.5134
	_ !	1			17.247				0.5131
11.623			4.6511	0.9653			0.4853	0.005	0.5137
4	4		4.4508	0.9369	13.223	0.1685	0.4853	4	0.5137
0.1517 11.59 4.4508	4	ω i	4.4508	0.9369	- 1		0.4857		0.5141
100	1				13.279				0.5138
9.825	_				10.994	0.1684	0.4867	0.002	0.515
				1.0024	10.981		0.4871	0.002	0.5154
0.139 9.0144 3.40			3.4082			0.1684	0.4878		0.5161
		-			10.985				0.5155

SPL (db)	150.6	150 6	150.6		150 B	0 0	150.0	0	150 6	50.0	150.6		152.2	152.2	152.2		152	152.2	152				152.2		153.6	153.6	153.6		153.6	153.6	153.6		153.6	153.6	153.6	-
% H.J	0.9471	0.9471	0.9471		0 9489	0.07	0.0471		0.0471	0.9489	0.9489		1.1338	1.1338	1.1338	:	1.1338	1.1338	1.1357		1.1357	1.1357	1.1357		1.3282	1.33	1.3282		1.3282	1.3282	1.3282		1.3282	1.3282	1.3282	
As A	1.268	1.268	1 268	1.268	1 273	1 26g	1 268	197	1 26A	1 273	1.273	1.272	1.818	1.818	1.818	1.818	1.818	1.818	1.824	1,82	1.824	1.824	1.824	1.824	2.494	2.501	2.494	2,497	2.495	2.495	2.495	2,495	2.495	2.495	2.495	0 40E
As*As			0.055	-0.00			0.00		١		0.107				0.028	. 5	1	0.041		. 65			0.051	77	0.01	0.0	0.016	0.01	0	0	0.022	0	1	0		0000
(2AA(II)	1456	1456	1		L		1456	_	L		1456	1456		_	1456	1456		_		1456		1456		1456		1456	:	1456	1456	1456	:	1456	1456			1456
V+V	2287			-16.	1		2287		1	2287		100			2287	2287		2287		27		2287		2287	i	2287		2287	1	!	2287	2287	2287	2287	2287	25R7
KC (TE)	0	0	0.512	0.512	0.513	0.512	0.512	0.513	0.512	0.513	0.513	. 3000011	0	0		0.614	0.614								0.719	0.72	0.719	0.719	0.719	0.719	0.719		0	0.719	0	0.719
3	-	0.163	:	0.163	0.163		0.163			0.163		0.163	0.195	0.195	0.195	0,195	0.195	0.195	0.196	0.195	0.196	0.196	0.196	0.196	0.229	0.229	0.229	0.229	0.229	0.229	0.229	0.229	0.229	0.229	0.229	0.229
X	0.041	0.041	0.041	0.041	Į.		0.041	. 8383	10		0	0.041				0.041	0.041					0.041			0.041			0.041	0	0.041	0		0	0.041	0	0
rr Is Nu	6 11.1	6.01 9.	11	6 11	1		6 10.2					8 10				9 10.7	9 10.2	•	•	7	N.	N	2 10.3	300	19 11.2		19 10.9	19 11	7	19 11.7	19 11.9	Ξ	Ξ	9 11.5	Ξ	9 1 6
Corr h As n^2K		55.69 60.6	i	56,15 60,			52.19 60.6		L	51.18 60.9		400 L	55.61 86.9		4.		52.12 86.9			8	87	87	52.84 87	81 87.2		55.97 12		42	60.53 119	60.01	_			59.07 119	•	21
Rs W/	8	60.8 55	8	56	L	_	60.8 52		8	-	61.1 5	3334	N)	47	Ci.	54,	87.2 52	α;	5	~~ `	2	87.5 52	2	9300	120 5			56,		120 60		Some L		20 59		59
mia V (mV)	205	502	502		503	502	502	!	1	0,503 6			وأوسم	0,601 8	sees A		601	601	602		805	602	602		704	202	704		704	704	704		704	704	704	
2000		***	897 0				897 0.		330	897 0			200	897 0			897 0,		897 0		***	897 0			897 0	897 0,				897 0,	50		897 0	897 0	697 0	
Volt (V)	7.07	7.15	7,21		9.04	9.16	9.24		10.28	10.08	10.12		7.14	7.23	7.26		9.08	80'6	9,14		10.21	10,21	10.32		7.82	7,82	2.9		9.51	9.51	9,61		10,91	10.91	10.98	
G.	~	~	~		230	ó	0		0	0.09	0		0.07		30 W (c)		0.08	0 '	٥.	3330 M	200	0.09	Sec. 2.52		0,07	3	0			60'0	60'0		Ú,	0.1		
TS (9-T)	30.2 7.2						.8 11.6		.1 14.8	38 14.6	38 14.6		7.39				9 11.5					- 4			.1 7.84				- 1		2 11.7			15.2		_
######################################		31			35,3 34		35.6 34.8			38.9			31.3 30.8	11.5			35,6 34.9					38.7 37.8			31.7 31.1				35.9	36 35.1	35,			40 38.9		
T ₀	44.	23.1	23.1		23.2					23.4		200	4.50				20 0 4. 0 20 0					23.5 3			23.00 20.00 20.00			300 A 10		23.4		ones I	1.77	23.7	8.003	
Information	~ 900 Hz	E)	6.0	u ost i to			0.065.03	::::::::::::::::::::::::::::::::::::::	400000				900 HZ	۳. د	- i	of la	notivis	2500 ju			1			-	ZHO	= 0	.3					One S				200
) Info	1 ~ 90 1	L = 67 cm	PR ~ 0.9		-							,	200	L = 6/ cm	ĭ			:						100	ZH 006 ~ 1	L = 0/ CII	Ľ									

Experimen	11					100				
Information V	ηV	_	Та	To	Rea	Nisselt	T.	MicV		overall
	uncert	incert uncert	uncert	uncert.	uncert	uncert - urser	Section 25	uncert uncert	Uncert	CIDCRIT
1 ~ 900 Hz	0.2049		_	8 6.9488		18.638	2	3 1.3147		Ľ
67	0.203		9	9		18.489	o.		0.005	1.3255
PH ~ 0.9	0.2018	3 15.816	6.77	6.77	0.6287	18.5	0.1688	1		-
		1				18.542				1.3255
	0.171				0.5855	15.19	0.1687	1.3121	0.005	-
	0.1694		4	4.2907	_		0.1687	_	:	-
	0.1685	13.841	_	4.2931	-	15		1.3147	0.002	7
		-				15.145			:	-
	0.1567			3.3882	0.5768	13.263	0.1687		0.002	13255
	0.1587	12.348	က		0.5726	,	0.1686	13	0.005	1 3229
	0.1583		3.4313	3.4313	0.575	1		i -	0.002	1.3229
			-			13.276				1.3238
텕	0.2032	!			0.6222	18.487	0.1686	1.0982	1	1.11
γ ,	0.2011		6.5927	9	0.6139	18.35	0.1686	i	0.005	
FH ~ 1.1	0.2004	15.783		6.5088	0.6086	18.282	0.1686	1.0982		1
						18.373				1.11
	0.1705	- 1	4.3632			-			1	1.111
	0.1/02			4	0.5832	15.165	0.1685	1.0982		1.1
	0.1088	13.846	4.365	4.365		-:				1.1092
	0 4575	_	-		- 1	15.168				1.1104
	0.13/6		3.4811	3.4811		13.326	0.1685	1.0963	0.002	1.1092
The state of the s	0.15//				0	13.344	_ !	1.0963	0.005	1.1092
	0.1365	12.372	3.4594	3.4594	0.5912	13.318	0.1685	1.0963	0.005	1.1092
-11 000	1001	7.	0	ď		13.329				1.1092
	0.1900	10.030	0.3/59		0.6422	18.237	0.1687	0.9375	0.002	0.9526
3 2	0.1302	15.612		9	0.6262	18.104	0.1687	0.9362		0.9512
	000	0.0	0.1453	0.1453	0.6253	18.053	0.1687	0.9375	0.005	0.9526
	000,	00,				18.131				0.9521
	0.1668	12.482		4.3001		13.902	0.1686	0.9375	0.002	0.9525
	0.1667		,			13.873	0.1686	0.9375	002	0.9525
	0.1656	12.485	4.2668	4.2668	0.679	13.884	0.1685	0.9375	-	0.9525
	1027					13.886				0.9525
	1261.0	11.2/4					0.1684	0.9375		0.9525
	0.132	1.209	3.2921	3.2921			0.1684	0.9375	0.002	0.9525
	0.1314	11.269			0.6611	201	0.1684	0.9375		0.9525
						12.214				0.9525

SPL (dB)	154.7	154 7	154 7		1547	1	101	134.7	+ 64.3	104.7	154.7	: ;	158.6	1586	158.6		158.6	158.6	158.6	1	158.6	158.6	158.5		159.3	159.3	159.3	!	159.3	159.3	159.2	1	159.2	159.3	159.3
PB %	1.5111	15111	1.5111		15111	,	2 4	2	4 6000	2000	1 5093	:	2.362	2.3639	2.3658	:	2.3677	2.3695	2.3677		2.3677	2.362	2.3601	1	2.5639	2.5639	2.562		2.5639	2.562	2.5544	:	2.5582	2.562	2.562
SB v	3.23	3.23	3.23	3 23	3 23	0.0	3 0	0.60	3 223	2 222	3 222	3.222	-		14.94		_	-	14.97			91	88		17.57	17.57	17.54	17,56		17.55					17.55
Hs*Rs	0	0.01	0.01	0.01	0.013	0.00		2 6			0.017		6E-04	6E-04	6E-04	6E-04	9E-04	9E-04	9E-04	9E-04	0.001	0.001	0.001	0.001	4E-04	4E-04	4E-04	4E-04	6E-04	6E-04	6E-04	6E-04	8E-04	9E-04	8E-04
β (2.4.Α.π)	1456	1456	1456	1456	1456	1456	1456	700	1450	1456	1456	1456	949.5	951.1	951.1	920.6	951.1	951.1	951.1	951.1	951.1	951.1	951.1	951.1	951.1	951.1	951.1	951.1		951.1				952.7	952.3
Λ*Λ ,	2287	2287	2287	2287	2287	228.7	2200	2000	7022	2287	2287	2287	1491	1494	1494	1493	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1497	1495	1497	1497	1496
KC (TE)	0.818	0.818	0.818	0.818	0.818	0 818	9 0	9.00	0.010	0.00	0.817	0.817	1.956	1.954	1.956	1,955	1.958	1.96	1.958	1.959	1.959	1.955	1.953	1.956	2.122	2.122	2.121	2.121	2.122	2.121	2.112	2.118	-		2.12
3	0.26	0.26	0.26	0,26	0.26	0.26	0.26	0.10	0.20	0.26	0.26	0.26	0.623	0.622	0.623	0.622	0.623	0.624	0.623	0.623	0.623	0.622	0.622	0.622	0.675	0.675	0.675	0.675	0.676	0.675	0.672	0.674	0.673	0.674	0.675
X			0.041	0.041	0.C41						0.041		0	0.027	0	0.027		0.027		0.027				0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027
n N		_	4 10.6	4 10.6	_		10.4	- 33	<u> </u>		12.2				7 24.3	3 24.9		9 24.2		3 24.2		3 22.6		7 22.8		26.6		26.5		26	26.	. 1		27.	26.4
h Дз r/2K	-	_	-	-	_		41 154				47 154	03 154		5.7 576		.4 576		6.9 579		23.8 579		15.6 576				:	35.5 678	629 9	:			36		.1 678	
Rs W/m²	55 54.			54.28		55 62 45	55 63 41	_360	-		155 62.47	200			579 124.3	127.		581 123.9		123		578 115			_		681 135		682 139.1	681 134			. ,	680 141.1	
4	0.801	-			0.801	: ::	Jugar	70000		in a	0.8			253 5				256 5				,252 5	:		359 6	arrier.			a i voção	358 6				358 6	
Freq m (Hz) (897 0,	897 0		897 0.		20		897	897	897		-	586 1,	T		7	586 1,	7		586 1.	7	586 1.		586 1.	7	-		-	586 1,	-		7	587 1.	7
2477/2753	æ	8	8.12		6.79	9.79	9.85		11.03	11.03	11,14		10,51	10.71	10,74		12.87	12,95	2.91		14,75	15,04	14.96		1.08	7 . -	1,29			3,59				6.65	
₽(A)	0.07	0.07	0.07		00'0	0.09	0.09		0.1	0.1	0,1		0.1	0.1	0,1			0.12				0.13			0.1	- 	- - 0		0.12	0.12			0,14	0.15	0.14
	9 8.43						11.5		. 1		14.7				4 7.11			10.3			<u>ප</u>	13.9	5		6.77				9.47				13.4	3 14.6	13.8
	32,5 31.9		7 32.1		36 35.1						39,5 38.4			(4 29.3				.4 32.8				.8 36.8			.8 29.7				22.6					5 38	
	23.5 32			S 100 B	23.5	10000			1000	403 30	23.7 39			22.3 30,4				22.5 34.4				22.9 38.8		26 6 G	22.9 30.8	3 (3 (400	3.2	23,1 34,2	1000			1400	23.4 40.5	22.00
u O	2H 000	= 67 cm	~ 1.5										Jang. :	in halip	J., 84.)										2000ys.,	1 1 2 CIII	~ 2.0		Section 1						

_	1-	-	_	-	T_	-	-		101	101	101	n.	12			ım	-	101	7	~	+		+	m	01	0:	110	~	_			.~	-	_	-	_
overal Rs uncert	0.84	0.841	0.841	0.841	0.84	0.841	0.841			0.842		0.842	0.5537	0.5533	0.5529	0.5533	0.5524	0.552	0.5524	0.5523	0.5524	0.5536	0.554	0.5533	0.5142	0.5142	0.5145	0.5143	0.5141	0.5145	0.5158	0.5148	0.5151	0.5144	0.5144	0.5147
f uncert	0.002	0.005	0.002		0.005	0.002	0.005		0.005	0.002	0.002		0.002	0.002	0.002		0.002	0.002	0.002		0.002	0.002	0.002		0.002	0.002	0.002		0.002	0.002	0.002		0.002	0.002	0.005	
Mic V uncert	0.824	0.824	0.824			0	0.824	i			0.825			0.5267			0.5259	0.5255	0.5259		0		0.5276		0.4857	0.4857	0.486		0.4857	0.486	0.4874		0.4867	0.486	0.486	
Ta	0.1685	0.1685			1		0.1685	1					0.1693	0.1692			0.1691	0.1691	0.1691		0.169	0.1689	0.1689		0.1689	0.1688	0.1688		0.1688	0.1687	0.1687		0.1687	0.1686	0.1685	
overall Nusselt uncert	17.889	17.832	17.795	17.839	13.934	13.934	13.972	13.947	12.291	12.327	12.319	12.312	16.399	15.845	15.743	15.996	12.347	12.36	12.403	12.37	10.894	10.737	10.753	10.795	16.17	16.201	16.104	16.158	12.865	12.619	12.511	12.665	10.515	9.8496	10.406	10.257
Requincent	0.6112	0.604	0.6066		0.6987	0.6987	0.7094		0.6868	0.6962	0.6989		1.481	1.4059	1.3906		1.376		1.3945		1.3319	1.2934	1.2949		1.5056	1.5246	1.5165		1.5568	1.5098	1.5033		1.554	1.5787	1.5144	
Ta uncert	5.9318	5.8622	5.8001		4.3097	4.3097	4.3492		3.3839	3.4304	3.4096		7.6586	7.1344	7.0371		4.8423	4.8469	4.892		3.7749	3.5953	3.6186		7.3849	7.3983	7.3002		5.2815	5.0314	4.9015		3.7331	3.4355	3.6267	
Ta	5.9318	5.8622	5.8001		4.3097	4.3097	4.3492		3.3839	3.4304	3.4096			7.1344			4.8423	4.8469	4.892		3.7749	3.5953	3.6186		7.3849	7.3983	7.3002		5.2815	5.0314	4.9015		3.7331	3.4355	3.6267	
2000	15.787	15.775	15.779		12.51	12.51	12.524		-		11,313		12.223	12.136	12.118		10.18		10.198		9.4013	9.3664			12.252	12.274	12.264	- 1		10.311	10.305		8.9588	8.4203	8.9253	
V I Uncert uncert	0.1869	0.1867	0.1848		0.1639	0.1639	0.1634		0.1514	0.1515	0.1506		0.169	0.1657	0.1652		0.1472	0.1468	0.1472		0.1352	0.1332	0.1336		0.1636	0.1628	0.1619		0.1465	0.1443	0.1423		0.1311	0.1293	0.1304	
Experiment Information V u	f ~ 900 Hz	L = 67 cm	PR ~ 1.5										f ~ 585 Hz	L = 73 cm	PR ~ 2.3										f ~ 585 Hz	2	PR ~ 2.6									

SPL	159 A	150.8	159.8) !	1500	0.00	0.00	0.80	0	010	000	0.0	160.4	100.1	160.4	1:	160.4	160.4	160 4		160.4	160.4	160.4		161	161	160.9		160.9	161	160.9		160 9		160	
PR%	2 7393	2 7412	2 7355		9 7494	27.7	0 7200	- 1	0 7000	0 707	0 7055	6.7.0	9 0317	9 9317	2 9374	3	2 9374	2 928	2.9355		2.9317	2.9204	2.9299		3.1147	3,1147	3.1015		3.1129	3.1147	3.1129		3.1034	3.1091	3.094	
Bs			19.99	_	20.1	2	20.00	20.00					22 01	22 92	23.01	22.95	23.02	22.88	23	22.97	22.95	22.78	22.93	22,89		25.91		25.84		25.92		·	25.74	25.83	25.59	08 70
Gr Rs*Rs	3E-04	3E-04	3E-04	3F-04	SE-04	1 11	л П С	1000	PE OF	200	L C	6F-04	3F-04	3F-04	3F-04	3E-04	- 1	4E-04		4E-04	5E-04	5E-04	5E-04	5E-04	_			2E-04		<u> </u>		3E-04	4E-04	4E-04	_	- 3
g (ZAAVIL)	952.7	952.7	952.7	952.7	9527	0507	0507	0.00	0507	0507	955.7	952.7		9511			951.1	951.1	951.1			951.1			951.1	951.1	951.1	951.1	951.1	951.1	951.1	951.1		951.1	951.1	0811
V*V	1497	1497	1497	1497	1497	1407	1497	1407	1407	1407	1497	1497	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1494	1101
₹		2.266	2.262	2.264	2 26R	0 260	2 265	0.067	2 26E	2 264	2 263	2.264	2.421	2.422	2.427	2.423	2.428	2.421	2.427	2.425	2.425	2.416	2.424	2,421	2.577	2.577	2.566	2.573	2.576	2.578	2.576	2.577	2,569	2.573	2.561	OKER
ω.	0.721	0.721	0.72	0.721	0.722	0 700	0 721	0.725	0 721	0.721	0.72	0.721	0.771	0.771	0.772	0.771	0.773	0.771	0.772	0.772	0.772	0.769	0.772	0.771	0.82	0.82	0.817	0.819	0.82	0.82	0.82	0.82		0.819		7140
×	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	0.027	7600
N	30.3		30.5	30.2	29.9		29.7		-	1	30	ĕ	32.1		32.1			34.3		34.1	31.8		33	A40.1	36			35.7		က	35	34.9	35.1	35		34 8
L As 2K	2 776	6 777		7 775	3 778	8 779		_839	1	2	7 774	. 36	1 886	8 886	<u> </u>	7 887		4 884		3 888		2 880		3 885	-:	- !		666 6		1	1001	Τl	_ 1	,		700
s W/m/2		9 152.6		154.7	0 153	-	1		L	7 153.7	1	153.9	7 164.1	162	1 164.2	163.7		7 175.4	_	174.3				167.3	- :				- 1	- '		836	T-			- 426
<u> </u>		33 779			ł.	5 781	1		1	777	:		4 887	4 888	7 891		7 892		6 892			ĺ	3 890			-1	4 997				5 1005	1		-	4 993	
, mlo V (vin) (s	100				7 1.454	-				7 1,451				6 1.554				_	6 1,556		7	6 1,548	$\overline{}$		7	6 1,651		ı	200	7	6 1,65			ë	0 1.64	
Volt Tr. (V) (Hz	100	11,92 637				26	4.59 587		16.8 58		96 587		51 586	43 586			5,25 586					7,65 586	M			3.17 586	N.				52 586			18,17 586	08 586	
₹3 20 20	question of the second		1		1 3	5	€ -				15 16,		11 12.	11 12,48	11 12.				.14 15,			16 17.			ou i	N (23.	₩¥	0.14 16.08		5.			0.16 18.		
₽ ⊙	34,533	7.06 0.	main		20.00		10.3 0,				13.6 0,			6.91 0,	6.9 0.		O.	10.1	0	~	440	13.7 0,			0 (O 60.	Э.	2000	10.6 0.	900000		888 B		13.3 0.		
2012 Sept. 12000000	30.3						33.6				37.2			28.7 6	į		32 9					36.3			29.8 7				33.5			1		e i		
4 9	31.6	31,7	31.6		35.3	35,5	35.5		39.4	39,8	39.8		ဓ္ထ	30,1	30.2		34.2	34,6	34,8		38.4	39,2	39,4		کر خ	† 1	C		35.B	4,00	35.1	9	365	39,3	33.6	
	23.4	23.3	23,4		23.3	23.3	23,3		23.4	23.6	23.6		21.7	21.8	21,9		22.1	22.3	22.3		22.5	22.6	22.8	100	75.7	ייני עניי	7.5	100	ZZ.9	KK.9	8.7.8 8.7.8	00	3	23	3	
Information	~ 585 Hz	L = 73 cm	R ~ 2.7										~ 585 Hz	= 73 cm	PR ~ 2.9									1, 100	202 HZ	100	.°.									

III DI III DI III	>		Та 🌼 🧽	9	Hed	lessin	Ta	> No.		Rs
	Uncert	uncert	uncert	uncert.	uncert	uncert - unc	uncart	uncert	uncert	uncort
585 Hz	0.1607	11.436	7.2708	7.2708	1.7365	15.478	0.1686	0.4545	0.002	0.4848
73 cm	0.1592	11.406	7.0787	7.0787	_	15.273		0.4542	0.002	0.4845
	0.1603	11.447	7.2779	7.2779	1.747	15.494	0.1686	0.4552	0.005	0.4854
						15.415				0.4849
	0.1419			4.9571	1.7122	12.128	0.1687	0.4539	0.005	0.4842
	0.1406	9.7439	4.8746	4.8746		12.059	0.1687	٠	0.002	0.484
	0.1409	9.7351	4.8702	4.8702		12.047	0.1687			0.4848
						12.078				0.4843
	0.1305	8.5373	3.7227	3.7227	1.7261	10.178	0.1686			0.4848
	0.1297	8.5325	3.6747	3.6747		10.138	0.1685	0		0.4851
	0.1297	8.5325	3.6747	3.6747	1.7201	10.138	0.1685			0.4854
						10.152				0.4851
285 Hz	0.1564	11.541	7.2493	7.2493		15.547	0.1696		0.002	0.4573
73 cm	0.1568	11.527	7.2399	7.2399	1.8214	15.525	0.1695	0.4247		0.4573
	0.1563	11,543	7.2504	7.2504	1,8373	15.55	0.1695		0.005	0.4565
						15.541				0.457
1	0.1409	9.319	5.0385	5.0385	1.9793	11.898	0.1693		0.002	0.4565
1	0.14	9.3043	4.9487	4.9487	1.9619	11.807	0.1692		0.005	0.4577
-	0.1392	9.2602	4.808	4.808	1.9098	11.647	0.1692			0.4567
						11.784				0.457
	0.1289	8.6128	3.7792	3.7792	1.8212	10.299	0.1691	0.4247	0.002	0.4571
-	0.129	8.1661	3.644	3.644	1.8935	9.841	0.1691	0.4264	0.005	0.4587
	0.1289	8,171	3.6461	3.6461	1.9	9.8478	0.1689	0.425	0.002	0.4573
						9.9961				0.4577
~ 585 Hz	0.1544	10.848	7.0581	7.0581	2.0587	14.886	0.169	0.3998	0.002	0.434
73 cm	0.1545	10.841	7.0532	7.0532	2.051	14.874	0.169	0.3998	0.002	0.434
	0.154	10.818	6.9586	6.9586	2.0281	14.765	0.169	0.4015	0.002	0.4356
						14.842				0.4345
1	0.1365	9.2967	4.7147	4.7147	1.9529	11.607	0.1689	0.4	0.002	0.4342
	0.1372	9.3693	4.9036	4.9036	2.0387	11.834	0.1689	0.3998	0.00	0.434
ĺ	0.1398	9.3382	5.0075	5.0075	2.002	11.89	0.1689	0.4	0.002	0.4342
						11.777				0.4341
	0.1287	8.2521		3.7733	2.0084	10.031	0.1688	0.4012	0.002	0.4353
1	0.1284	8.2499	·	3.7491	2.0055	10.011	0.1688	0.4005	0.002	0.4346
i	0.1282	8.2188	3.6897	3.6897	1.9639	9.9323	0.1688	0.4024	0.002	0.4364
-						9.9913				0 4354

SPL (dB)	150.6	150.6	150.6		1507	150.7	150.7	3	1507	150.7	150.7		151.4	151 4	1514) !	151.4		1514		1514		51.4		152.2	152.2	152.2			152.2			1522	152.2	152.2	11
PR%	0.9508	0.9508	0.9508		0 9546	0.05.5	0 9546		0 9565	0.9565	0.9546		1.0376	1 0376	1.0376		1.0376	1.0357	1 0357		1 0357	1 0357	1.0357		1.1357	1.1357	1.1357		1.1338	1.1338	1.1357		1 1357	1.1357	1.1319) :
BS	0.812	0.812	0.812	0.812	0.816		7.1	31.0	0.823	0.821	0.819	0.821	0.967		0.967		0.969	0.965	0.965	0.966	0.966	0.962	0.963	0.964	1.158	1.158	1.158	1.158	1.154	1.155	1.159	1.156	1	1.162		
As*As	¥	_	0.107	-	L	0.168	0 169	0.166	0.219	0.216	214	0.216		0.079	0.075	07		117	0.116	~ (C)	0.149	0.156	0.156	0.154	0.055	0.056	0.055	0.056	0.078	0.078	0.08	0.079	0.11	0.105	0.107	0.107
(2AA/ft)	1946	1946	1946	1946	1948	1948	1948	1948	1948	1949	1949	1949	1949	1949	1949	1949	1949	1949	1949	1949	1949	1951	1951	1950	1951	1951	1951	1951	1951	1951	1951	1951	1951	1951		- 22
(**	3057	3057	3057	3057	3059	3059	3059	3059	3059	3062	3062	3061	3062	3062	3062	3062	3062	3062	3062	4.83	1	3064		3064	3064	3064	3064	3064	3064	3064	3064	3064	3064	3064	3067	3065
Э. Э.	0	0	0	0.383		0.385	<u> </u>	0.385	0.385	0.385	38	0.385	0.418	0.418	4	0,418	0.418	0.417	0.417	0.418	0.418	0.417	0.417	0.417	0.458	0.458	0.458	0.458	0.457	0.457	0.458	0.457	0.458	0.458	0.456	0,457
3	0.122	0	0.122	0.122	0.1		0	0	0		0.122	0.123	0.133		0.133	0.133	0.1	0.133	0.1	0.133	0.1	0	0.1	0.133	0	0	0	0.146		0		0,145	0.146	0		0,146
X		0		0,056	0	0	0.056	0	0	0	0	0		0	0	0.056	0	0.056	0	0.056	0	0.056	0	0	0	0.056	0	0.056		0	0	0.056	0.056	0.056	0.056	0.056
Rs Nu	9 9.23	6	6		2 9	4	2:0	2 9.8		တ်	8	4	5 9.61	2	2	5 8.95	-	_	4 10	5 10			-		10.3			1 10.2	9 10.8	_	1 10.5	4 10.7	11.1	3 11.5	Ξ	11.3
h F m/2K	22	66	.88 44	.36 44	98 4	.88 45	.57 45.	50,14 45.	69 45.5	49.54 45.		47	21 53.	59	63	81 53,	.31 53.6			2599				53	52.86 64.1			200			.84 64.	5000	97 64.3		63	91 64.
Rs Wit	45.5 47	10	2	47.	S		5.8 49.	50	6.1 48	46.1 49	6		54.2 49		CV.	45	5	54.1 51	1 51	. Paka	_	77		0.000.00	65 52.		-	43	64.8 55.	- !	65 53.	54	5.1 56.97	-	6 58	57.
Mig. V (mV)	0.504 4				506 4	507	506 4		202	203	506	- 1	0.55 5				0.55	649	549		548	549	549		.602	,602	709		conde	601	,602		802	602	0,6 64	
A 100 CONT.	1199 (000	1199 C		1200 0,	94.			* *	1201 0.	380		1201	1201	1201		1201	1201 0	1201 0		940	1202 0,	-500	828	1202 0.	1202 1000 1000	100		1202 0.		1202 0.	Ø	1202 0	1202 0	1203	
Volt S			6,87		8.31	8.64	8.59		66'6	10,09	10.15		7.01	6.79	6.6		8.8	8.96	8.88		10.25		10,6		\$ 8 \$ 1	60° v	6.83		8.91		9.03		10.81	10.55	10,63	
	8 0.06	A	8000		7 0.08	o			0	0.09	0	10000	o i		0		3 0,08			Sec. 15.	4044	2 0.1	12000	- (0.0	> (٥.	. '	0.08	٠,	o .			0.1		
T3 IS-T	27.5 6.8	7.7 6.99		-	1.6 10.	2.4 11.	32.4 11.4		36.5 15.2		3.7 15		29.8 8.2				33.1 11.3					37.1 15.2			29.9 7.88	00 00			32.9 10.9		11.4	_1.		.7 15.3		_
Ta Ttc Tsfe-fa (C) (C) (C) (C)	27.9 2	7 6	28.3		32.3 3				37.4 36			- 1	30.3 28				33.8					38,2 37			\$0.4 20.4	0.00	S'2			35 35				38.8 37.7		
	20.7			97.	1000	~ ~			21.3			A	21.5		1000	232	21,8 21,8	61 Y 1990	2000	600 10	25,900	21.9		900	u c	777	0.770				alleria.	- C		4.77 4.70		
Information	~ 1200 Hz	00 00	FR ~ 0.9							-			2 1 200 HZ	mo co = 7	PH ~ 1.0							1		-17000+	= 1200 FIZ	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0										dini

	(dB)	5 156.1	156.1	156.1		156.1		1		156.1	: "	3 156.1		156.6	156.6			5 156.6	5 156.6	5 156.6		3 156.6	3 156.6	;		5 157.1		157.1		-	•	157	:		157	
PR %		1.7885	1.7904	1.7904		1.7885	1.7885	1.7885		1.7866	1.7866	1.7866		1.8941	1.8904	1.8866		1.8885	1.8885	1.8885		1.8866	1.8866	1.8866		1.9885	1.9809	1.9885		1.9866	1.9809	1.9828		1.9771	1.979	1.979
	γ	2.867	2.876	2.873	2.872	2.87	:	CA	2.871	2.866		2.866	2.866	3.222	3.209	3.196	3,209	3.206	3.203	3.206	3.205	3.202	3.202	3.202	3.202				3.545		3.527	3.534	3.536	3.52	3.527	3.527
Rs*Rs		0.008	0.008	0.008	0.008	0.012			0.012	0.017		0.017	0.017	0.007	0.007	0.007	0.007		0.01		0.01	0.014	0.014	0.014	0.014				0.006		0.008	0.008	0.008	0.011	0.011	0.011
Α	$2\Lambda\Lambda\pi$	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953	1953
V*V	Ĭ	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	!		3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067	3067
Š	(TE)	0.72	0.721	0.721	0,721	0.72	0.72	0.72	0.72	0.72	0.72	0.72	0.72	0.763	0.762	0.76	0.762	0.761	0.761	0.761	0.761	0.76			0.76	0.801	0.798	0.801	0.8	0.8	0.798	0.799	0.799	0.797	0.798	0.798
ω		0	0.23	0.229	0.229	0.229			0.229	0.229	0.229	0.229	0.229	0.243		0.242	0.242		0.242	0.242	0.242	0.242	0.242		0.242			-	0.255		0.254		0.254	0.254	0.254	0.254
×		0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056	0.056		<u> </u>	:	0.056				0.056		0.056	0.056	0.056	950'0	0.056	0.056
a Nu		1	9 11.8		9 11.7	9 12.9			9 12.9	9 12.5	9 12.5	9 12.5	3 12.5	3 11.1	11.1	7 13.1	3 11.8		7 12.8	3 12.7	7 12.8	7 13.5	7 13.4	_	7 13.6			_	3 13,4		_	•	3 12.9	5 13.8	5 13.8	5 13.9
h As	√2K	159	33 159	71 159	159	17 159	26 159	99 159	159	97 159	159	159	3 159	35 178	74 178	33 177	24 178	92 178	39 17,	178	17.	33 17,	34 17.	177	13 17,	ļ		_	58 196	33 196	21 195	55 196	66 196	13 195	195	195
Rs	W/m^2K	59.49	52 60.63	2 59.71	59.94	1 66.17	11 66.26		66.04	1 63.97	1 64.07	1 64.06	64.03	11 57.05	0 56.74	-	60.24			0 65.14	65,46	69.03		10 70.44	69.43				68,58		198 66.21	65	9			8 70.96
	(mV)		49 162	49 162			48 161	1		l	47 161	47 161		.004 181	.002 180	1 179		.001 18	l.n.	.001 180		1 180	1 180	180				54 199		_		051 198		.048 197		.049 198
			1203 0.949	1203 0.949		203 0.948	03 0.948			203 0.947		03 0.947		203 1.0		8				1203 1.0)33	33	33		•		03 1.054	A Southern			1203 1.0			•	1203 1.0
	(V) (Hz)	3.36	18.18.3	7.63 12(9.75 12(9.68 1203	9.85 1203		7	11.55 1203	1.62 1203			7.92 1203	8.08 1203		10.05 1203		10.18 120		11.88 1203	11.92 1203	11.96 1203		8.16 12(8.19 1203	8.15 1203			1	10.24 12(12.03 1203	12.11 1203	
Cur	(A)			0.07		10	60.0	0:00			0.1					0.08		3 10 3	0.09 1			0,11		0.11				0.08		180 "	0.09					0.11
Tsfs-Ta	<u>(</u>)	7.45	7.35	7.36	003	10.9	10.8	11.1		14	14.8	4		8.03	8.03	7.94		11.3	11.5	11.6		15.6	15.7	15.4		7.83	7.83	7.84		11.3	11.4	11.6	X3-12	15.5	15.5	15.3
		3 29.8	3 29.7	2 29.7		2 33.3	1 33.2	5 33.6			5 37.3			1 30.5	1 30.5	1 30.4		(C)	9 34	1 34.2		5 38.3	7 38.4	4 38.1		1 30.4			2	3 33.9	34	1 34.2		3 38.3	7 38.3	\$ 38.1
	(O)	22.3 30.3	2.4 30.0	2.3 30.2		2.4 34.5	2.4 34.	22.5 34.5		2.5 38.4	22.5 38,5	2.5 38.6		22.5 31.1	2.5 31.1	2.5 31.		2.6 34.8	22.5 34.9	2.6 35.		2.7 39.0	22.7 39.7	2.7 39.4		22.6 31.1	2.6 31.1	2.6 31.		22.6 34.8	2.6 34.5	2.6 35.		2.8 39.6	22.8 39.7	2.8 39.4
Information		7	r.800	100 500		Ñ	ત્યું	ત્યું		Š	Ň	7	Suid	2		97 : 1		તો.	તો .	N.		2	તૉ	2		f ~ 1200 Hz 23				8	ત્યું .	Ñ		2	હ્યું	Ň
Infor		f ~ 120	L = 65	PR ~ 1.8										f - 121	L = 65 cm	PR~										f ~ 12	L = 65	PR~:								

						overall				overall
Information	>	_	Та	To	Req	Nusselt	Ta	Mic V	_	Вŝ
	uncert	uncert	uncert	uncert	uncert	uncert	uncert	uncert	uncert	uncert
~ 1200 Hz	0.1948	15.8	9	6.3609	0.6194	18.193	0.1692	0.824	0.005	0.8412
_ = 65 cm	0.197	15.826	6.607	6.607	0.6348	18.39	0.1692	0.824	0.005	0.8412
PR ~ 1.5	0.1961	15.818	6.5233	6.5233	0.6302	18.324	0.1692	0.824	0.005	0.8412
						18.302				0.8412
	0.167	12.515	4.4541	4.4541	0.7029	14.03	0.1692	0.824	0.005	0.8412
	0.1675	12.519	4.4926	4.4926	0.706	14.058	0.1692	0.824	0.005	0.8412
	0.1661	12.507	4.3786	4.3786	0.6961	13.974	0.1692	0.824	0.005	0.8412
			ī			14.021				0.8412
	0.1511	11,296	3.3619	3.3619	0.6842	12.276	0.1691	0.824	0.002	0.8411
i	0.1506	11.294		3.3406	0.6829	12.263	0.1691	0.824	0.002	0.8411
	0.152	11.299	3.4054	3.4054	0.6867	12.303	0.1691	0.824	0.002	
						12.28				0.8411
~ 1200 Hz	0.1928	15.818	6.3678	6.3678	0.6299	18.213	0.1692	0.7765	0.005	0.7947
- = 65 cm	0.1929	15.831	6.45	6.45	0.638	18.283	0.1692	0.7765	0.005	0.7947
PR ~ 1.6	0.1956	15.853	6.7013	6.7013	0.6517	18.483	0.1692	0.7765	0.002	0.7947
						18.326			1	0.7947
	0.1666	12.529		4.496	0.7133	14.069	0.1691	0.7756	0.005	0.7938
	0.1663	12.532		4.4971	0.7157	14.072	0.1691	0.7765	0.002	0.7947
	0.165	12.526	4.4212	4.4212	0.7109	14.019	0.1691	0.7765	0.002	0.7947
						14.053				0.7944
	0.1495	11.3		3.3216	0.6876	12.258	0,1691	0.7765	0.002	0.7947
	0.149	11.298			0.6863	12.245	0.1691	0.7765	0.005	0.7947
	0.1502	11.304	3.3644	3.3644	0.6915	12.285	0.1691	0.7756	0.002	0.7938
						12.263				0.7944
~ 1200 Hz	0.1933	15.843		- 1	0.6455		0.1691	0.7309	0.002	0.7502
	0.1933	15.843	6.5336	9	0.6455		0.1692	0.7293	0.005	0.7486
PR ~ 1.7	0.1943	15.85		6.617	0.6494		0.1692	0.7293	0.002	0.7486
						18.375				0.7492
	0.1663	12.541				14.108	0.1691	0.7333	0.002	0.7526
	0.1652	12.533				14.05	0.1691	0.7333	0.002	0.7526
	0.1657	12.538	4.4993	4.4993	0.7205	14.08	0.1691	0.7341	0.002	0.7534
						14.079				0.7528
	0.1489	11.311		3.3456	0.6975	12.282	0.1691	0.7358	0.00	0.755
	0.1498	11.315		3.3889	0.7009	12.309	0.1691	0.7358	0.00	0.755
1	0.1489	11.312	3.3458	3.3458	0.6981	12.283	0.1691	0.735	0.005	0.7542
						12.291				0.7547

Freq	Cur Volt Fred	Freq		20000000000000000000000000000000000000	Rs		8	3	×	မ	Ā.	V*V	60.	Rs*Rs	Bs.	PR %	SPL
			(V) (HZ)		000000	W/m/2K		388 L		-		۲	TANT.		4		(dB)
7.86		~			115	55.36	- 1	10.8		_ :	0.609	3067	1953	0.017		1.5111	154.7
7.57		0000	Such		115	56.74	113	-		0.194	0.609	3067	1953	0.017		1.5111	154.7
30.1 7.66 0.07			7,5 1203	0,801	115	56.32	113				0.609	3067	1953	0.017	:	1.5111	154.7
					30.30	56,14	113	11 (0.056	0.194	0.609	3067	1953	0.017	2.049		
-	11.00		9.53 1203	8.39	115	62.83	113			0.194	609.0	3067	1953	0.025	2.049	1.5111	154.7
gille.	gille.			1 300	115	63.11	113			0.194	609.0	3067	1953	0.025		1.5111	154.7
11.4	antin		9,6 1203	0.801	115	62.22	113	12.2		0.194	609.0	3067	1953	0.025	2.049	1.5111	154.7
		3598				62.72	113	20.00	0.056	0.194	0.609	3067	1953	0.025	2.049		
14.9 0.1	0.1		11.06 1203	0.801	115	61.15	114	1		0.194	609.0	2906	1953	0.033	2.051	1.5111	154.7
37.5 15 0.1 1	0.1	•	90000		115	61.04	114			0.194	0.609	3067	1953	0.033		1.5111	154.7
14.7 0.1	0.1		10.96 1203	0.801	115	61.38	114	12	0.056	0.194	0.609	3067	1953	0.032	2.051	1.5111	154.7
						61.19	114	72	0.056	0.194	0.609	3067	1953	0.033	2.051		
30.3 7.85 0.07	3380		7.68 1203	0.85	130	56.3	128	Ξ	0.056	0.206	0.646	3067	1953	0.014		1.6036	155.2
7.75			7.68 1203	0,85	130	57.03	128	<u>-</u>	0.056	0.206	0.646	3067	1953	0.013	2.307	1.6036	155.2
29.9 7.46 0.07			7.55 1203	0.85	130	58.25	128	11.4		0.206	0.646	3067	1953	0.013	2.307	1.6036	155.2
						57.19	128	1.2	0.056	0.206	0.646	3067	1953	0.013	2.307		
000000	000000		9.58 1203	0.851	130	63.75	128		1	0.206	0.647	3067	1953	0.019	2.315	1.6055	155.2
-	8880		9.61 1203	0.85	130	63.97	128	12.5	0.056	0.206	0.646	3067	1953	0.019	2.309	1.6036	155.2
33.8 11.3 0.09		300	9.71 1203	0.85	130	63.54	128		0.056	0.206	0.646	3067	1953	0.02	2.309	1.6036	155.2
	ı					63.75	128	12.5	0.056	0.206	0.646	3067	1953	0.019	2,311		
15.1 0.1	0.1		11.25 1203	0.85	130	61.45	128	12	0.056	0.206	0.646	3067	1953	0.026	2.311	1.6036	155.2
37.7 15.1 0.1	0.1	103	11.3 1203	30.77%	130	61.34	128	12		0.206	0.646	3067	1953			1.6036	
14.9 0.1	0.1		11.17 1203	O	130	61.81	128	12.1		0.206	0.647	3067	1953	0.025		1.6055	155.2
						61.53	128	12	0.056	0.206	0,646	3067	1953	0.026	2,313		
7.65	2.00		7.67 1203	0.903	146	69.75	144			0.218	0.686	3067	1953	0.01			
30.6 30.1 7.65 0.07			7,67 1203	0.905	147	69.75	145	6.1	0.056	0.219	0.688	3067	1953	0.01	2.615	Τ,	-
7.56		SH4	7.62 1203	0.302	147	58.05	145			0.219	0.688	3067	1953	0.01	_	1.7074	155.7
						57.81	145	1.3	0.056	0.219	0.687	3067	1953	0.01	2.612		
Ξ			9.62 1203		145	64.62	143	2.6	0.056	0.218	0.684	3067	1953		2.589	_	
11.2			9.7 1203	6.0	145	64.04	143	2		0.218	0.684	3067	1953	0.015		1.6979	-
33.6 11.1 0.09			9.67 1203	0.899	145	64.4	143	2.6	0.056	0.217	0.683	3067	1953			1.696	155.7
						64.35	143	2.6	0.056	0.218	0.684	3067	1953	0.015	2.587		
10	10	1.00	11.33 1203	1 0.897	144	62.34	143	 	0.056	0.217	0.682	3067	1953	0.021	2.574	1.6923) .
14.8 0.1			11.24 1203		144	62.64		12.2	0.056	0.217	0.682	3067	1953		ca	1.6923	
			11.34 1203	0.898	145	62.4	143		0.056	0.217	0.683	3067	1953	0.021	2.58	1.6941	155.7
						((010	1	144	1000	CLC		١		

						overall				overall
Information V		_	_a	2	Req	Nusselt	Ta E	MicV		Rs
	uncert	uncert	uncert	uncert	uncert	uncert	uncert	uncert	uncert	uncert
f ~ 1200 Hz	0.1971		6,1921	6.1921	0.587	17.805	0.1692	1.0092	0.005	1.0233
L = 65 cm	0.198	15.451	6.2669	6.2669		17.823	0.1692	1.0092	0.005	1.0233
PR ~ 1.2	0.1933	14.834	6.6083	6.6083	0.6367	17.545	0.1692	1.0092	0.002	1.0233
						17.724		•	:	1.0233
	0.165	13.524		4.3747		14.885	0.1692	1.0092	0.005	1.0233
	0.1673		4	4.2601	0.6639	13.766	0.1691	1.0092	0.005	1.0232
	0.1674	12.392	4.2242	4.2242	0.6582	13.773	0.1691	1.0076	0.002	1.0217
						14.142				1.0227
	0.1524		က	3.1868		12.236	0.169	1.0061	0.002	1.0202
	0.1509			3.23	0.6532	12.103	0.1689	1.0061	0.005	1.0202
	0.1502	11.103	3.2517	3.2517	0.6593	12.037	0.1689	1.0061	0.002	1.0202
						12.125			:	1.0202
~ 1200 Hz	0.1986	-	9	6.5148		18.299	0.1696	0.9375	0.002	0.9527
L = 65 cm	0.1995		9	6.5118	0	18.291	0.1696	0.9375	0.002	0.9527
PR ~ 1.3	0.1962	15.789	6.3563	6.3563	0.6124	18.18		0.9375	0.002	0.9527
						18.257				0.9527
	0.1675	12.484	4.3354	4.3354	0.6785	13.926	0.1695	0.9362	0.002	0.9514
	0.1679	12.473	4.2968	4.2968		13.891	0.1695	0.9362	0.005	0.9514
	0.168	12.48	4.334	4.334	0.6754	13.921	0.1695	0.9388	0.002	0.954
						13.913				0.9523
	0.1521	-	က	3.2701		12.193	0.1693	0.9388	0.002	0.954
	0.1512		:	3.23		12.17	0.1692	0.9375	0.002	0.9527
	0.1508	11.252	3.1895	3.1895	0.6467	12.141	0.1692	0.9348	0.002	0.95
						12.168				0.9522
- 1200 Hz	0.1963	15.971	,	6.4381	0.6203	18.395	0.1692	0.8812	0.002	0.8973
œ۱	0.1973	15.872	9	6.5191	0.6239	18.367	0.1692	0.88	0.002	0.8961
PR ~ 1.4	0.1955	15.76	6.2782	6.2782	0.6073	18.1	0.1693	0.88	0.005	0.8961
						18.287				0.8965
	0.1667	12.492		4.3381		13.935	0.1693	0.8788	0.002	0.895
	0.166	12.36	4	4.3029	0	13.795	0.1693	0.8788	0.005	0.895
	0.1671	12.637	4.2991	4.2991		14.041	0.1692	0.8788	0.002	0.895
						13.924		_		0.895
:	0.1508	11.28	- :	3.2523			0.1692	0.88	0.002	0.8961
	0.1525	11.516	က	3.3807	0.6774	↽.	0.1691	0.88	0.002	0.8961
	0.1508	11.178	3.2954	3.2954			0.1691	0.88	0.005	0.8961
						12.273				O ROR1

Experiment									,	500							ij			
Information			Ts fs-T	a Cur		Freq	wio V	Rs	ے	Яз	32	×	ట	ð	٧*٧	6	As*Rs	Bs	PR%	SPL
	<u>(</u>)		(O) (O)	_	3	(Hz)	(mV)	ý	W/m^2K					(au)	(2	$(2\Lambda\Lambda/\pi)$		Y		(dB)
f ~ 1200 Hz	22.4	31 3				1203		-		75.6 1	10.3 0.0	0.056 0.1	158 0.	764.	3067	1953	0.04	1.366	1.2338	152.9
L = 65 cm	22.4	30,9 3		8 0.07		1203			52.74 7				0.158	0.497	3067	1953	0.04	1.366	1.2338	152.9
PR ~ 1.2	22.4	- 1	30 7.5		7.01	1203	0.654	7.97						0.497	3067	1953	0.038	1.366	1.2338	152.9
								,7 ′	900	75.6 1	10.6 0.0	0.056 0.	0.158 C	0.497	3067	1953	0.039	1,366		
	22.4	34.6	33.8 11.	4 0.08	9.21	1203	0.654				0.6 0.0		0.158 0	0.497	3067	1953	0.057	1.366	1.2338	
	22.5	35.1	4.2 11.	7 0.09		1203			59.34 7	75.7	1.6 0.0	0.056 0.	0.158	0.497	3067	1953	0.058	1.367	1.2338	
	22.5	35.2	4.3 11.	ance,	9:36	1203	0.655	77		75.9 1		0.056 0.	0.158 0	0.498	3067	1953	0.058	1.371	1.2357	152.9
									57.52 7	75.8 1	11.2 0.0	0.056 0.	0.158 C	0.497	3067	1953	0.058	1,368		
	22.7	39.5	15.	7		12.00		77.3		76.3 1	1.2 0.			0.499	3067	1953	0.075	1.378	1.2376	
	22.8	39,4		5 0.1						•	1.4 0.0			0.499	3067	1953	0.074	1.379	1.2376	152.9
	22.8	39.3	15	4	10.86	1203	0.656	77.3	58.93 7	76.4	11.5 0.0	0.056 0.	0.159 0	0.499	3067	1953	0.074	1.379	1.2376	152.9
				Section 4				70.50	58.13 7	76.4 1	11.4 0.0	0.056 0.	0.159 C	0.499	3067	1953	0.074	1.379		
f ~ 1200 Hz	21.6	29.8		70.0 7		1202		88.8		87.3 1	10.8 0.0	0.056 (0.17 C	0.535	3064	1951	0.03	1.577	1.3282	153.6
L = 65 cm	21.6	29.8	29.3 7.6	æ	7.31	1202				87.3		0.056	0.17	0.535	3064	1951	0.03	1.577	1.3282	153.6
PR ~ 1.3	21.7	30.1	7.8	7		1202	0.704	88.8		87.4 1				0.535	3064	1951	0.03	1.579	1.3282	153.6
		2 10						-	54.91 E	87.4 1	10.7 0,0	0,056 (0.17 C	0.535	3064	1951	0.03	1,578		
	21.9	34.3	Ξ	40000		1203				87.5 1	1.8 0.1			0.535	3067	1953	0.044	1.58	1.33	153.6
	21.9	34,4		6 0.09	M	1203		89			11.7 0.0			.535	3067	1953	0.044	1.58	1.33	153.6
	21.9	34.3	Ξ		9.41	1203	0.703		96.09	87 1	11.8 0.0	0.056		0.534	3067	1953	0.044	1.571	1.3263	153.
						Sec. 13.7		\$.	- 1	87,3 1	1.8 0.	0.056 (0.17	0.535	3067	1953	0.044	1.577		
	22.2	38.6	2	က		1203			58.57 8	87.2 1	1.4 0.0	0.056	0.17 0	0.534	3067	1953	0.058	1.575	1.3263	153.5
	22.3	38,9	5	5 0.1	10.99	1203	. :	88.9						0.535	3067	1953	0.058	1.581	1.3282	153.6
	22.3	39.1	38 15.			1203	0.706			T				0.536	2908	1953	0.058	1.59	1.3319	153.
								~′	58.25	87.6 1	11.4 0.0	0.056 (- 33	0.535	3067	1953	0.058	1,582		
f ~ 1200 Hz	22.3	30.6 30.1	0.1 7.77	7 0.07			0.749	-		:				0.569	3067	1953	0.023	1.79	1.413	154.1
L = 65 cm	22.3	30.5	30 7.6		7.43		0.75							0.57	3067	1953	0.022	1.795	1,4149	154.
PR ~ 1.4	22.2	30.7 3	30.2 7.9			1203	0.75	10						0.57	3067	1953	0.023	1.793	1.4149	154.
									11.1	99.3	- L	- 1	0.181	0.57	3067	1953	0.023	1.792		
	22.2	34.6		16.00		1203	0.751			9.66	12 0.0			0.57	3067	1953	0.033	1.798	1.4168	-
	22.2	34.7	Ŧ.	60'0 9	9.59	1203	0.751	101				0.056 0.		0.57	3067	1953	0.034	1.798	1.4168	154.
	22.3	34.8	Ξ			1203	0,751			_	1.8 0.0			0.571	3067	1953	0.033	1.799	1.4168	154.
									60,86	99.6	11.9 0.0	0.056 0.	0.182	0.57	3067	1953	0.033	1.798		
	22.4	38.9	37.8 15.		11.05	1203	0.75			99.5 1	1.5 0.0	0.056 0.	0.181	0.57	3067	1953	0.044	1.796	1.4149	
	22.5	38,4	37.3 14.8	8 0.1	10,89		0.75					:	0.181	0.57	3067	1953	0.042	1.798	1.4149	-
	22.5	38.8	7.7 15.		11.06	1203	0.75	101		9			0.181	0.57	3067	1953	0.043	1.798	1.4149	154.
			_						59.86	99.5	11.7 0.0	0.056 0.	0.181	0.57	3067	1953	0,043	1.797		

EADEIIII						overall				overall
Information V	>	-	Ta	72	Red	Nusselt	Ta	Micv		Вŝ
	uncert	uncert	uncert	uncert	uncert	uncert	uncert	uncert	uncert	uncert
~ 1200 Hz	0.2155				0.5283	20.943	0.1702	1.3095	0.005	1.3205
. = 65 cm	0.2118				0			1.3095		1.3205
PR ~ 0.9	0.2072	18.185	7.0623	7.0623	0.5356	20.756	0.1701	1.3095	0.005	1.3205
						20.833				1.3205
	0.1811	13.822		4.6632	0	15.326		1.3043		1.3154
	0.176	13.804	4.3877		0.558	15.146	0.17	1.3018	0.002	1.3128
	0.1766	13.799	4.3862	4.3862	0.5546	15.14	0.17	1.3043	0.005	1.3154
						-				1.3145
	0.1592		3.2931	3.2931	0.5448	13.176	0.1698	1.3018	0.005	1.3128
	0.1583	-	3,3169	3.3169	0.5542	13.199	0.1697	1.3018	0.002	1.3128
	0.1578	12.333	3.3403	3.3403	0.5614	13.22	0.1696	1.3043	0.002	1.3153
						13.198				1,3136
~ 1200 Hz	0.2044		- 1			17.911	0.1696	-	0.005	-
= 65 cm	0.2079	18.094		6.5064		20.307	0.1696	1.2	0.002	•
PR ~ 1.0	0.2127		6.8528		0.4993	20.553	0.1696	1.2	0.005	1,2119
						19.59				1.2119
	0.174	13.827	4.4317	4.4317	0.5741	15.193	0.1695	1.2	0.002	1.2119
	0.1719	13.828	4.3595		0.575	-		1.2022		1.2141
	0.173	13.828	4.3953	4.3953	0.5745	_	0.1695	1.2022		1.2141
						15.172				1.2134
	0.157			3.4107		_		1.2022		1.2141
:	0.1554		က		0.6349	_		1.2022	0.002	
:	0.1545	11.241	3.2638	3.2638		-:	0.1694	1.2022		
						12.541				1.2141
~ 1200 Hz	0.199	15.755	6.3425	9	0.5914	18.14	_	1.0963	0.005	1.1094
ဖွ	0.201	15.733			0.578	18.062		1.0963	0.005	1.1094
PR ~ 1.1	0.2049	15.734	6.4104	6.4104	0.5788	18.169	0.1694	1.0963	0.005	1.1094
						18.124				1.1094
	0.1677	13.892	-	4.5667	0.6191	15.333	0.1694	1.0982	0.005	1.1112
	0.1662	13.888		4		15.283	0.1693	1.0982	0.005	1.1112
:	0.1654	13.868	4.3719	4.3719	0.6024	15.196	0.1693	1.0963	0.002	1.1093
		- 1				15.271				1.1106
	0.1499		,	က				T		1.1093
	0.1501	=	က်	3.2708	0.6572			1.0963	0.002	1.1093
	0.1512	11,255	3.2482	3.2482		12.1				1.1129
***************************************	-					12.10	-			1,1105

overall Rs uncert			0.7158	0.716			0.7165		0		0		0.6788		0.6813	0.6801	0	0.6807		0.6807	0.6813			. 1		0.6509		0.6494	0.6492	0.6509	0.6503	0.6501	0.652	0.6515	0.6515
f uncert		0.002	0.002		0.002	:	0.005		i		0.002		0.00	0.002	0.002		0.002	0.002	0.002		0.002	0.002	0.002		0.005	0.005	0.002		0.002	0.005	0.005		0.005	0.002	0.005
Mia V uncert		0.6955	0.6955		0.6962		0.6962	į	0.6969	0.6969	0.6969		0.6574	0.6587	0.66		0.6593	0.6593	0.6593		0.66	0.66	0.66		0.6262	0.6286	0.6262		0.6268	0.6286	0.628		0.6298	0.6292	0.6292
Ta uncert	0.1692	0.1692	0.1692		0.1692		0.1691			0.1691			0.1691	0.1691			0.1691	0.1691	0.1691		0	o	0.169		0.1691	0.1691	0.1691		0.1691	0.1691	0.1691		0.1689	0.1689	0.1689
overall Nusselt uncert	18.51	18.596	18.576	18.561	14.158		-	14.148			1	12.318	18.127		16.676	17.641	14.059	14.003	13.979	14.013	11.386	11.373	11.428	11.396	16.763	16.769	16.762	16.765	14.063	14.04	13.986	14.03	11.416	11.406	11.446
Req uncert	0.6656	0.6784	0.668		0.7403	0.7413				0.7169			0.6382	0.6348	0.7489		0.7378	0.7304	0.7288		0.7723	0.7701	0.788		0.7666	0.7696	0.7656		0.741	0.7408	0.7334		0.7879	0.7885	0.7939
To uncert	6.7109		6.7976		4.5848	4.6245			3.394	3.3732	3.3522		6.2251	6.2229	6.2964		4.4334	4.3586	4,323		3.2117	3.1922	3.2554		6.382	6.384	6.3813		4.4349	4.3987	4.3251		3.236	3.2171	3.277
Ta uncert	6.7109	6.8048	6.7976		4.5848		4.506		3.394	3.3732	3.3522		6.2251	6.2229	6.2964		4.4334	4.3586	4.323		3.2117	3.1922	3.2554		6.382	6.384	6.3813		4.4349	4.3987	4.3251		3.236	3.2171	3.277
1 uncert	15.876	15.897	15.88		12.563	12.565	12.557		11.333	11.334	11.334		15.831	15.826	14.079		12.56	12.551	12.549		10.412	10.409	10.428		14.104	14.109	14.103		12.564	12.564	12.555		_	429	10.435
ncert	0.1931	0.1927	0.1945		0.1651	0.1659	0.1639		0.1481	0.1474	0.1469		0.188	0.1887	0.1882		0.1617	0.1609	0.1602		0.1457	0.1453	0.1453		0.1872	0.1868	0.1873		0.1614	0.1605	0.1597		0.1447	0.1442	0.1453
Experiment Information V	t~ 1200 Hz	L = 65 cm	PR ~ 1.8										f ~ 1200 Hz	65	PR ~ 1,9						1				f ~ 1200 Hz	65	PR ~ 2.0			1					

Cur Volt Freq mic V Rs h Rs h Rs (A) (A) (Hz) (mV) Mm*zh Rs (B) (a) (10.08 8.2 1203 1.104 2.18 71.63 2.15 0.08 8.3 1203 1.104 2.19 70.16 2.15 0.09 10.11 1203 1.104 2.19 66.97 2.15 0.09 10.11 1203 1.104 2.19 66.75 7.114 2.15 0.09 10.11 1203 1.104 2.19 66.75 7.114 2.15 0.09 10.11 1203 1.104 2.19 66.75 7.115 2.15 0.09 10.11 12.04 1.103 2.18 71.83 2.15 0.09 10.14 1203 1.103 2.18 71.83 2.15 0.09 10.14 1203 1.156 2.40 69.04 2.37 0.09 10.25 1203 1.156 2.40 69.04 2.37 0.09 10.25 1203 1.155 2.40 69.04 2.37 0.09 10.25 1203 1.155 2.40 69.04 2.37 0.09 10.38 1203 1.155 2.40 72.39 69.22 2.36 0.09 10.38 1203 1.155 2.40 72.39 2.1159 2.11 12.26 1203 1.155 2.40 72.39 2.11 12.26 1203 1.155 2.40 72.39 72.3 2.10 0.09 10.38 1203 1.155 2.40 72.39 72.3 2.38 0.00 6.43 72.5 0.506 75.8 49.52 75.1 0.06 6.74 72.5 0.506 75.8 49.52 75.1 0.08 8.64 72.5 0.506 75.8 50.34 75.3 2.00 0.08 8.64 72.5 0.506 75.9 50.34 75.3 2.00 0.08 8.64 72.5 0.506 75.9 50.34 75.3 20.00 0.08 8.64 72.5 0.506 75.9 50.34 75.3 20.00 0.08 8.64 72.5 0.506 75.9 50.34 75.3 20.00 0.08 8.64 72.5 0.506 75.9 50.34 75.3 20.00 0.08 8.66 72.5 0.506 75.9 50.34 75.3 20.00 0.08 8.66 72.5 0.506 75.9 50.34 75.3 20.00 0.08 8.66 72.5 0.506 75.9 50.34 75.3 20.00 0.08 8.66 72.5 0.506 75.9 50.34 75.3 20.00 0.09 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 8.86 72.5 0.506 75.9 50.34 75.3 20.00 0.00 0.00 0.00 0.00 0.00 0.00 0	1 1 1 1
Cut Volit Fig. 10 No. X F. KC Av. M. Fis. Fis. Av. M. Fis. Fis. Av. M. Fis. Fis. Av. M. Fis. Fis. Av. M. Fis. Fis. Av. M. Av. M. Fis. Fis. Av. M. Av. M. <t< th=""><th></th></t<>	
Cut Voli Freq mic V Rs	1.751
Cur Voli Freq micv Rs h	halist 4
Cour. Volt. Freq. mic.Y	0.061 0.076 0.077 0.078
Cour Voit Freq mic V Rs	1178
Cur Voll Freq micy Rs h Rs Nu X e KC (my) (A) (Hz) (my) Mrm/2K (my) 0.08 8.3 1203 1104 218 71.63 215 14 0.056 0.267 0.838 0.08 8.3 1203 1.104 218 71.63 215 13.1 0.056 0.267 0.838 0.09 10.11 1203 1.104 219 66.97 215 13.1 0.056 0.267 0.838 0.09 10.11 1203 1.104 219 66.97 215 13.1 0.056 0.267 0.838 0.09 10.25 1203 1.101 217 218 71.93 216 13 0.056 0.267 0.838 0.11 12.04 12.03 1.105 219 71.83 216 14 0.056 0.267 0.838 0.11 12.04 12.03 1.105 219 71.83 216 14 0.056 0.267 0.838 0.11 12.04 1.105 219 71.83 216 14 0.056 0.267 0.838 0.11 12.04 1.105 219 71.83 215 14 0.056 0.267 0.838 0.11 12.04 1.105 219 71.83 215 14 0.056 0.267 0.838 0.11 11.91 1204 1.105 220 71.82 235 14 0.056 0.279 0.876 0.08 8.12 1204 1.154 239 71.82 235 14.1 0.056 0.279 0.876 0.09 10.25 1204 1.154 239 71.82 235 14.1 0.056 0.279 0.876 0.09 10.25 1204 1.155 240 69.01 235 13.5 0.056 0.279 0.876 0.09 10.25 1203 1.155 240 69.01 235 13.1 0.056 0.279 0.876 0.09 10.25 1203 1.155 240 69.01 235 13.4 0.056 0.279 0.876 0.09 10.26 1203 1.155 240 69.01 236 13.5 0.056 0.289 0.876 0.09 10.36 1203 1.157 240 1.157 236 13.4 0.056 0.229 0.876 0.09 10.36 1203 1.157 240 1.157 236 13.5 0.056 0.280 0.881 0.11 12.12 12.03 1.157 240 69.01 236 13.5 0.056 0.289 0.876 0.09 10.36 1203 1.157 240 1.157 236 13.5 0.056 0.280 0.881 0.01 1.12.11 12.12 12.03 1.157 240 1.157 236 13.5 0.056 0.280 0.881 0.01 1.12.21 12.03 1.157 240 1.157 236 13.5 0.056 0.280 0.881 0.00 1.157 220 0.00 0.00 6.43 725 0.506 75.8 47.07 75.1 9.20 0.034 0.203 0.636 0.06 6.07 725 0.506 75.8 9.84 0.034 0.203 0.637 0.08 8.84 725 0.506 75.8 50.34 75.3 9.84 0.034 0.203 0.637 0.09 8.86 725 0.506 75.9 50.34 75.3 9.84 0.034 0.203 0.637 0.09 8.86 725 0.506 75.9 50.34 75.3 9.84 0.034 0.203 0.637 0.008 8.86 725 0.506 75.9 50.34 75.3 9.84 0.034 0.203 0.637 0.038 0.038 0.038 0.038 0.034 0.203 0.034 0.203 0.637 0.038 0.038 0.038 0.038 0.034 0.203 0.034 0.203 0.033 0.038 0.038 0.038 0.038 0.034 0.203 0.034 0.203 0.033 0.038 0.038 0.038 0.038 0.038 0.038 0.038 0.038 0.038 0.038 0.038 0.038 0.038 0.038 0.038 0.038 0.038 0.038 0.0	1848 1851 1851
Cur Voll Freq mic V	0.637 0.637 0.636
Cur Volt Freq micv Rs h Rs Nu X (A) (V) (Hz) (mv) Micva Micra Micva Micra Micva Micra Micva Micra Micva Micra Micva Micra Micva Micra Micr	0.203 0. 0.203 0. 0.203 0.
Cour. Voit Freq. mic.y Rs. h Rs. Nu. Nu. Nu. Nu. Nu. Nu. Nu. Nu. Nu. Nu	20 14 14 14 14 14 14 14 14 14 14 14 14 14
Cul (A) (Hz) (mb) Rs h Rs (A) (A) (Hz) (mb) Wm/2h (B)	00000
Cul Volt Freq micv Rs hww. Mm. Val 0.08 B.2 1203 1.104 218 71.63 0.08 B.3 1203 1.104 219 70.16 77.114 0.09 10.11 1203 1.104 219 66.97 0.09 10.11 1203 1.104 219 67.57 0.09 10.11 1203 1.104 219 67.57 0.09 10.11 1203 1.104 219 67.57 0.09 10.11 1203 1.104 219 77.83 0.11 12.04 1203 1.104 219 77.83 0.11 12.04 1203 1.104 219 77.83 0.09 8.12 1204 1.154 239 77.82 0.09 8.12 1204 1.154 239 77.82 0.09 10.14 1203 1.156 240 69.04 0.09 10.25 1203 1.156 240 69.04 0.09 10.25 1203 1.156 240 69.04 0.09 10.25 1203 1.156 240 69.04 0.09 10.25 1203 1.156 240 69.04 0.09 10.36 1203 1.155 240 77.83 0.09 10.25 1203 1.155 240 69.04 0.09 10.25 1203 1.155 240 77.83 0.09 10.25 1203 1.155 240 69.04 0.09 10.25 1203 1.155 240 77.83 0.09 6.43 725 0.506 75.8 49.52 0.06 6.74 725 0.506 75.8 47.83 0.08 8.64 725 0.506 75.9 50.34 0.08 8.64 725 0.506 75.9 50.34 0.08 8.64 725 0.506 75.9 50.34 0.08 8.64 725 0.506 75.9 50.34 0.08 8.64 725 0.506 75.9 50.34	75.4 9.85 75.4 9.94 75.2 9.91 75.2 9.91
Cur Voll Freq micy Rs 0.08 B.2 1203 1.104 2.18 0.08 B.3 1203 1.104 2.19 0.08 B.3 1203 1.104 2.19 0.08 B.3 1203 1.104 2.19 0.09 10.11 1203 1.105 2.19 0.09 10.11 1203 1.105 2.19 0.09 10.11 1203 1.105 2.19 0.09 10.11 12.04 1203 1.105 2.19 0.11 12.04 1204 1.155 2.39 0.09 B.12 1204 1.155 2.39 0.09 10.14 1203 1.156 2.40 0.09 10.14 1203 1.156 2.40 0.09 10.36 1203 1.156 2.40 0.09 10.36 1203 1.157 2.40 0.09 10.36 1203 1.155 2.41 0.11 12.26 1203 1.156 2.40 0.09 10.36 1203 1.156 2.40 0.09 10.36 1203 1.156 2.40 0.09 10.36 1203 1.156 2.40 0.09 10.36 1203 1.156 2.41 0.11 12.26 1203 1.156 2.41 0.11 12.21 12.03 1.159 2.41 0.06 6.43 725 0.506 75.8 0.06 6.74 725 0.506 75.9 0.08 8.86 725 0.506 75.9 0.08 8.86 725 0.506 75.9 0.08 8.86 725 0.506 75.9 0.08 8.86 725 0.506 75.9	50.4 7 51.18 7 50.89 7 50.74 7
Cur Volt Freq mia.V 0.08 B.2 1203 1.104 0.08 B.3 1203 1.104 0.08 B.3 1203 1.104 0.08 B.3 1203 1.104 0.09 10.11 1203 1.105 0.09 10.11 1203 1.105 0.09 10.11 12.04 1.105 0.09 10.11 12.04 1.105 0.09 10.14 1203 1.105 0.09 10.14 1203 1.156 0.09 10.25 1203 1.156 0.09 10.25 1203 1.156 0.09 10.36 12.03 1.156 0.09 10.36 12.03 1.156 0.09 10.36 12.03 1.157 0.11 12.26 1203 1.158 0.11 12.26 1203 1.159 0.06 6.43 725 0.506 0.06 6.43 725 0.506 0.08 8.64 725 0.506 0.08 8.66 725 0.506 0.08 8.66 725 0.506 0.08 8.66 725 0.506 0.08 8.66 725 0.506 0.08 8.66 725 0.506 0.08 8.66 725 0.506 0.08 8.66 725 0.506 0.08 8.66 725 0.506 0.08 8.66 725 0.506 0.09 0.08 8.66 725 0.506 0.09 0.09 8.66 725 0.506 0.09 8.66 725 0.506	76.2 5 76.2 5(75.9 5(
Cur Volt Freq (A) (H2) (D.08 B.2 1203 0.08 B.3 1203 0.09 10.11 1203 0.09 10.11 1203 0.09 10.11 1204 0.09 B.12 1204 0.09 B.12 1204 0.09 B.12 1204 0.09 B.12 1204 0.09 10.25 1203 0.09 10.25 1203 0.09 10.25 1203 0.09 10.25 1203 0.09 10.25 1203 0.09 10.25 1203 0.09 10.25 1203 0.09 10.25 1203 0.09 10.25 1203 0.09 10.25 1203 0.09 10.25 1203 0.09 10.25 1203 0.09 10.25 1203 0.00 0.00 6.74 725 0.00 6.74 725 0.00 8.86 725 0.00 8.26 725 0.00 8.86 725 0.00 8.	0,507 7
Cur (A) (V) 0.08 8.34 0.08 8.34 0.09 10.11 0.09 10.11 0.09 10.25 0.08 8.12 0.09 10.14 0.09 10.14 0.09 10.25 0.09 10.25 0.09 10.14 0.09 10.14 0.09 10.14 0.09 10.14 0.09 10.14 0.09 10.14 0.09 10.08 0.09 10.08 0.09 10.08 0.09 10.08 0.09 10.08 0.09 10.08 0.09 10.08 0.09 0.09 0.09 0.09 0.09 0.09	726 0 726 0 726 0
Cut 0.08 0.09 0.09 0.09 0.008 0.008 0.008 0.008 0.008 0.008 0.008 0.008 0.008 0.008 0.008 0.008 0.009 0.000	0.08
	0.09 0.09 1.000
11.1. 1.1. 1.1. 1.1. 1.1. 1.1. 1.1. 1.	14.6
30.3 29.6 30.3 29.6 30.5 29.8 30.7 30.6 29.8 34.3 33.4 33.4 33.7 30.6 29.9 30.5 29.8 30.6 29.9 30.6 20.9 30.0 20.0 20.0 20.0 20.0 20.0 20.0 20.0	36.4 36.7
T. T. T. C.	1 36.9 5 37.3 3 37.6
22.2 22.2 22.2 22.2 22.3 22.3 22.3 22.3	21.5 21.5 21.8
Experiment (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (C) (

uncen
-
14.131 6.394
12.584 4.5157
572 4.4(
_
က
10.447 3.3412
\perp
14.162 6.6426
12.605 4.5999
12.501 4.4845
-+
<u> </u>
0.482 3.3942
<u> </u>
18.168 7.151
-+
<u> </u>
4
815 4.2169
က
12.341 3.3216

J 7	6	Ω	υį,	<u>η</u> .	L	Ωį	ر.	iO.		2	3	4	_	S.	110	110	V:	I	ViV	NIC	N I	_	Q:	CV:	N:	T.	o) (וות	D :	17	7	6 ∶	7		- T	ກ . c	
	"		151.5	- !	_ *	-	151	-:		151	151	151		152	150	110	20	7	20.	22	25		152.	152.	152.		152.	32	25.	2.	152.	152.	152.		152.0		
#H.	27.70	0452	0452	2040	0474	1	.0489	0489		0471	0489	0395		1376	1 1376	1276	2:	4076	0/0	3/0	3/0		1376	1.1376	3/6	100	333	2000	255	000	338	2338	338		338	22338	000
Bs -	1	- 1	- ' +	-	Ľ			_		-		_						0 0	0 0	0 0		. .	-		!			-!-		*		2010	_ :	- 1	- ' -		
	83 —		2.033		ட	2 2		_ '	202	CU:		2.073	2.09	2.47	2.478	2 47	0 478	0 47	۰,۰۰	07.7	u T	4,4	2.48	2.481	2.40	40	7	0 0 17	0.017		5	2.91	, d	8.7	2.0	0.0	2.010
As*As	000	0.00	0.020	0.020	041		0.047	0.04	0.041	0.055	0.055	0.056	0.055	0.021	0.021	001	100	0 03	3 6	000	0000	San C	0.0	0.04	800.	5 6	0 4	0.0	200	200	000	0.022	770.	200	0000	0.020	028
- F	α		178	640	-1-	1		- 5			1178 (178 (178				178		-8	1		2 0		83		178				170		-	178		
\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	Ļ		- ' +		Ľ		+		<u> </u>		- :				_	-		L			- 100	Ľ			_ [Ľ		-	_ [L	- ! •			Ľ	-! -		
VaV	Γ	186			-	100		- 3	<u>[</u>]	_ :	182			_	1851		7	185	185	185	8	1		00.4		A B C	185	1851	8	185	Did	20.4	2 4	2 2	8	1851	185
S (S	0.696	0.696	0.696	0.696	0.698	0 800	0090		2000	0.098	0.699	0.693	0.697	0.758	0.758	0.758	0.758	7.58	7.58	0.758	0.758	750	750	0.758	0.758	0 822	0.822	0.822	0.822	822	7000	822	0.022	800	822	0.822	0.822
m	222			222		_	0.222	_ ' 2	ાં_	0.666			-	_	0.241		- 0000	-		0.241	-280.	·		0 241	- (4)	1_		262 0	150	1	0 262			4_		262 0	97.50
~	c		0	o	.034 0.			1.77	200				: L				. 3 %	1			35	1			- 1000	10	0	0	C	1-				10	10	0	0
			O	0.034	0		0.034	- 30	1		0.034	_	ுட		0.034		0.034	!		0.034	- 23333	10		0.034	_ 33.8	-		0.034	0.034	0.034	0.034	0.034			0.03		0,034
DN.	_			10.4	L		10.2		ŧĽ.		_ : £	- 3		10.5	10.3	10.4	10.4	10.1	10.2	10.4	10.3	113	4	1.4	1.4	10.8	10.7	10.7	10.7	11.9	α.	11.8	1.8	116	11.5	11.6	11.5
S 3.		90.1	:	90			90.8	13.6	1	000	0 0	200	000	10/	107	107	107	107	107	107	107	107	107	107	107	126	126	126	126	126	126	126	128	126	126	126	126
h W/m^2ł	53.12	53.2	~	53.14	51.94	51.86	52.04	51.94	57 79	577	59 57	20.02	20.04	33.72	25.69	23	53.14	1.92	2.46	53.19	52.52	57.96	8.19	58.55	58.23	55.04	54.66	4.96	54.89	0.73	60.3	60.3	60.44	9.13	8.84	59.21	90.6
As ,	90.9	91	90.9		3	9	91.7	1	-		90.1		00		80	80	S.		80			-		-	(2)	27 5	<u> </u>	27 5	വ	27 6			్లి	-		127 5	, while
mie V (mV)	0.554	554	:		555				4		551 0		600		603			80	603	ප		l	1	1		54	.654			-				a durino	654 1	- marianta	
200000000000000000000000000000000000000					0	Ö	o		lo	· C	' –		9) (<u>ن</u> د	o .	1	0	0	Ö			2000.00	5 0.603		0	0	0		3200	200	3 0.654		0	0	0	
II Freq /) (Hz)		100	7 726	1	5 726						9 726		7.0E		97/ 5		ା	100	3 726			5 726					726					726			726		
	7.1				9,05	9.1	9.1		10.6	10.7	10.89		7.9) (70°	₹.		9.12	Ö,6	8.9		10.8	10.89	10.82		7.43	7,56	7.42		9.31	9.4	9,4		10.92	10,87	10,8	
G.C.					0.08	0.08	0,08		0.1	0			70.0	100		9.0		0.08	0.08 80.0	0.08		0.1	. .	0,1		0.07	0,07	0.07		0.09	0.09	60:0			0.1	-,	
R-Ta	7.69	7.97	7.88		•)		11.6		15.2	15.3	15.3	1	7 88	200	200	0.0		11.6	4.	=		15.4	15.4	15.2		7.77	7.96	7.77	20.6		11.11	11.5	****	15.2	15.2	15	885,
	29.1	29.5	29.3	1	33	33.1	33.2		36.8	36.9	37				2000			33			i		37				29.5		•	32.8				8.9	36.8	9.9	-
	29.6	8	29.8		33.7						38,1		29.8		3 6	3		23.0			l.	i.	38.1			29.8		29.8			34				37,9		
	21.4				0.7			999	21.6							<u>:</u>	90 B W	4. 5	19.5.	· Colin			21.6		350		21.5	4000		21.5	9.	9:			21.6		
tion	escu.	my.	800,30			. 	1		·u	CU	· u	li di	naca	gano !	30,000		1	u C	V (V	9	V	CV.	ા		!			1	CN (CV.	N		N	CV (N	
Experiment Information Tie (C)	~ (S)	L = 60 cm	~ -			-							~ 725	50 cm	PR 11	:		1				-	-		2	200	L = 60 cm	7							:	:	
n e	1	1 0	Ĭ.		-	-	į		-				<i>1</i> ~ <i>j</i>	11	PA			!	-			-			,	27/~	11 0			-		-		-		:	

overall Rs uncert	-	1.2034	_	1.2034	1.2012	1.1991		_	1.2012	1.1991	1.2098	1.2034	1.1076	1.1076	1.1076	1.1076	1.1076	1.1076	1.1076	1.1076	1.1076	1.1076	1.1076	1.1076	1.0233	1.0233	1.0233	1.0233	1.0233	1.0233	-	-	1.0233	_	-	1.0233
f uncert	- 1	0.002				0.002					0.002			0.002	:				0.002		0.002		0.002				0.002	:		0.005			1	0.002		
Mic V undert		1.1913			L	1.1871								1.0945			-	_	1.0945		-	1.0945	1.0945		1.0092	1.0092	1.0092		1.0092	-	-	·			1.0092	
Ta uncert	0.1698		0.1698							0.1696				0.1698			0.1698	0.1697	0.1697				0.1696		0.1697		0.1697			0.1696			i		0.1696	
overall Nusselt uncert	18.255	18.095	18.145	18.165	15.162		15.143	15.149	12.193			12.189	18.158	18.034	18.04	18.077	15.141	-	15.271	15.202	12.174	12.177	12.204	12.185	- 3	-	18.239	18.202	13.972	. :	13.92	13.937	12.212	12.208		12.219
Red uncert	0.5943	0.5952	0.5941		0.5811	0.5802	0.5822		0.6466	0.6456	0.6553		0.601	0.5895	0.5929		0.5808	0.5869	0.595		0.6484	0.651	0.655		0.6158	0.6116	0.6149		0.6794	0.6746	0.6746		0.6615	0.6583	0.6624	
Tc uncert	6.4991	6.2703	6.3442		4.3622	4.3264	4.3273		3.2872	3.2669	3.2701		6.3488	6.1937	6.1959		4.3267	4.401	4.5167		3.248	3.2489	3.2901		6.4351	6.281	6.4346		4.4069	4.3337	4.3337		3.2923	3.2912	3.3335	
Te	6.4991	6.2703	6.3442		4.3622	4.3264	4.3273		3.2872	3.2669	3.2701		6.3488	6.1937	6.1959		4.3267	4.401	4.5167		3.248	3.2489	3.2901		6.4351	6.281	6.4346		4.4069	4.3337	4.3337		3.2923			
l uncert	15.759	15.761	15.759		13.837	13.836	13.839		11.252	11.251	11.262		15.77	15.752	15.757		13.837	13.845	13.857		11.254	11.257	11.262		15.795	15.788	15.793		12.485	12.479	12.479		11.27	11.266	11.271	
ncert	0.2034	0.1979	0.1999		0.1708	0.17	0.1697		0.1538	0.1533	0.1521		0.1984	0.1974	0.1967		0.1699	0.1708	0.1724		0.1524	0.1521	0.1528		0.1972	0.1946	0.1974		0.1693	0.1681	0,1681		0.152	0.1524	0.1531	
Expariment Information V	f ~ 725	L = 60 cm	PR ~ 1.0										f ~ 725	L = 60 cm	PR ~ 1.1										f ~ 725	L = 60 cm	PR ~ 1.2									

7 6	S C	2. 3	Į.C	;	C	2 (0	2) (C)	19	919	9	!;	1-	-			1-				1-	. —		1	7	7	7	1	7	7	7	i i	1	7	7	_
SPL	153	153	153		153	5.5	153.6			153.6	1		154	154	154) :	154	54		•	154	154	154	:	154	154	154.7		154	154.7	154		154	154.7	154	;
PR %	3282	3263	3263	1	3282	3282	3282		3282	3282	3282	1	4225	4225	4206	1:	4206	4206	4206		4206	4206	4206		5168	5168	5149		5149	5149	5149	!	5149	5149	5149	
Hs F		-:-					-	j	L	_			-	1		• ! _i	-				-		-	!	_	-	-		-	_	: -		_		_	
	3.38	3.37	3.37	3.37	1	3.38	ניס	c					3.882	: ന				3.881					3.883		ł	4.409	4	4.405	4.401	4.401	4.403	4.401	4.406	4.409	4.409	4,408
Or Rs*Rs	0.01	0.01	0.011	0.01	0.015	0.016	0.016	0.016	0.02	0.02	0.021	0.05	0.00	0.008	0.008	0.008	0.012	0.012	0.012	0.012	0.015	0.015	0.015	0.015	900.0	0.006	0.006	900.0	0.009	0.009	0.009	0.009	0.01	0.011	0.012	0,011
β F	178	178		178	-	_	178	- 2700		1178			178			- 46.0	\vdash			a - 6 - 1			178	178 (180		180	180 (_	180	180				180
	Ŀ		! -	<u>.</u>	_	-			Ŀ			10	_	_				_		- 13	L				3 1	·	-		1	<u>-</u>	0	3	-	_	-	7
V*V	185	_	1851	. 185	185	-	185		L	1851		1851	_	185	-	. [1851	:-	. 7			1851	1851	1853	185	1853	1853	185	1853	1853	1853	185	1853	185	1853
S (ag	3.885	0.884	3.884	0.884	3.885	0.885	0.885	0.885	3.885	0.885	0.885	0,885	3.948	0.948	0.947	0.948	0.947	0.948	0.948	0.948	0.948	0.948	948	948	.011	.011	.009	1.01	1.01	1.01	1.01	1.01	1.01	1.0	1.0.1	1.01
ప	282			0.281	282			282 (282 (45.4		0.302		302		0.302			드			302 (322	0.322	321	0.322	321	321	321	0.321	32.1	0.322	322	321
×	0	-		300	0	:0	0	0	0	0	0	0				0				- 34	┝			0				\$ P		4 0.321	-		4 0.32		0	0
	3	0.034		0.034	0.034		0.034	0.034	<u> </u>	0.034		0.034	0.034	0.034	0.034	0.034		0.034		0.034	0.034	0.034	0.034	0.034	0.034	0.03	0.034	0.034	0.034		0.034	0.034	0.03	0.034		0,034
Ž		11.3	11.3	11.3	12.3	12	12.1	12.2	11.9	_	11.8	11.9	11.6	11.4	11.2	7:	12.4	12.4	12.4	12.4	12.4	12.3	12.2	12.3	11.8	11.7	11.7	11.7	12.8	12.8	12.9	12.8	12.8	12.7	13.9	13.1
چ ک	145	145	145	145	145	145	146	145	146	146	146	146	167	167	167	167	167	167	167	167	167	167	167	167	190	190	189	190	189	189	190	189	190	190	190	190
h W/m^2l	57.36	58.01	7.72	57.7	63.1	61.75	62.13	62.33	60.83	61.06	60.39	92'09	59.12	58.46	57.42	58,33	63.4	63.62	63.34	63,45	63.34	63.19	62.58	63,04	60.32	9.76	59.65	59,91	65.33	5.39	65.87	65,53	9.68	65.04	0.94	77.
H3 V		146	146	Z.	147	147 6	147 6	<u></u>	147		147		169			u,	89		168	9		169		9		92		ഗ	191		191	9		95 6		P
mic V (mV)									94	4	8		i mari									,			4	2	8		:						-:	
	37		5 0.703		8.32	3 0,704	3 0,704			3 0,704				3 0,754	do	6.0			3 0,753		1.388	3 0.753	80 M		0.804	VO.4.)	W. 30			0,803	800		4000	0,803	- 300	
Freq (Hz)			726		22	726	72(726	œ				726				726			ш,	727	:08		727				Æ.	727	***	
Volt (V)	7.35	7.24	7,58		9,41	9.61	9.51		10,8	10.7	Ξ.		7,56	7.67	7.73		9,69	9.8	9.84		11,07	11,26	11.3		.7.8	7.83	7.72		8'6	6,83	9.71		-13	11.42	971	
Cur (A)	20'0	20'0	0.07		0.09	0.09	0.09		0.1	0.1	-		0.07	0.07	0.07		60'0	0.09	60'0		.; -:	.	1.		0.07) (0	207		0.09	60.0	60.0		-	- - -		
a-Ta (C)	7.38	7.18	7.56		2 2.500	11.5	saanj		14.6	14.4	15		7.36	w.		1000	w.zp		11.5		14.4	14.7	4.8	200 k	7.44		1000	333		www.				14.4		8
Tsfe-Ta (C) ** (C)	9.8	28.7	29.1		32.5		32.9			36.1					29.5	- 1		33.5				37			29.6	7.7	9.6		33.4					37		-
, 2 O	9.4	29.2	96			33.9				37.2 3			29.6				Qì Gì	34,4 3	4,5 G			38.1			30.2	χ, α Ο (α	N N		34,3					36.2		
C.a	21.5 2					21.5 3		92.5	200	21.7 3	163		21.7	0.35.50	·	2 TO 10	sayan sa	22.1 3	SW 50.			22.3			22.2				22.3 3					2 4 4 5 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6		
on	CA.		~		CV.	CU	7		C)	Cù I	~			501.4ÇE	1000000	Sureza Be		čί	ď		Ň.	ૉ	Ň			201520			N S	સં હ	7		7	ŭ ĉ	3	
Experiment Information T.a. T	~ 725	0 cm	~ 1.3						i		:	,	रा ।	c cu	PR ~ 1.4			1	-				:	,	Q d	L = 00 cm	ر. د		:	:					1	
into Pinto	1 ~ 1	9=7	Y.						-	:	:	ì	(7) ~	= 0	PR							i			62/~	1 0	ř		;	:				:		

Nidermation Ta	Experimen	-					overall				overall
725 0.1993 15.837 6.7792 6.7792 0.6418 18.525 6.60 cm 0.2017 12.54 6.6145 6.6145 0.6458 18.411 6.60 cm 0.1669 12.519 4.5303 4.5409 0.6908 13.944 0.1659 12.5 4.3409 4.3409 0.6908 13.944 0.1657 12.505 4.414 4.414 0.6951 13.995 12.50 0.1659 11.295 3.47 3.47 0.6832 12.395 0.1659 11.295 3.47 3.47 0.6832 12.395 0.1659 11.295 3.47 3.47 0.6832 12.395 0.1659 11.295 3.47 3.47 0.6832 12.395 0.1659 11.295 3.47 3.47 0.6832 12.395 0.1659 11.295 3.47 3.47 0.6832 12.395 0.1659 11.295 3.47 3.48 0.1659 11.295 3.47 3.47 0.6832 12.395 0.1651 11.295 3.47 3.48 0.6615 18.563 0.1657 12.298 0.1657 12.298 0.1657 12.298 0.1657 12.298 0.1657 12.298 0.1657 12.298 0.1657 12.298 0.1657 12.298 0.1657 12.298 0.1657 12.298 0.1657 12.298 0.1657 12.298 0.1657 12.253 13.395 0.1657 12.353 13.3372 0.1657 12.353 13.3372 0.1657 12.395 12.253 13.3372 0.1657 12.253 13.3372 0.1657 12.253 13.3372 0.1657 12.253 12.259 13.3372 0.1657 12.253 12.259 13.3372 0.1657 12.253 12.259 13.3372 0.1657 12.253 12.259 13.3372 0.1657 12.253 12.259 13.3372 0.1657 12.259 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.1657 12.259 13.3372 0.16	Intermation	V.1	-	B	To	****	Nusselt	Ta	MicV	_	Rs
Column C		UNGBU	uncert	negun		uncert	uncert	uncer	uncert	uncert	uncert
- 1.3 0.1949 15.844 6.6145 6.6145 0.6491 18.67 - 1.3 0.1949 15.844 6.6145 6.6145 0.6458 18.411 - 0.1685 12.519 4.5303 4.5303 0.706 14.082 - 0.1689 12.5 4.3409 4.3499 0.6808 13.944 - 0.1671 12.505 4.414 0.6851 13.995 - 1.4 0.1921 15.838 6.4529 6.4529 0.6428 18.237 - 0.1652 12.524 4.4205 0.6428 18.237 - 0.1652 12.524 4.4205 0.6428 18.237 - 0.1641 12.523 4.3489 0.7086 12.397 - 0.1641 13.24 3.4791 0.7086 12.395 - 0.1938 15.891 6.7173 0.7086 12.395 - 0.1938 15.891 6.7173 0.7086 12.395 - 1.5 0.1928 15.891 6.7173 0.6749 18.528 - 0.1939 15.891 6.7173 6.7122 0.6687 18.456 - 0.1644 12.552 4.4672 0.7316 14.095 - 0.1644 12.552 4.4672 0.7316 14.095 - 0.1648 11.355 3.534 3.3372 0.7318 12.429 - 0.1648 11.355 3.534 3.3372 0.7317 12.391 - 0.1648 11.355 3.534 3.3372 0.7317 12.391 - 0.1498 11.355 3.534 3.3372 0.7317 12.391 - 0.1487 11.347 3.4635 3.4635 0.7277 12.391	-	0.1993	_		:		18.525	0.1697	0.9375	0.005	0.9527
-1.3 0.1949 15.844 6.6145 6.6145 0.6458 18.411 0.1685 12.519 4.5303 4.5303 0.706 14.082 0.1659 12.5 4.3409 4.3499 0.6908 13.944 0.1671 12.505 4.414 4.414 0.6951 13.995 0.1543 11.292 3.4249 3.4249 0.6806 12.307 -1.4 0.1921 15.838 6.4529 6.4529 0.6426 18.293 0.1652 12.524 4.4205 4.4205 0.6426 18.292 0.1652 12.524 4.4205 4.4205 0.6428 18.292 0.1641 12.527 4.3856 0.7117 13.997 0.1642 12.527 4.3856 0.7107 13.995 0.1643 11.324 3.4791 0.7086 12.397 0.1644 12.552 4.4672 0.7086 12.397 0.1644 12.552 4.4672 0.7086 12.397 0.1644 12.552 4.4672 0.7081 18.538 0.1644 12.552 4.4672 0.7081 18.538 0.1644 12.552 4.4672 0.731 14.095 0.1658 15.891 6.7173 6.731 14.095 0.1659 12.552 4.4672 0.731 14.095 0.1659 13.347 3.4635 3.4635 0.7277 12.391 0.16487 11.355 3.534 3.3372 0.7317 14.1037 0.1487 11.347 3.4635 3.4635 0.7277 12.391 0.1487 11.347 3.4635 3.4635 0.7277 12.297		0.2017					18.67	0.1697	0.9388	0.005	0.954
0.1685 12.519 4.5303 4.5303 0.706 14.082 0.1659 12.55 4.3409 4.3409 0.6906 13.995 12.505 4.414 4.414 0.6951 13.995 14.007 0.1543 11.295 3.474 3.47 0.6832 12.395 14.007 0.1543 11.295 3.47 3.47 0.6832 12.395 12.298 0.1652 12.629 0.1652 12.629 0.1652 12.629 0.1652 12.629 0.1652 12.629 0.1652 12.524 4.4205 0.7093 14.016 0.1692 12.527 4.3489 0.7086 0.7097 13.995 0.1692 11.315 3.3674 3.479 0.7096 13.995 0.1492 11.323 3.479 0.7096 12.233 0.1492 11.315 3.3674 3.3674 0.7097 12.239 0.1492 11.315 3.3674 3.3674 0.7097 12.331 0.1654 12.552 4.4502 0.707 0.731 14.095 0.1654 12.552 4.4572 0.7316 14.075 0.1654 12.552 4.4672 0.7316 14.075 0.1492 11.352 4.4572 0.7316 14.095 0.1654 11.352 4.4572 0.7316 14.095 0.1654 11.352 3.43372 0.7316 12.552 4.4672 0.7316 14.073 0.1492 11.355 3.534 3.534 0.737 14.153 0.1497 11.352 3.4635 0.737 14.153 0.1497 11.352 3.4635 0.737 14.153 0.1497 11.352 3.4635 0.737 14.153 0.1497 0.1497 11.352 3.4635 0.737 14.153 0.1497 0.14	7	0.1949			6.6145	0.6458	18.411	0.1697	0.9388	0.002	0.954
0.1685 12.519 4.5303 4.5303 0.706 14.082 0.1659 12.5 4.3409 0.6906 13.944 0.1671 12.505 4.414 4.414 0.6906 13.944 0.1543 11.292 3.4249 3.4249 0.6806 12.305 0.1543 11.295 3.47 3.47 0.6832 12.335 0.1541 11.296 3.3381 3.3381 0.6656 12.253 20.1956 15.869 6.783 6.783 0.6615 18.563 60 cm 0.1934 15.838 6.4529 6.4529 0.6426 18.259 0.1652 12.527 4.3856 6.4529 0.6426 18.427 0.1652 12.524 4.4205 0.7093 14.016 0.1652 12.524 4.3489 0.7096 13.995 0.1652 12.524 4.4205 0.7096 13.995 0.1644 11.323 3.4125 3.4126 0.707 12.396											0.9536
0.1659 12.5 4.3409 4.3409 0.6906 13.944 0.1671 12.505 4.414 4.414 0.6905 13.995 0.1673 11.292 3.4249 3.4249 0.6806 12.305 0.1543 11.295 3.47 3.47 0.6832 12.335 0.1556 15.869 6.793 6.793 0.6615 18.563 60 cm 0.1936 15.838 6.4529 0.6425 18.229 7.4 0.1956 15.838 6.4529 0.6425 18.253 0.1652 12.527 4.3265 4.3205 0.703 14.016 0.1652 12.527 4.3856 0.703 18.427 0.1652 12.527 4.3265 4.3205 0.7066 13.995 0.1652 12.524 4.4205 0.7096 13.995 0.1652 12.524 4.3265 0.707 12.395 0.1652 11.323 3.4125 0.7096 12.395 0.1644		0.1685	12	_	4.5303		-	0.1697	0.9375	0.002	0.9527
0.153 11.292 3.4249 3.4249 0.6806 12.307 0.1533 11.295 3.4249 0.6806 12.307 0.1543 11.295 3.4249 0.6806 12.203 12.253 0.1556 15.869 6.793 6.793 0.6615 18.563 60 cm 0.1934 15.875 6.6201 6.6201 0.654 18.427 0.1652 12.524 4.4205 4.4205 0.7093 14.016 0.1652 12.524 4.4205 4.4205 0.7093 14.016 0.1634 12.523 4.3489 0.7086 12.395 0.1635 11.324 3.4125 3.4125 0.7006 12.395 0.1492 11.324 3.4125 3.4125 0.7006 12.297 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 13.3372 0.1498 11.355 3.534 3.3372 0.7337 14.153 0.1487 11.348 3.3372 0.7337 11.48		0.1659		<u>:</u>	4.3409	0		0	0.9375	;	4
0.1533 11.292 3.4249 3.4249 0.6606 12.307 0.1543 11.295 3.4249 3.4249 0.6606 12.307 0.1555 11.295 3.4249 0.6605 12.253 11.295 0.1956 15.869 6.793 6.793 0.6615 18.563 60 cm 0.1924 15.838 6.4529 6.4529 0.6425 18.427 0.1652 12.524 4.4205 4.4205 0.7093 14.016 0.1652 12.524 4.4205 4.4205 0.7093 14.016 0.1634 12.523 4.3489 0.7086 12.395 0.1492 11.324 3.4125 3.4125 0.7002 12.297 12.395 0.1492 11.324 3.4791 0.7006 12.395 0.1492 11.315 3.3674 3.3674 0.7002 12.297 12.397 0.1634 12.552 4.4672 6.7122 0.6674 18.528 0.1909 15.881 6.6302 6.6302 0.6687 18.459 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1656 11.355 3.534 3.3372 0.7378 12.429 0.1487 11.355 3.534 3.3372 0.7378 11.488 12.597 12.397 0.1487 11.348 3.3372 0.7387 11.488	the state has been party on a right or state of	0.1671	12.505	;	:	_	13.995		0.9375		
0.1533 11.292 3.4249 3.4249 0.6806 12.307 0.1543 11.295 3.47 0.6832 12.335 0.155 0.1956 15.869 6.793 6.793 0.6615 18.563 60 cm 0.1934 15.838 6.4529 6.4529 0.6425 18.223 0.1652 12.524 4.4205 4.4205 0.7093 14.016 0.1652 12.524 4.4205 4.4205 0.7093 14.016 0.1634 12.523 4.3489 0.7086 13.997 0.1492 11.324 3.4125 3.4125 0.7006 12.397 0.1492 11.324 3.4125 0.7006 12.397 0.1492 11.324 3.4125 0.7006 12.397 0.1634 12.552 4.4672 0.731 14.095 0.1634 12.552 4.4672 0.731 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 13.353 3.43372 0.7337 11.48							14.007			1	1
0.1543 11.295 3.47 3.47 0.6832 12.335 725 0.1515 11.286 3.3381 3.3381 0.6756 12.253 725 0.1956 15.869 6.793 6.793 0.6615 18.563 60 cm 0.1934 15.838 6.4529 6.4529 0.6615 18.253 0.1652 12.524 4.4205 4.4205 0.7093 14.016 0.164 12.527 4.3856 0.7107 13.995 0.1635 12.523 4.3489 0.7086 13.995 0.1636 11.324 3.4791 0.7086 13.995 0.1637 11.324 3.4791 0.7086 12.39 255 0.1492 11.323 3.4125 0.707 12.39 60 cm 0.1492 11.315 3.3674 3.5674 0.7002 12.297 255 0.1909 15.881 6.6302 6.6687 18.459 0.1644 12.552 4.5041 4.5041		0.1533	11.292	,	i	0.6806	1			0.005	0.9527
725 0.1956 15.869 6.793 6.793 0.6615 18.263 60 cm 0.1934 15.875 6.6201 6.6201 0.654 18.427 1.2.298 15.838 6.4529 6.4529 0.6425 18.263 18.292 0.1632 12.524 4.4205 4.4205 0.7093 14.016 0.1635 12.523 4.3489 0.7086 13.995 0.1635 12.523 4.3489 0.7086 13.995 0.1635 12.523 4.3499 0.7086 12.395 0.1492 11.324 3.4791 0.7086 12.395 0.1492 11.315 3.3674 3.3674 0.7002 12.297 12.329 0.1492 11.315 3.3674 3.3674 0.7002 12.297 0.1634 12.552 4.4672 6.7122 0.6674 18.513 18.456 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 4.4672 0.7316 14.095 0.1634 12.552 4.4672 4.4672 0.7316 14.095 0.1656 12.559 4.5833 4.5833 0.737 14.153 0.1498 11.355 3.534 3.3372 0.7348 12.429 0.1487 11.343 3.3372 0.7373 11.418		0.1543	11.295		3.47			0.1696	0.9375	0.002	0.9527
725 0.1956 15.869 6.793 6.793 0.6615 18.2298 6.00 cm 0.1934 15.836 6.4529 6.4529 0.6425 18.229 18.427 0.1652 12.524 4.4205 4.4205 0.7073 14.016 0.1635 12.523 4.3489 0.7086 12.397 0.1492 11.324 3.4725 0.1909 15.891 6.6302 6.6302 0.6697 18.456 0.01909 15.891 6.6302 6.6302 0.6697 18.459 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 4.4672 0.7316 14.095 0.1634 12.552 4.4672 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1634 12.552 4.4672 0.7316 14.095 0.1656 11.355 3.534 3.534 0.7348 12.429 0.1498 11.355 3.534 3.3372 0.7373 11.418		0.1515		က	3.3381			0.1696	0.9375		0.9527
725 0.1956 15.869 6.793 6.793 0.6615 18.563 60 cm 0.1934 15.857 6.6201 6.6201 0.664 18.427 0.1652 15.838 6.4529 6.4529 0.6428 18.427 0.1652 12.524 4.4205 4.705 17.17 13.995 0.1635 12.523 4.3489 0.7086 13.997 0.1635 12.523 4.3499 0.7086 13.995 0.1635 11.324 3.4791 3.7086 13.995 0.1636 11.324 3.4791 0.7086 12.399 0.1492 11.315 3.3674 0.7002 12.297 255 0.1909 15.881 6.6302 6.6897 18.459 0.1644 12.552 4.5041 0.707 12.397 0.1656 15.891 6.7122 6.7122 0.6674 18.459 0.1644 12.552 4.5041 0.731 14.095 0.1656 12.559											0.9527
60 cm 0.1934 15.857 6.6201 6.6201 0.654 18.427 - 1.4 0.1921 15.838 6.4529 6.4529 0.6425 18.292 0.1652 12.524 4.4205 4.4205 0.7093 14.016 0.1634 12.527 4.3856 4.3856 0.7117 13.997 0.1492 11.324 3.4791 3.4791 0.7086 12.399 0.1492 11.324 3.4791 3.4791 0.7086 12.399 0.1492 11.315 3.3674 0.7002 12.297 1255 0.1915 15.891 6.7173 6.7173 0.6749 18.528 0.1644 12.552 4.4672 6.7122 0.6674 18.456 0.1656 12.559 4.5833 4.583 0.731 14.095 0.1498 11.355 3.534 3.534 0.7348 12.429 0.1497 11.343 3.3372 0.7318 12.429	•	0.1956			6.793			0.1696		1	0.8916
- 1.4 0.1921 15.838 6.4529 6.4529 0.6428 18.292 18.427 0.1652 12.524 4.4205 4.4205 0.7093 14.016 18.427 0.1635 12.523 4.3489 0.7086 0.7117 13.997 0.1635 12.523 4.3489 0.7086 0.7117 13.995 0.1492 11.324 3.4725 3.4725 0.7006 12.397 0.1492 11.315 3.3674 0.7002 12.297 12.331 0.1909 15.881 6.6302 6.6302 0.6687 18.456 0.1634 12.552 4.4672 0.7316 14.095 0.1655 12.559 4.5833 4.583 0.737 14.153 0.1498 11.355 3.534 3.3372 0.7318 12.429 0.1498 11.355 3.534 3.3372 0.7318 12.429 0.1487 11.343 3.3372 0.7377 12.381 12.097 11.348 12.429 0.1487 11.343 3.3372 0.7377 12.381 12.097		0.1934				0.654			0.8753	0.002	0.8916
0.1652 12.524 44205 44205 0.7093 14.016 0.1635 12.523 4.3489 0.7086 13.995 0.1635 12.523 4.3489 0.7086 13.995 0.1634 11.324 3.4791 3.4791 0.7086 12.368 0.1497 11.323 3.4125 3.4125 0.707 12.329 0.1492 11.315 3.3674 0.7002 12.297 255 0.1909 15.891 6.7173 6.7173 0.6749 18.528 60.0cm 0.1909 15.881 6.6302 0.6687 18.456 0.1634 12.552 4.4672 0.7316 14.073 0.1655 12.559 4.5833 4.5833 0.737 14.153 0.1487 11.355 3.534 3.3372 0.7377 12.381 0.1487 11.355 3.534 3.3372 0.7937 11.488	7	0.1921	15.838			0.6423		0.1695			0.8927
0.1652 12.524 4.4205 0.7093 14.016 0.164 12.527 4.3856 4.3856 0.7117 13.997 0.1635 12.523 4.3489 0.7086 13.997 0.1614 11.324 3.4791 3.4791 0.7086 12.368 0.1492 11.315 3.3674 3.3674 0.7002 12.297 0.1492 11.315 3.3674 3.3674 0.7002 12.297 0.1634 12.552 4.5041 4.5041 0.731 14.095 0.1634 12.552 4.4672 0.7316 14.073 0.1655 12.553 4.673 0.7316 14.073 0.1498 11.355 3.534 3.3372 0.7377 12.429 0.1487 11.347 3.4635 3.4635 0.7277 12.381							18.427				0.892
0.1635 12.523 4.3489 6.7117 13.997 0.1635 12.523 4.3489 0.7086 13.997 0.1514 11.324 3.4791 3.4791 0.7086 12.368 0.1492 11.315 3.3674 3.3674 0.7002 12.2997 1.5 0.1928 15.891 6.7173 6.7173 0.6749 18.528 6.00 cm 0.1909 15.891 6.6302 6.6302 0.6687 18.456 0.1634 12.552 4.4672 0.7122 0.6674 18.513 0.1634 12.552 4.4672 0.7316 14.095 0.1635 12.559 4.4672 0.7316 14.073 0.1498 11.355 3.534 0.737 14.153 0.1487 3.3372 0.7377 12.381 0.1487 11.347 3.4635 3.4635 0.7277 12.381 0.1487 11.347 3.4635 3.4635 0.7277 12.381 0.1487 11.347 3.3372 0.7337 11.48		0.1652	12.524	_	4.4205	0.7093	14.016	0.1694	0.8765		0.8927
0.1635 12.523 4.3489 4.3489 0.7086 13.95 0.1514 11.324 3.4791 3.4791 0.7086 12.368 0.1492 11.315 3.3674 3.3674 0.7002 12.297 255 0.1915 15.891 6.7173 6.7173 0.6749 18.528 6.0 cm 0.1909 15.819 6.6302 0.6687 18.456 0.1634 12.552 4.672 0.7122 0.6674 18.513 0.1634 12.552 4.4672 0.7316 14.095 0.1655 12.559 4.5833 4.5833 0.737 14.153 0.1487 11.355 3.534 0.737 12.429 0.1487 11.355 3.534 0.737 12.429 0.1487 10.434 3.3372 0.737 12.381 12.097		0.164			4.3856	0.7117	13.997	0.1693		0.005	0.8927
0.1514 11.324 3.4791 3.4791 0.7086 12.368 0.1492 11.312 3.3674 0.7002 12.329 0.1492 11.315 3.3674 3.3674 0.7002 12.2297 12.331 0.1909 15.891 6.7173 6.7173 0.6749 18.528 6.0 cm 0.1909 15.819 6.7172 6.7172 0.6677 12.331 0.1634 12.552 4.672 6.7122 0.6674 18.513 0.1634 12.552 4.4672 0.7316 14.095 0.1655 12.559 4.4672 0.7316 14.073 0.1498 11.355 3.534 0.737 14.153 0.1487 11.347 3.4635 3.4635 0.7277 12.381 0.1487 10.434 3.3372 0.737 11.48		0.1635	12.523	!	4.3489	0.7086	13.97	0.1693	0.8765	0.002	0.8927
0.1514 11.324 3.4791 3.4791 0.7086 12.368 0.1497 11.323 3.4125 0.707 12.329 0.1492 11.315 3.3674 3.3674 0.7002 12.297 12.331 0.1909 15.891 6.7173 6.7173 0.6749 18.528 6.0 cm 0.1909 15.891 6.6302 6.6302 0.6687 18.459 0.1634 12.552 4.5041 4.5041 0.731 14.095 0.1655 12.559 4.4672 0.7316 14.073 0.1498 11.355 3.534 0.737 14.153 0.1487 11.347 3.4635 3.4635 0.7277 12.381 0.1487 10.434 3.3372 0.737 11.48				- 1			13.995				0.8927
0.1497 11.323 3.4125 3.4125 0.707 12.329 0.1492 11.315 3.3674 3.3674 0.7002 12.297 725 0.1915 15.891 6.7173 6.7173 0.6749 18.528 60 cm 0.1909 15.891 6.6302 6.6302 0.6687 18.456 0.1634 12.552 4.5041 4.5041 0.731 14.095 0.1655 12.559 4.4672 0.7316 14.073 0.1497 11.355 3.534 3.534 0.737 14.153 0.1487 11.347 3.4635 3.4635 0.7277 12.381 0.1487 10.434 3.3372 0.7937 11.48		0.1514	11.324		3.4791	9	12.368	0.1692	0.8765	0.002	0.8927
0.1492 11.315 3.3674 3.3674 0.7002 12.297 725 0.1915 15.891 6.7173 6.7173 0.6749 18.528 6.00 cm 0.1909 15.891 6.6302 6.6302 0.6687 18.456 0.1634 12.552 4.672 6.7122 0.6674 18.513 0.1634 12.552 4.4672 4.4672 0.7316 14.095 0.1655 12.559 4.5833 4.5833 0.737 14.153 0.1497 11.347 3.4635 3.4635 0.7277 12.381 0.1487 10.434 3.3372 0.7937 11.48		0.1497	11.323		3.4125		12.329	0.1692	0.8765	0.005	0.8927
725 0.1915 15.891 6.7173 6.7173 0.6749 18.528 6.60 cm 0.1909 15.881 6.6302 6.6302 0.6687 18.456 6.7122 0.1928 15.879 6.7122 6.7122 0.6674 18.513 0.1634 12.552 4.5041 4.5041 0.731 14.095 0.1655 12.559 4.672 4.4672 0.7316 14.073 0.1655 12.559 4.5833 4.5833 0.737 14.153 0.1497 11.347 3.4635 3.4635 0.7277 12.381 0.1487 10.434 3.3372 0.7937 11.48	and desirable of the property of the same	0.1492	11.315	က	3.3674	0.7002	12.297	0.1693	0.8765	0.002	0.8927
725 0.1915 15.891 6.7173 6.7173 0.6749 18.528 60 cm 0.1909 15.881 6.6302 6.6302 0.6687 18.456 6.7122 0.1928 15.879 6.7122 0.6674 18.513 0.1634 12.552 4.5041 4.5041 0.731 14.095 0.1655 12.559 4.672 4.4672 0.7316 14.073 0.1655 12.559 4.5833 4.5833 0.737 14.153 0.1497 11.347 3.4635 3.4635 0.7277 12.381 0.1487 10.434 3.3372 0.7937 11.48							12.331				0.8927
60 cm 0.1909 15.881 6.6302 6.6302 0.6687 18.456 -1.5 0.1928 15.879 6.7122 6.7122 0.6674 18.513 0.1634 12.552 4.5041 4.5041 0.731 14.095 0.1655 12.559 4.5833 4.5833 0.737 14.153 0.1498 11.355 3.534 3.534 0.7378 12.429 0.1487 11.347 3.4635 3.4635 0.7277 12.381 0.1487 10.434 3.3372 3.3372 0.7937 11.48	• ~	0.1915	15.891	6.7173	6.7173	0.6749	18.528	0.1693	0.8209	0.002	0.8382
- 1.5 0.1926 15.879 6.7122 6.7122 0.6674 18.513 (18.49) 0.1644 12.552 4.5041 4.5041 0.731 14.095 0.1655 12.559 4.4672 4.4672 0.7316 14.073 (14.163) 0.1655 12.559 4.5833 4.5833 0.737 14.153 0.1498 11.355 3.534 0.7378 12.429 0.1487 11.347 3.4635 3.4635 0.7277 12.381 0.1487 10.434 3.3372 0.7937 11.48		0.1909	15.881		6.6302	0.6687	18.456	0.1693	0.8209	0.002	0.8382
12.552 4.5041 4.5041 0.731 18.499 12.552 4.4672 4.4672 0.731 14.095 12.559 4.5833 4.5833 0.737 14.153 11.355 3.534 3.534 0.7348 12.429 11.347 3.4635 3.3372 0.7277 12.381 10.434 3.3372 0.7937 11.48	Ţ	0.1928	15.879		6.7122	0.6674	18.513	0.1693	0.8219	0.005	0.8392
12.552 4.5041 4.5041 0.731 14.095 12.552 4.4672 4.4672 0.7316 14.073 12.559 4.5833 4.5833 0.737 14.153 11.355 3.534 3.534 0.7348 12.429 11.347 3.4635 3.4635 0.7277 12.381 10.434 3.3372 0.7937 11.48							18.499				0.8385
12.552 4.4672 4.4672 0.7316 14.073 12.559 4.5833 0.737 14.153 11.355 3.534 3.534 0.7348 12.429 11.347 3.4635 3.3372 0.7277 12.381 10.434 3.3372 3.3372 0.7937 11.48		0.1644	12.552		4.5041	0.731	14.095	0.1692	0.8219	0.005	0.8392
12.559 4.5833 0.737 14.153 11.355 3.534 3.534 0.7348 12.429 11.347 3.4635 3.3372 0.7937 12.381 10.434 3.3372 3.3372 0.7937 11.48		0.1634	12.552		4.4672	0.7316	14.073	0.1692	0.8219	0.005	0.8392
11.355 3.534 3.534 0.7348 12.429 11.347 3.4635 3.4635 0.7277 12.381 10.434 3.3372 3.3372 0.7937 11.48		0.1655	12.559		4.5833	0.737	14.153	0.1692	0.8219	0.005	0.8392
11.355 3.534 3.534 0.7348 12.429 11.347 3.4635 3.4635 0.7277 12.381 10.434 3.3372 3.3372 0.7937 11.48							14.107				0.8392
11.347 3.4635 3.4635 0.7277 12.381 10.434 3.3372 3.3372 0.7937 11.48 12.097	alan be cannon a sample of hypothesis of an area and to	0.1498	11.355	3.534	3.534	0.7348	12.429	0.1691	0.8219	0.005	0.8391
10.434 3.3372 3.3372 0.7937 11.48		0.1487	11.347	3.4635	3.4635	0.7277	12.381	0.1691	0.8219	0.002	0.8391
12.097		0.147	10.434	3.3372	3.3372	0.7937	11.48	0.1691	0.8219	0.005	0.8391
							12.097				0.8391

20	22 5 30 4) (0) ((0)	€ 5	3	<u> </u>) (A				n N	×	9	KC (fte)	٧.	A CAA	B Rs*Rs n)		A PR	JAS %
1 0	22.5 30.5 30.5	20.0	3	3 6	- 60 1.87	/R/	0.85		62.07	213	12.1	0.034	0.34	1.069	1853	3 118	30 0.0	4.9	37 1.6036	
2	30.5	20.7	-		1.30	17)		215	61.82	213		0.034	0.34	-					-	
1		3	1.67)))	÷0,7	74/	1	523	33	213		0.034	0.34	_	_	3 1180	30 0.005	05 4.937	37 1.6036	6 155.2
S	5 34 7	33.8	=	000	00.01	703	-	i.s.: 1	80 h		:::I+	0.034	0,34	1.069	1853	3 1180	, en en e			
Ç	22.5 34.6	333.7			20.01	757	0.00	212	67.16	213	13.1	0.034	0.34	1.069	1853			4	上	1
Ņ	5 34.9	34	=	800	200	161		OLL	11./0	213	<u> </u>	0.034	0.34	1.069		3 1180	0.007	07 4.937	1.6036	6 155.
		1) }	2) (E)		0 0	00.04		i	0.034		1.069	_	_		4	!-	: 1
S	7 38.6	37.3	14	L	1 86		- 1	7	50.94	3 L	623 L	0.034	0.34	1.069	1853	Ξ	80 0.00	❖	1	1
8	7 38.9	37.6	7		000			ລ		,		0.034	0.34	1.069		_	_	4	3 1 6036	
100	38.8	27.0		- ;	C	121	C8.0	215 7	72.74	213	14.2 0	.034	0.34	1.069	-	!		. 4	_	-
į	2	3	+		20.		1	20	- 6	- 2		0.034	0.34	1.07	1853	3 1180	0.009		6 1 6036	5 2 2
18	306	000	880 BB	100	20.				73,32	213	14.3 0	0.034	0.34	1,069	1853			4		
0000		20.0	2.03	70.0	98.	728 0	0.901			539	12.7 0	0.034	0.36	1.132		-	-		+	
v (7000	- 11		0.08	8,23 8,23		2019-35			239	-	.034	0.36	1 132	-	1	<u> </u>		1 -	
ZK.		29.9	200	0,08	8.17		· Mercens	241 7	76.41	239	14.9	0.034	0.36	1 135		1102	0.003		7 1.6998	155.
18			2000	- I				* 500.00	72.4	, jij.	-0.00	0.034	0.36	190				75.037	_:	
3	3.4.6	33.7	~ [0,09	10.23	728 0	0.901	241 7			-	0.034	0.36	1 132	Œ.	۱ <u>-</u>	100			[
Ý,	. 45.	33.8	ωi			333.47	500 200 500 200	-		239		034	0.36	1 133		- : -		0.00	-:-	
ξ.	2.45. 20.45.	34	1.00		100	- 1444	300.00g	241 7		239	13.8	0.034	0.361	133	1856	1183	0.003	L	4 1.6998	155.7
18	000	1				200		7	100	239 1	4.00	-033	0.361	133		·			- I	-:
S	38.9	37.5	week!	_	7	9.40	200000	⊢	76.05	239 1		4	1361	1 100	Ľ	Π'	* 1			_1
23.2	39.5	37.8		0.11	12.26	728 0	0,901	241 78	1				36.1	200	_; •	- : •				
83	39,4	38	14.8	_	34	90.35	do. avon	<u> </u>	!		4.7	0.034	0.361	133	1856	1182	0.007	5.547	7 1.6998	3 155.7
18			20020					Z.	75,74	- 77			0.361	1.33	ું.				-:	
, C. C.	2	9	200000			200		<u> </u>	_	268 1	5.4 0		382	10	1866	Ľ		்ட		- 1
23.2		30.3	00000	0.08	8,44	728 0,	0,954 2	271 78	78.07		100	0.034	0.382	10	1000		0.003	olo	1.7998	156.2
3			7.21		80	2000		-			5.2		0.382	1 100	2000	1 102	-			٠,
19								7	78.18	268 1	· @		0.382	1 100	1000		0	0 0		,
23.3	35.2	34.1	10.8	0.1	0.64 7	728 0.	1	270 B		L	8	034	381	1 100	1000	Ľ	233	۲		
23.3		34.3	Ξ			848	1	-	1		7.0		000	0 0	000	_;		4 6.209	-	156.2
23.4		34.6	11.2			728 0	0 950 0	070			5	200	185	1.197	1856	-	0.004	9	3 1.796	156.
						38A			3		0	8	0.381	1.197	1856	1182		4 6.2		156.2
3.4	39.2	37 B		ľ	Ş	8 IS	. 1				ျ		.381	1.198	1856	1182	9.004	6.20	1	
3	39.6	38.5	147 6		1 4	38933		- 11		267	0	.034 0.	.381	1.197	185c	182	_		L	
23.6	39.5	38.4			7 0 0 0	, K	0.952 2	270 77	- 1		5.2 0.		0.381	1.198	1856	_		6.20	1 796	156.0
,) } }	3			8	92.3			78.4 2	_		0.034	381	1 100	1956	: *		. (٠,
					The second of th		ACCES OF THE PARTY	Ĵ	į			_	5	000	000	201		_		۳

overall Rs				<u>:</u>	_				-			0.7946	0.7517	0.7517	0.7517	0.7517	0.7517	0.7517	0.7517	0.7517	0.7517	0.7517	0.7517	0.7517	0.7121	0.7121	0.7121	0.7121	0.7128	0.7135	0.7135	0.7133	0.7135	0.7135	0.7135
funcer	0000	i	1		0.002	:	0.00	i	1		0.002				0.002		0.002	0.005	0.002		0.005	0.002	0.005		0.005	0.005	0.002		0.002				0.002		
Mic V f uncert uncert	0 7765	0.7765	0.7765	-	-	_	0.7765		0.7765	0.7765	:		0.7325	0.7325	0.7325		0.7325	0.7325	0.7325		0.7325	0.7325	0.7325		0.6918	0.6918	0.6918		0.6925	0.6933	0.6933		0.6933	0.6933	0.6933
Ta	0.1691	-	0.1691		0.1691	_	0.1691		0.169		0.1689	- 1		0.1689	0.1689		0.1688	0.1688	0.1688		0.1688		0.1687		0.1687	0.1687	0.1688		0.1687		0.1686		0.1686	0.1685	
overall Nusselt uncert	_	18.56	18.698	18.628	14.079	14.101		14.068				11.546	18.896	17.357	17.446	17.899	14.291	14.264	14.233	14.263	11.614	11.585	11.553	11.584	17.573	17.406	17.324	17.434		13.242	13.185	13.244			11.633
Req uncert	0.6944	0.6916	0.6973		0.7514	0.7509				0.8138	0.8234		0.729	0.8463	0.8549		0.7945	0.7938	0.7898		0.8509	0.8483	0.8429		0.8824	0.8735	0.8681		0.9051	0.8969	0.8917	_		0.8689	
Ta	6.8161		6.9055		4.4396		4.3649		3.4317	3.3647	3.4101		7.1103	7.0113	7.1086		4.6898	4.6494	4.6081		3.4629	3.4187	3.3747			0306	6.9379				4.4667	-		3.4046	
Ta	6.8161	6.7289			`		7				3.4101		7.1103	7.0113	7.1086		4.6898	4.6494	4.6081		3.4629		3.3747	0000		0306	6.9379			4.5429		!			3.4503
	15.923	5	15.928		- 1	12.577	_		10.466	10.456	10.466	00	15.98	14.219	14.231	0000,	12.633	12.632	12.02/		10.495	10.492	10.487	7.00	14.2/1	4.238	14.25	1		11.543		01	1	10.514	
V (uncert uncert	0.19	0.1887	0.1913		0.1602	0.1611	0.1592		0.1466	0.1457	0.146	4000	0,1898	0.1881	0.1887	0000	0.100	0.1088	0.1584	9,7,7	0.1449	0.1439	0.1433	0 4074	-		0.1838	-			0.1562			0.1418	
Experiment Information V	f ~ 725	9	PH ~ 1.6									705	~ / £3	10000	1.1 ~ H									705			2								

SPL (dB)	156.6	156.6	156.6	-	156.6	156.6	156.6		156.6	156.6	156.6	
PB %	.8922	.8922	.8922		.8922	.8922	1.8941		1.8941	1.8941	.8941	
8 <	.853 1	6.857	.853	.854			1	i	879 1	.883	.892	.885
B Hs	002 6	0.002 6	302 6	302 6	303 6	303 6	003	903 6	004 6	004 6	004 6	304 6
_β Rs*Rs /π)	3 0.0	3.0	0.0	3.0.	0.0	3.0	0.0		3.0	3.0	0.0	0.0
1 2.A.A./π		1183	1	38000	1			7550			!	
VAV		1859		1990	ŧ .	1		1000		:	1	-35533
SX.	1.26	1.26	1.26	1.26	1.26	1.26	1.262	1.261	1.262	1.262	1.262	1.262
ы	3.401	0.401	0.401	0.401	0.401	0.401	3.402	0.401	0.405	3.402	204.0	3.402
×	0.034	0.034	0.034	.034	0.034	0.034	0.034	0.034	0.034	.034	0.034	034 (
Nu	16 C	15.5	15.3	15.6 C	16 0	16.4	16.2	16.2 (15.8	15.6	17.4	16,3 (
Corr Fis	295	296	295	295	296	296	297	298	297	297	297	297
h Wm/2	81.66	79.49	78.43	79,86	81.89	83.98	82.78	82,89	80.83	80.04	89.02	83.3
Яŝ		299			299	299	300		300	300	300	
mic V (mV)	1.003	1,003	1.003		1.003	1,003	1.004		1.004	1,004	1.004	
Freq. (Hz)	729	729	729		729	729	729				729	
To X	8,68	8.8	8.91		10,94		11,23		12,67	12.8	13	3
Og.	80'0 6	0.08	0.08		 	-				5 0.11		
SvT a	6.99	7.28	7.47		F	10.9	11.2		4.	4.	14.4	Sala
20	30.4 6.9	30.8	30.9		6 34.5 11	34.6	36 34.9 11.		37.9	38.3	38.4	
<u> </u>		(C)	9.		35.6	36.7	36		39.3	39.7	40	
20	23,4 31,1	23.5	23,4		23.5 35.6 34.5	23.7	23.7		23,7 39,3 37.9	23.8	54	
tent tion							bec					an.
morna	~ 725	. = 60 cm	PR ~ 1.9				-					
	-		0			-	-	_	_ !			

	m	m	· m	im	m	IOI	ΙΩ	10	10	10	: (0	100
overal Rs uncert	0.002 0.6793	0.002 0.6793	0.002 0.6793	0.6793	0.002 0.6793	0.002 0.6792	0.6786	0.679	0.6786	0.6786	0.6786	0.6786
f unoert	0.005	0.005	0.002		0.002	0.005	0.005		0.002	0.002	0.005	
TaMic.Vf uncert ∎uncert_unbert	0.658	0.658	0.658		0.658	0.658	0.6574		0.6574	0.6574	0.6574	
Ta undert	0.1686	17.293 0.1685	0.1686		0.1685	13.328 0.1684	0.1684		0.1684	0.1684	11.013 0.1683	
overall Nusselt Ta uncert und	17.552 0.1686	17.293	17.14	17.329	13.269	13.328	13.233 0.1684 0.6574	13.277	11.707	11.656	11.013	11.459
Reg uncert	0.9137	0.8894	0.8775		0.9162	0.9396	0.9262		0.9043	3.4567 0.8955 11.656 0.1684 0.6574	0.996	
Tc uncert	7.1509	6.8659	6.6904		4.5517	4.6005	4.4822		3.5263	3.4567	3.4699	
Ta uncert	0.1819 14.316 7.1509 7.1509 0.9137	6.8659 6.8659 0.8894	6.6904 6.6904 0.8775		0.1558 11.566 4.5517 4.5517 0.9162 13.269 0.1685	11.593 4.6005 4.6005 0.9396	11.578 4.4822 4.4822 0.9262		0.1418 10.552 3.5263 3.5263 0.9043 11.707 0.1684 0.6574	10.543 3.4567	3.4699	
l unoeit	14.316	14.281	14.264		11.566	11.593			10.552	10.543	9.8081	
V (ihoart	0.1819	0.1798	0.178		0.1558	0.1547	0.1533		0.1418	0.1408	0.1409	
l≅petiment √intopraeuton	f ~ 725	L = 60 cm	PR ~ 1.9									

LIST OF REFERENCES

Carter, R. L., from Swift, G. W., "Thermoacoustic Engines", J. Acoust. Soc. Am., vol. 84, no. 4, pp. 1145-1180, 1988.

Davidson, B. J., "Heat Transfer From a Vibrating Circular Cylinder", *Int. J. Heat Mass Transfer*, vol. 16, pp. 1703-1727, Great Britain, 1973.

Feldman, K. T., "A study of heat generated pressure oscillations in a closed pipe", Ph.D. dissertation, Mechanical Engineering, University of Missouri, 1966.

Gifford, W. E., and Longsworth, R. C., "Surface heat pumping", Adv. Cryog. Eng., vol. 11, no. 171, 1966.

Gopinath, A., and Mills, A. F., "Convective Heat Transfer from a Sphere Due to Acoustic Streaming", *Journal of Heat Transfer*, vol. 115, pp. 332-341, 1993.

Hall, P., "On the stability of the unsteady boundary layer on a cylinder oscillating transversely in a viscous fluid", *J. Fluid Mech.*, vol. 146, pp. 347-367, 1984.

Honji, H., "Streaked flow around an oscillatory cylinder", J. Fluid Mech., vol. 107, pp. 509-520, 1981.

Lighthill, M. J., "Introduction. Real and Ideal Fluids", *Laminar Boundary Layers*, pp. 1-45, Clarendon Press, Oxford, 1963.

Lord Rayleigh, The Theory of Sound, 2nd ed., vol. 2, Dover, New York, 1945.

Mozurkewich, G., "Heat Transfer form a Cylinder in an Acoustic Standing Wave", *Journal of Accoust. Soc. Am.*, vol. 98, pp. 2209-2216, 1995.

Richardson, P. D., "Effects of sound and vibration on heat transfer", *Applied Mechanical Review*, vol. 20, no. 3, pp. 201-217, 1967.

Sarpkaya, T., "Force on a Circular Cylinder in Viscous Oscillatory Flow at Low Keulegan-Carpenter Numbers", *Journal Fluid Mechanics*, vol. 168, pp. 61-71, 1986.

Sondhauss, C., "Ueber die Schallschwingungen der Luft in erhitzten Glasröhren und in gedeckton Pfeifen von ungleichen Weite", *Annual Physics*, vol. 79, no. 1, 1850.

Stuart, J. T., "Double boundary layers in oscillatory viscous flows", *J. Fluid Mech.*, vol. 24, pp. 673-687, 1966.

Swift, G. W., "Thermoacoustic engines and refrigerators", Physics Today, pp. 22-28, 1995.

INITIAL DISTRIBUTION LIST

1.	Defense Technical Information Center 8725 John J. Kingman Rd., STE 0944 Ft. Belvoir, VA 22060-6218	No. Copies 2
2.	Library, Code 13 Naval Postgraduate School Monterey, CA 93943-5002	. 2
3.	Prof. Ashok Gopinath Department of Mechanical Engineering Naval Postgraduate School Monterey, CA 93943-5002	1
4.	Prof. Matthew Kelleher Department of Mechanical Engineering Naval Postgraduate School Monterey, CA 93943-5002	1
5.	Prof. Oscar Biblarz Department of Aeronautics and Astronautics Naval Postgraduate School Monterey, CA 93943-5002	1
6.	Prof. Daniel Collins Department of Aeronautics and Astronautics Naval Postgraduate School Monterey, CA 93943-5002	1
7.	LT. Don Harder N. 5404 Walnut Spokane, WA 99205	1